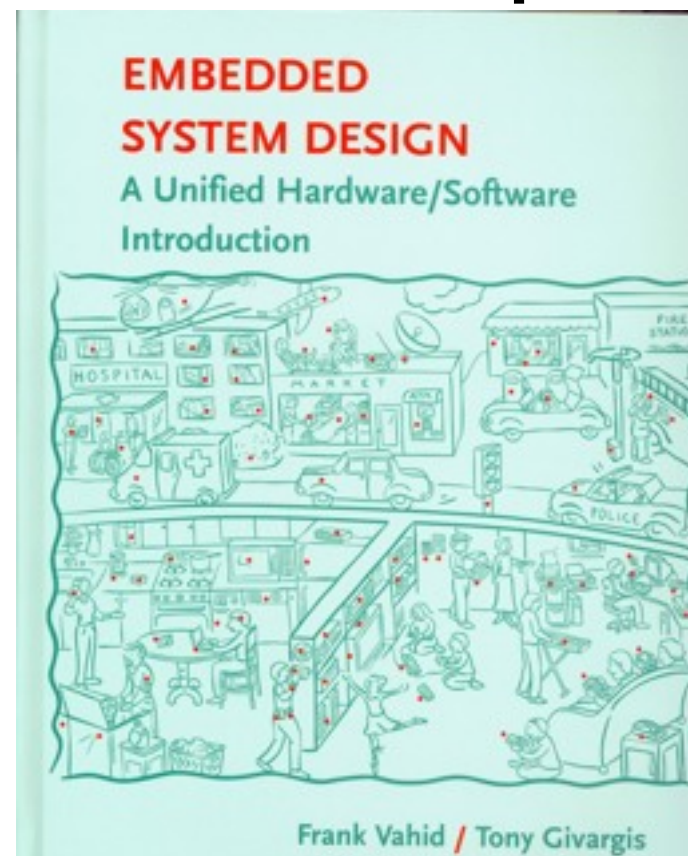


The background of the slide features a large, light blue circular seal of the University of Brussels. The seal contains a central emblem with a sunburst at the top, two lions on the sides, and two crossed keys at the bottom. The Latin text "SCIENTIA VINCERE TENEBRAS" is inscribed along the top arc, and "UNIVERSITAS BRUXELLENSIS" along the bottom arc.

Introduction to control theory

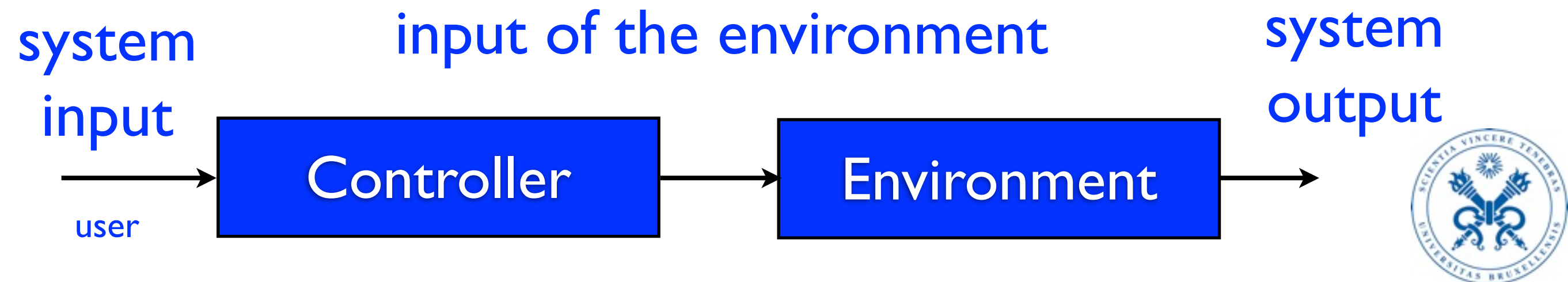
Reference

- These slides are freely adapted from the book *Embedded Systems design - A unified Hardware/Software introduction*, Frank Vahid, Tony Givargis
- Some figures are excerpt of the book



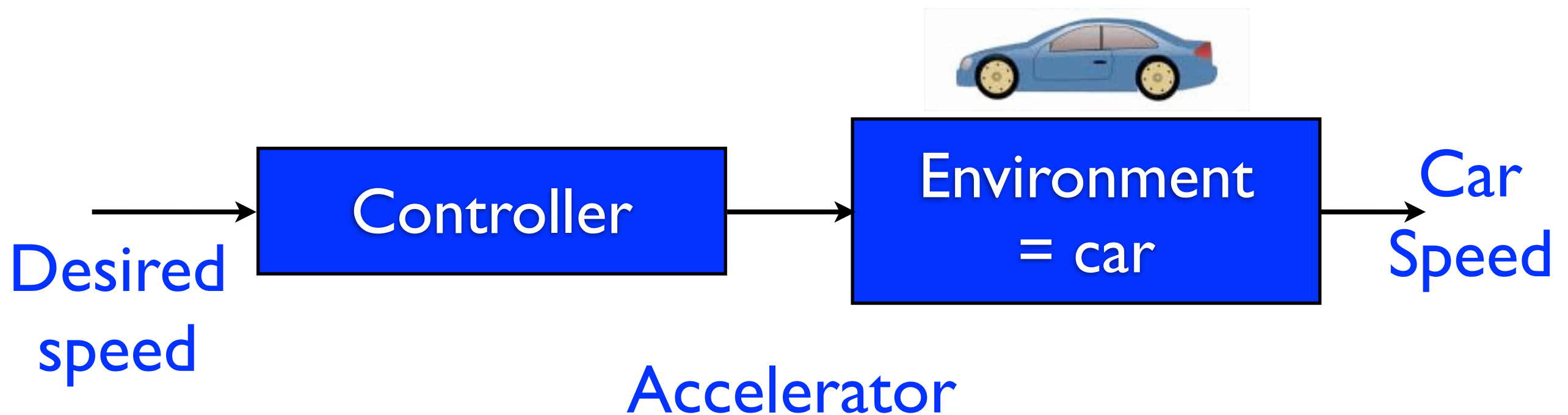
Controller

- A **controller** is a system acting on a **physical device** (called environment)
- The controller has to ensure that the **system output** (system = environment + controller) respects a **given property**
- The controller controls the **input** of the **environment**



Controller

- **Example:** a car “*cruise control*” system



The controller has to ensure that the car speed is as close as possible to the desired speed



Controller quality

- What is a **good controller** ?
- The **definition** we have given is **vague**.
 - A device that keeps the speed equal to 50 km per hour all the time is a controller...
- We will allow the actual speed to be “**more or less different**” from the requested speed

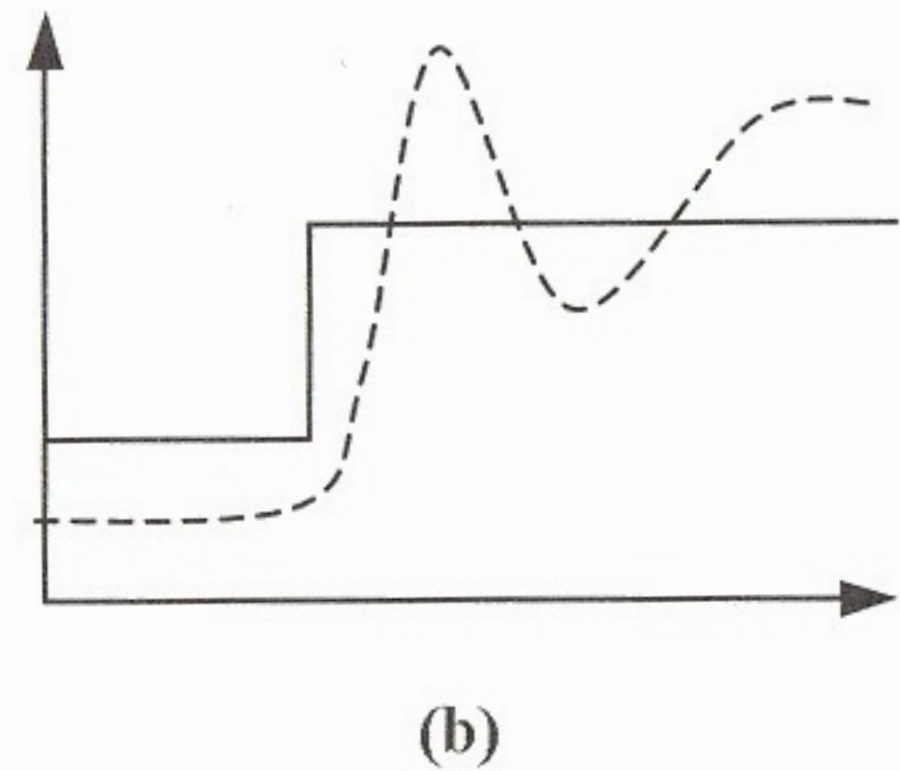
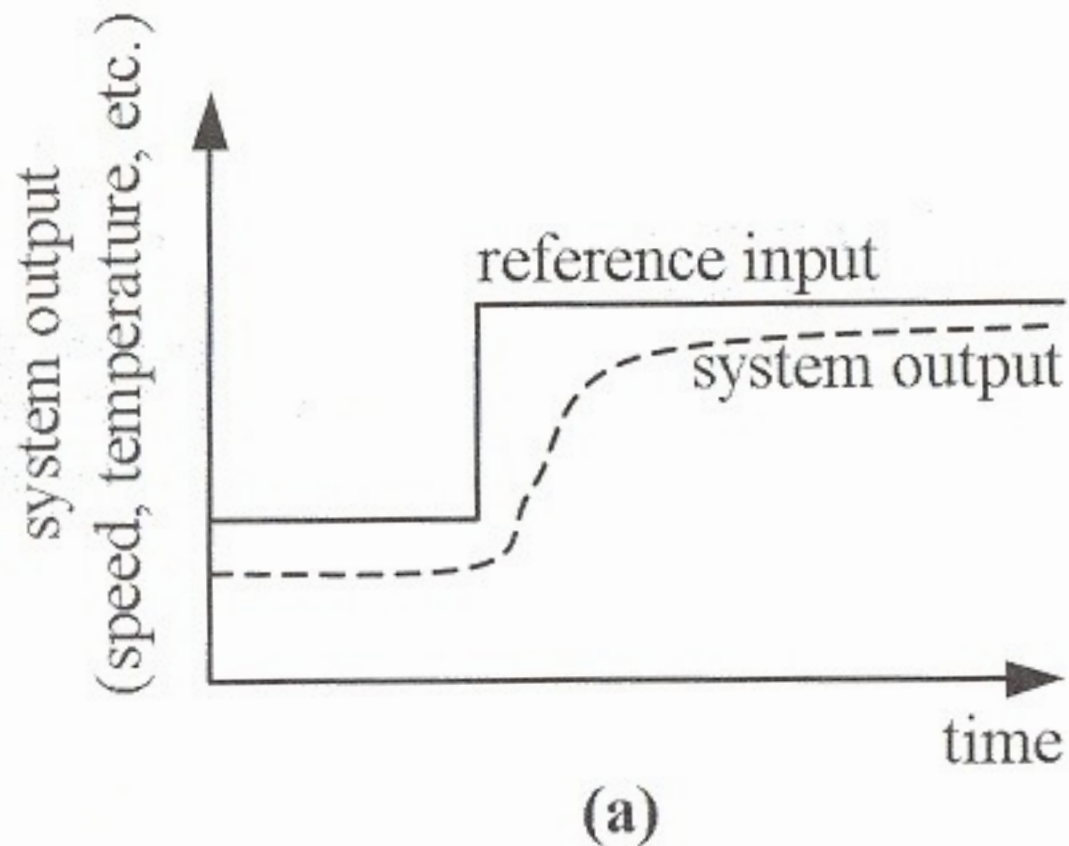


Controller quality

- What is a **good controller** ?
 - In this chapter, we will not provide a **formal definition** of a “**good controller**” (in contrast with next chapter)
 - The **quality** will be assessed by criteria **that are not strict**, mainly by testing.
 - We will often need **compromises**...



Controller quality



“good” controller

“acceptable” controller



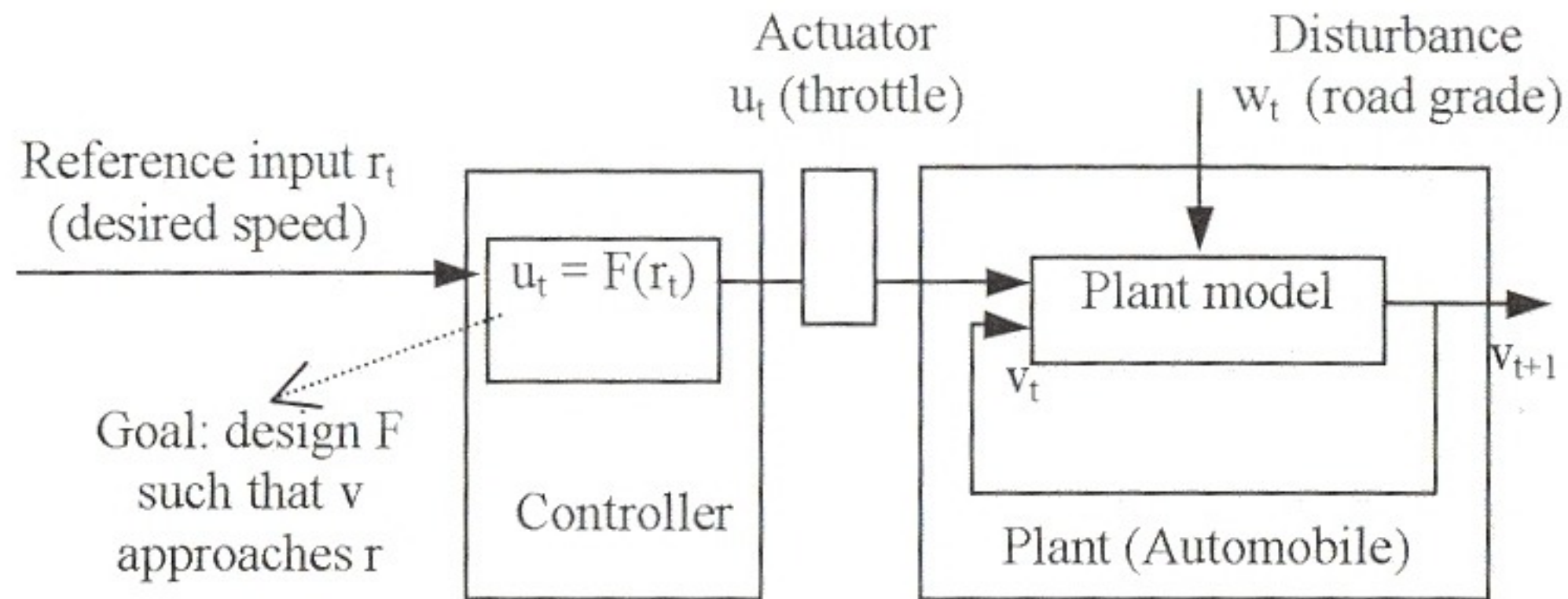
Vocabulary

- The **reference** is the **desired value** for the output
- An **actuator** is a **device** that allows to **control the input** of the environment (e.g.: accelerator pedal)
- A **disturbance** is an uncontrolled **parameter** of the **environment** (e.g.: road grade)
- A **sensor** is a **device** to measure the **output** of the environment (e.g.: speedometer)



Open loop systems

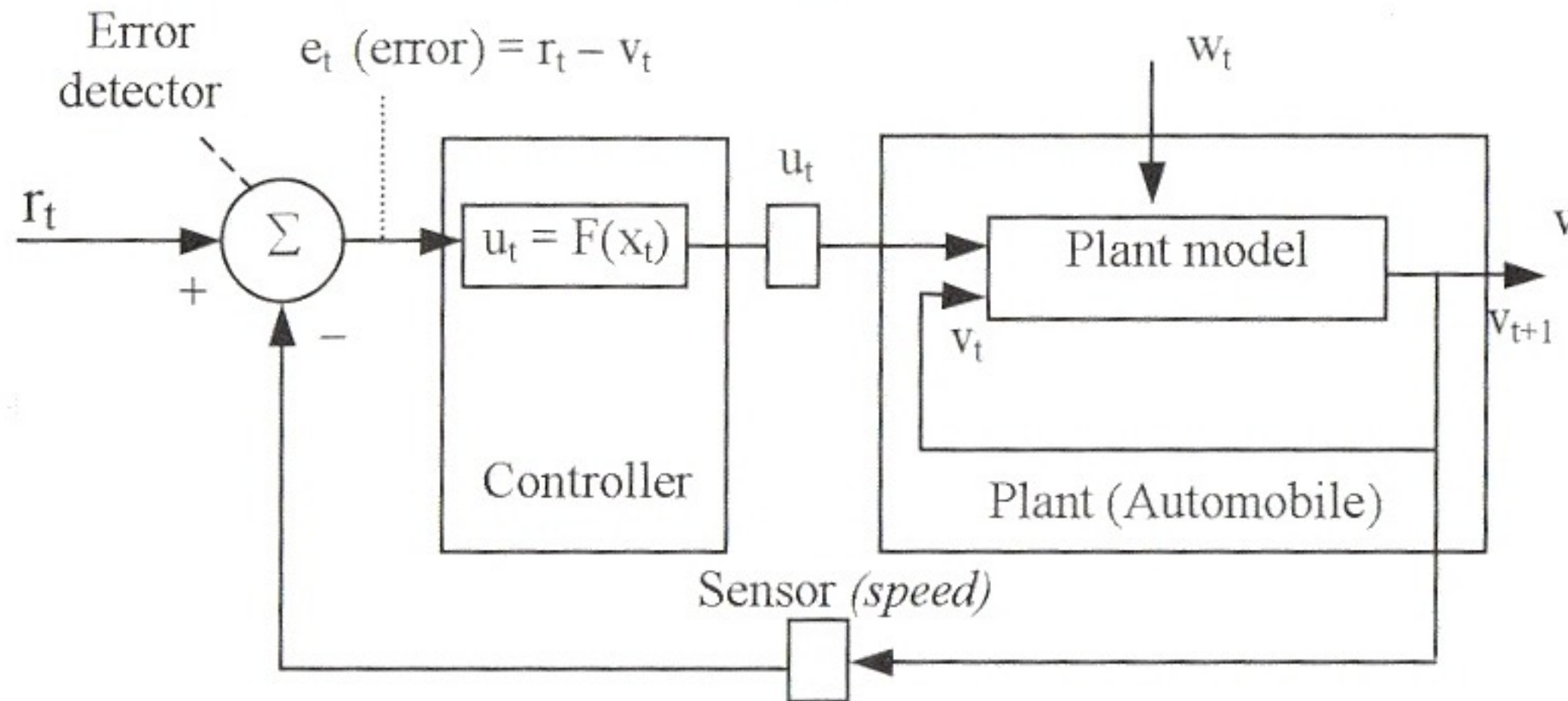
Speed at time $t+1$ depends on the speed at time t , the input and the disturbance



The controller has **no information** about the car's actual speed



Closed loop systems



The controller **knows** the car's speed.
It can use this information to **adapt** its output.



Controller - flows

- In this chapter, we'll see the **data** as “à la Lustre” **flows**
- The controller and the environment are thus **flow modifiers**
- **Example:**
 - r = flow that defines the ref. speed
 - r_t = reference at time t



Controller - flows

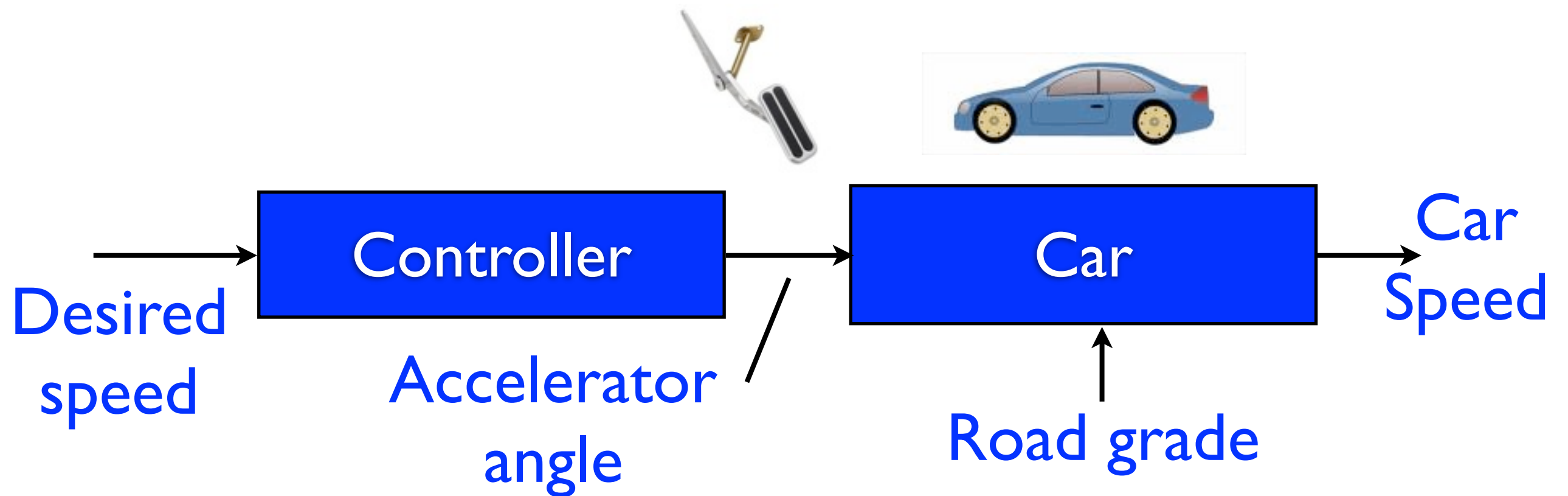
- Some flows will be regarded as **constant**
- We'll abuse notation and use the **flow name** instead of its (constant) **value**
- **Example:**
 - **the road grade** will be regarded as a constant
 - We have $w_t = w$ for all t



The background of the slide features a large, light blue watermark of the seal of the University of Brussels. The seal is circular and contains the Latin motto "SCIENTIA VINCERE TENEBRAS" at the top and "UNIVERSITAS BRUXELLENSIS" at the bottom. The central emblem depicts a sunburst at the top, two crossed torches in the middle, and two crossed keys at the bottom.

Open loop controller

Running example

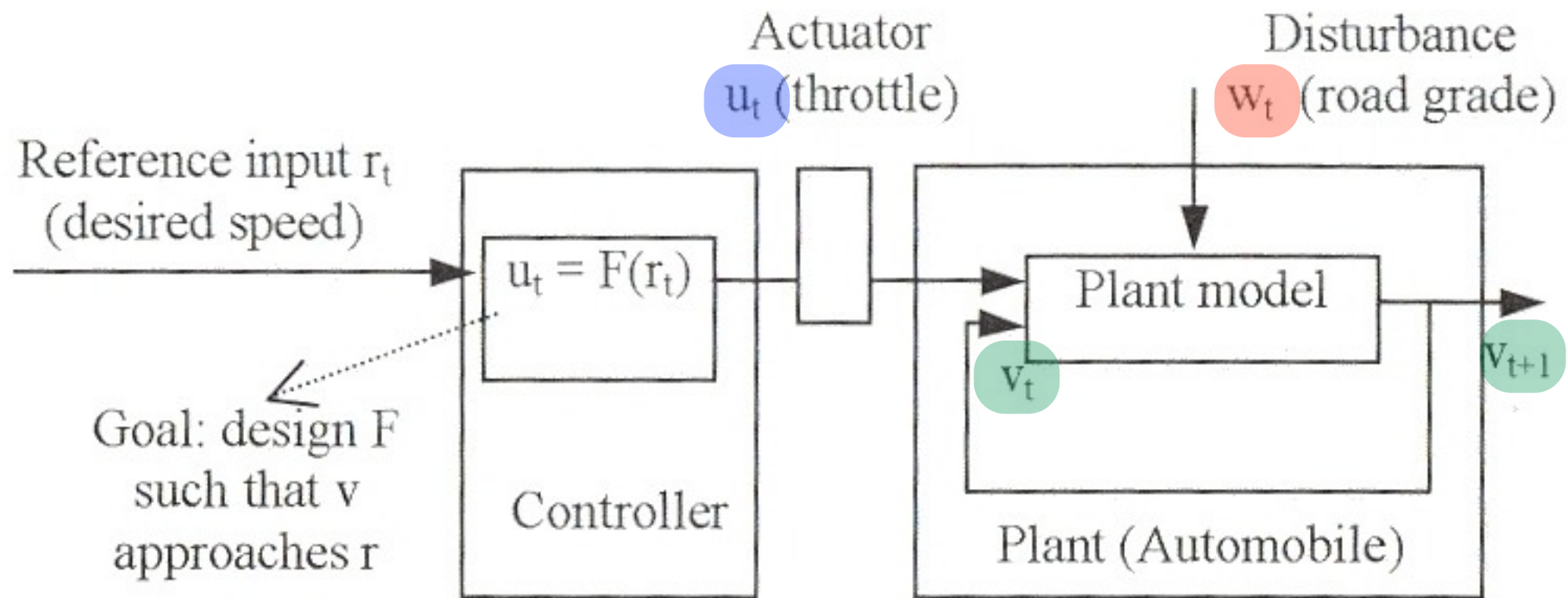


Flows at work in the example

- r_t = reference speed (constant)
- u_t = controller output = input to the car = accelerator position
- w_t = road grade (constant)
- v_t = actual speed car
- e_t = error = $r_t - v_t$



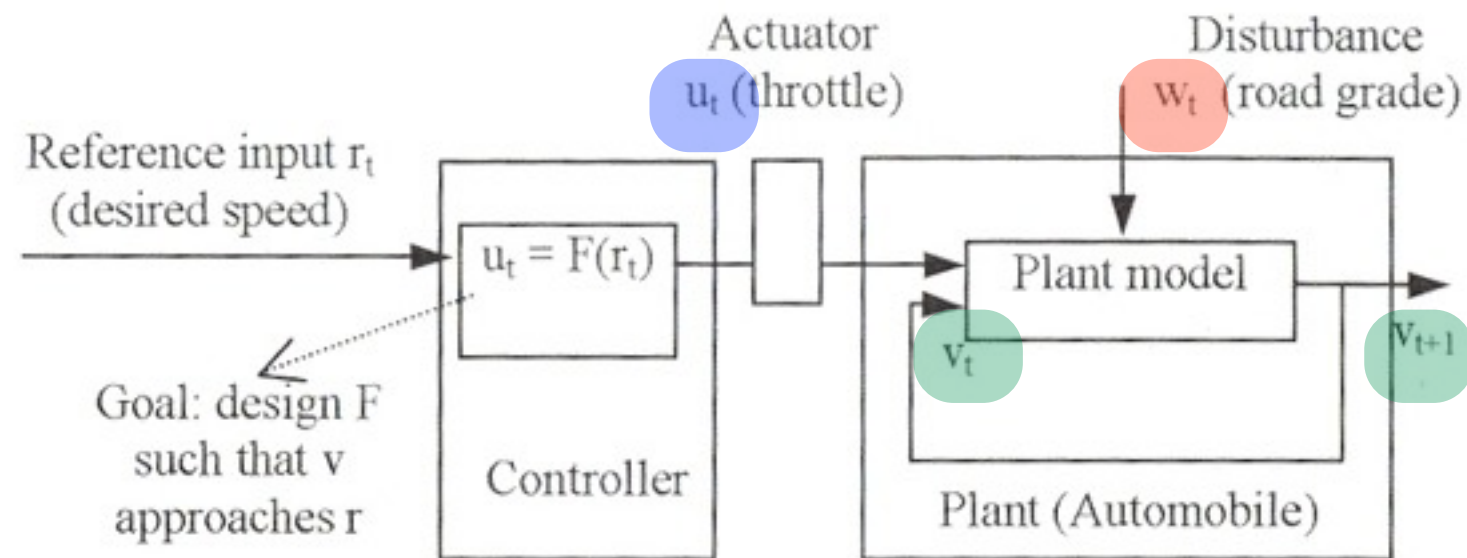
Open loop



- We first need a model for the car
- How is v_{t+1} computed from v_t , u_t and w_t ?



Open loop



- Let us fix $v_{t+1} = 0,7v_t + 0,5u_t$
- We ignore disturbances for the moment



Car model

- $v_{t+1} = 0,7v_t + 0,5u_t$
- At each time the **speed** depends on **the speed at previous step** and the **position** of the accelerator pedal
- For each speed v , there exists a pedal position that allow to **maintain v**
- e.g. $v_t = v_{t+1} = 50 \rightarrow u_t = 30$

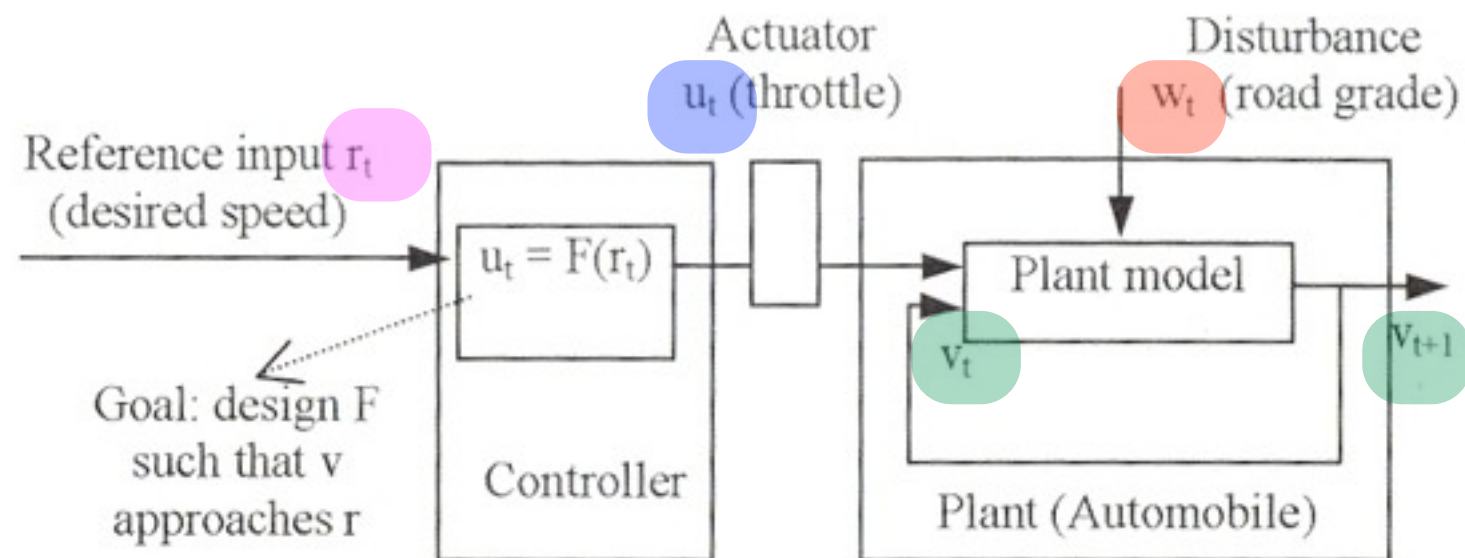


Car model

- $v_{t+1} = 0,7v_t + 0,5u_t$
- This can be simulated in SCADE... (file modele-voiture.avi)



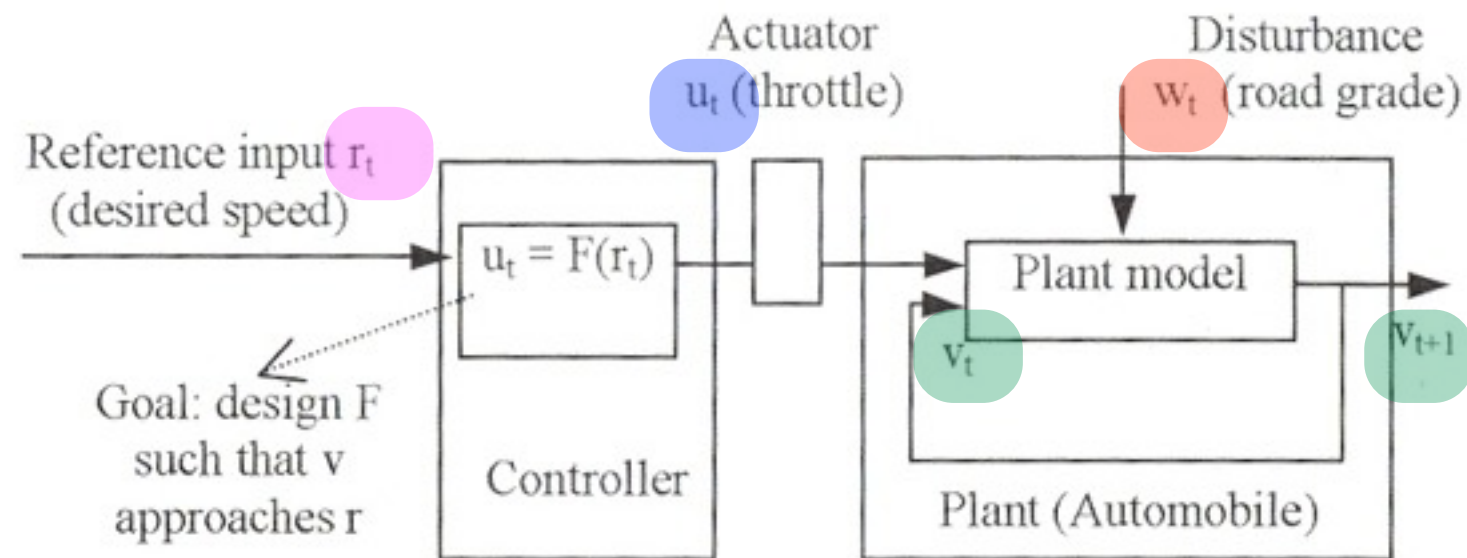
Open loop



- Let us now **design** a **controller** $F(r_t)$, that, depending on r_t , returns u_t to be fed to **the environment**



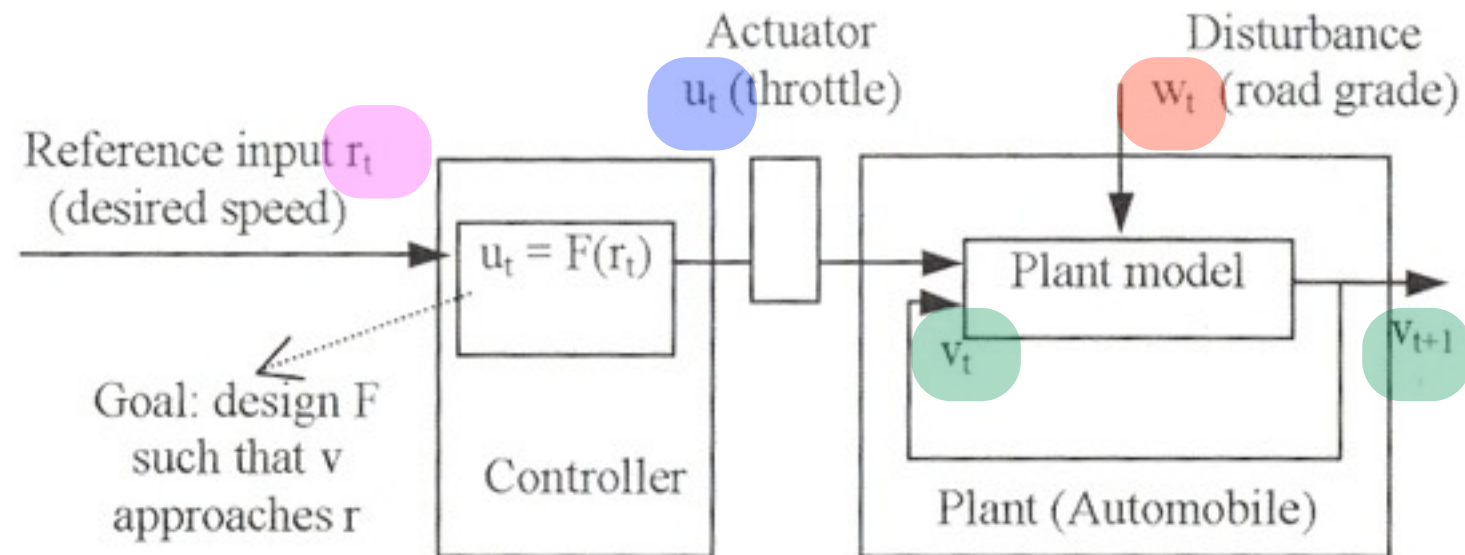
Open loop



- We'll start with a simple linear function $F(r_t) = P \times r_t$



Open loop



$$\left. \begin{aligned} F(r_t) &= P \times r_t \\ u_t &= F(r_t) \end{aligned} \right\} \text{Controller}$$

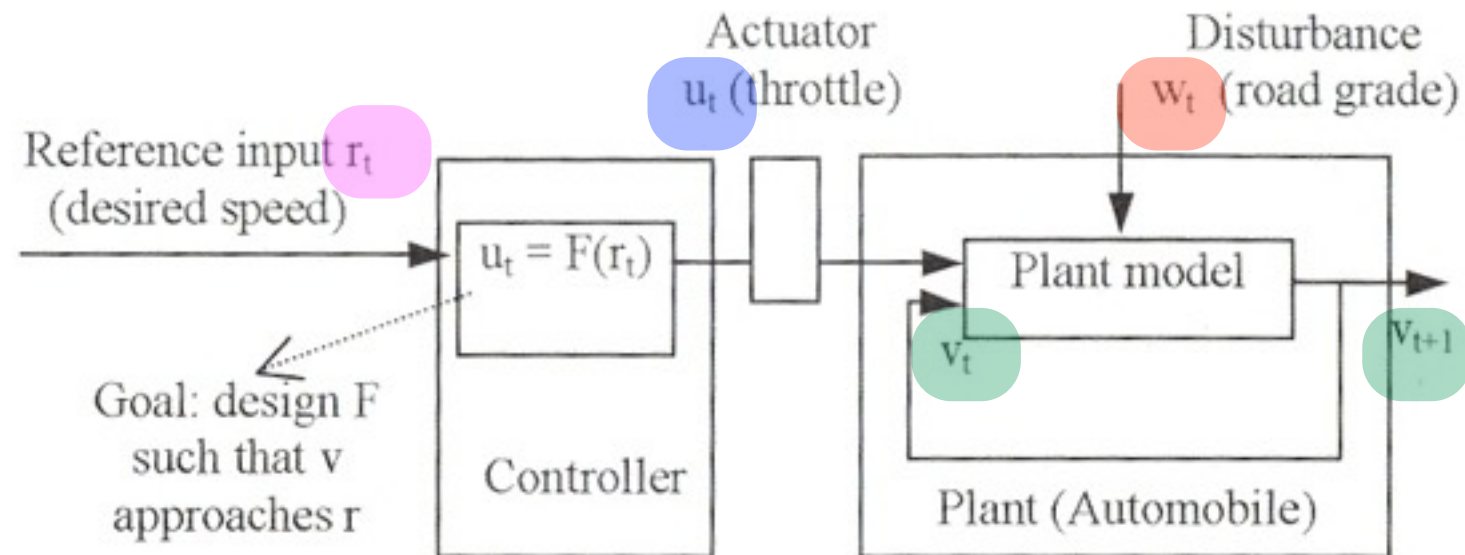
$$v_{t+1} = 0.7 \times v_t + 0.5 \times u_t \quad \left. \right\} \text{Car}$$

Hence:

$$v_{t+1} = 0.7 \times v_t + 0.5 \times P \times r_t$$



Open loop

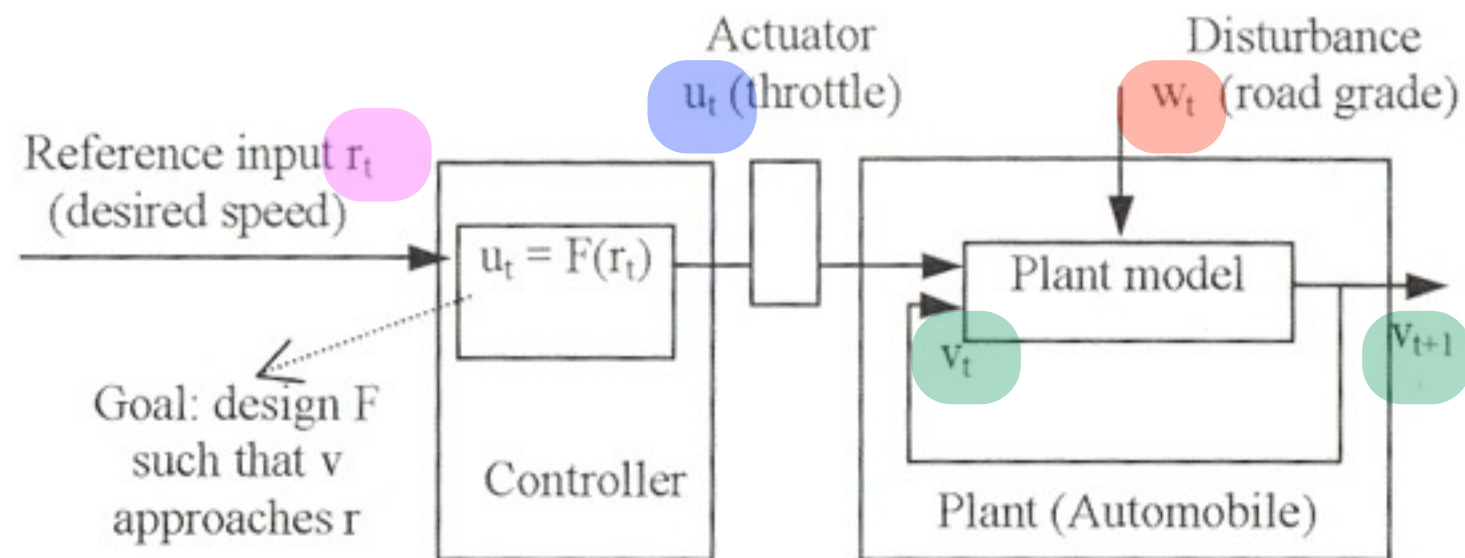


$$v_{t+1} = 0.7 \times v_t + 0.5 \times P \times r_t$$

- Now, we have to fix a **control objective**
- We can't impose $v_{t+1} = r_t$ for any t , because the car takes time to **accelerate** or **slow down**



Open loop

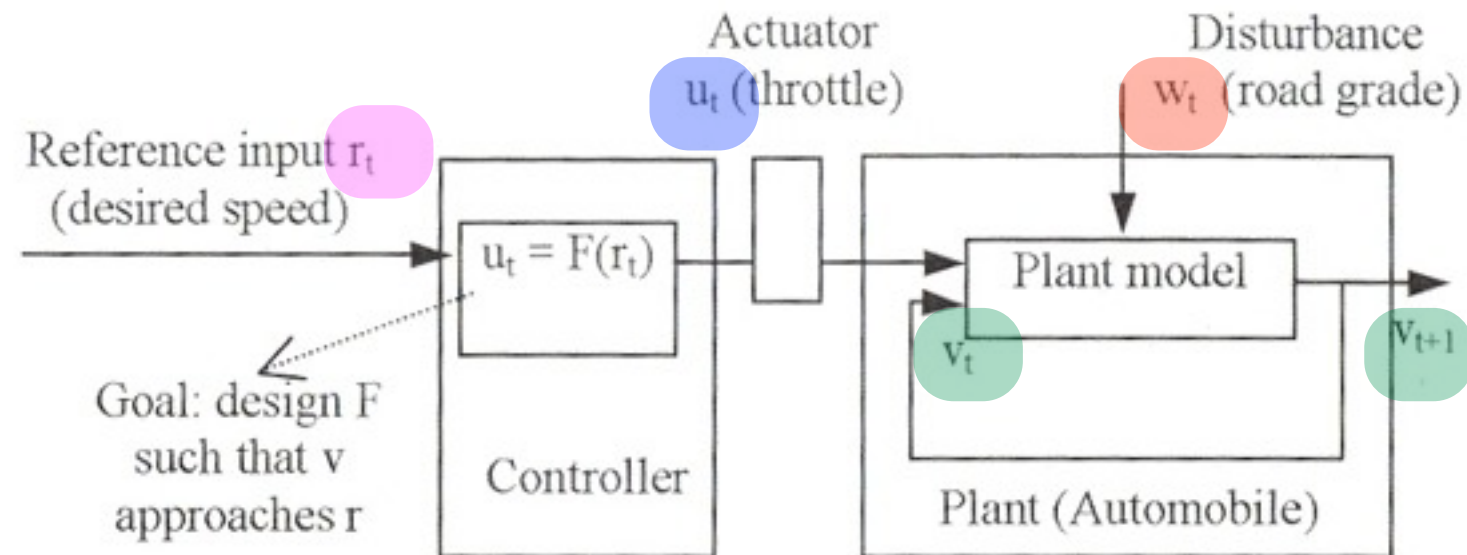


$$v_{t+1} = 0.7 \times v_t + 0.5 \times P \times r_t$$

- Now, we have to fix a **control objective**
- Let us assume $r_t = r$ for any t
- The control objective will be: **the output speed must be equal to the reference if the output is stable**, that is if $v_t = v_{t+1} = v_{ss}$



Open loop



$$v_{t+1} = 0.7 \times v_t + 0.5 \times P \times r$$

$$v_{t+1} = v_t = v_{ss}$$

Hence:

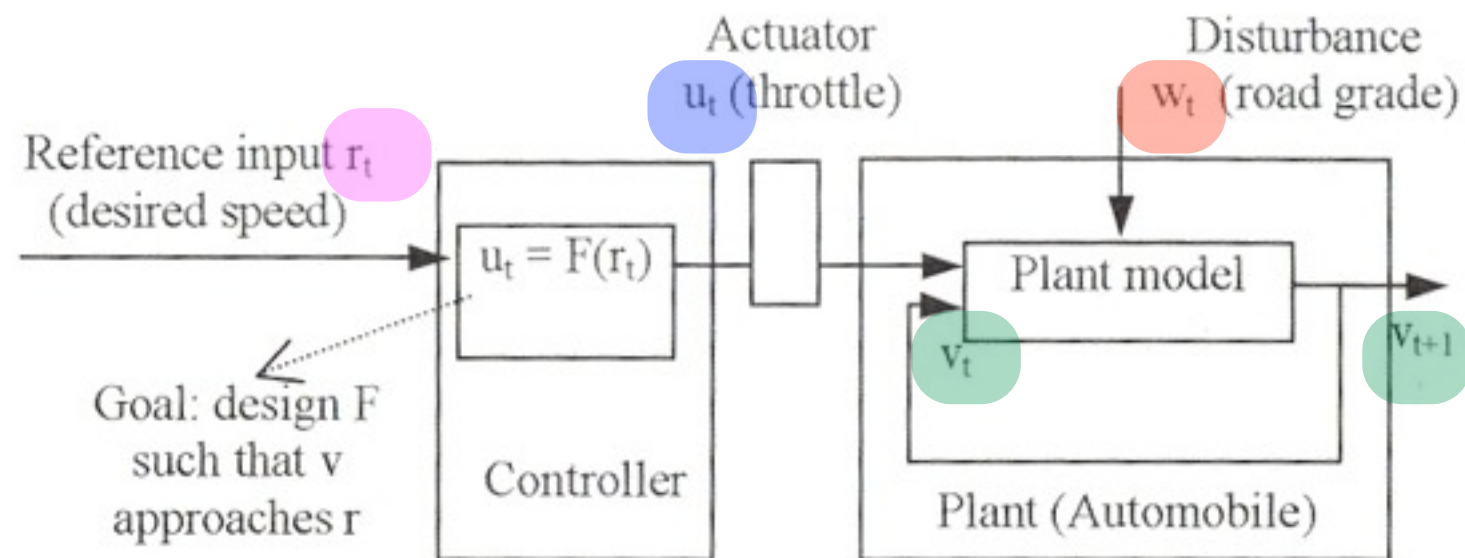
$$v_{ss} = 0.7 \times v_{ss} + 0.5 \times P \times r$$

Hence:

$$v_{ss} = 1.67 \times P \times r$$



Open loop



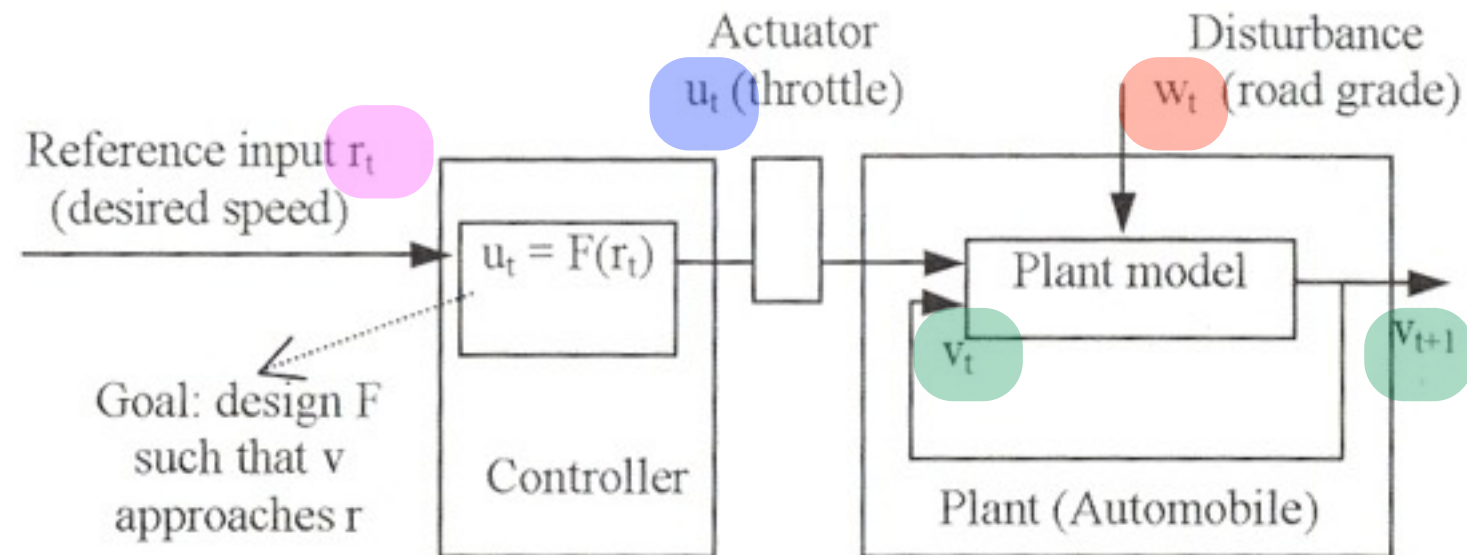
$$v_{ss} = 1.67 \times P \times r$$

That equation describes **all the speeds**
for which we want
the output speed to be equal to the reference speed:

$$v_{ss} = r$$



Open loop



$$v_{ss} = 1.67 \times P \times r$$

We want: $v_{ss} = r$

Thus, we choose $P = 1/1.67 = 0.6$



Example

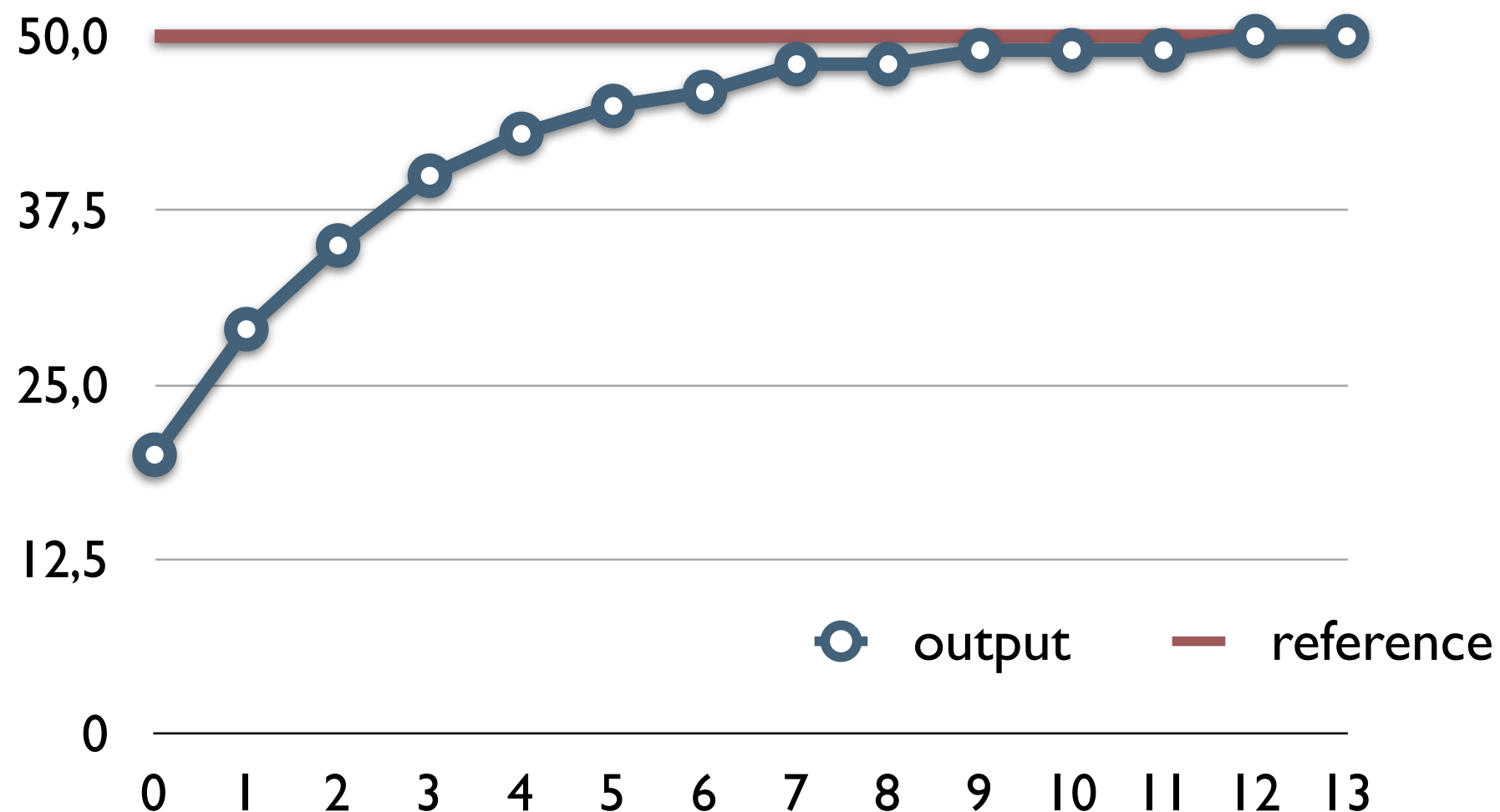
- Initial speed is 20 km/h, reference speed is $r=50$ km/h.
- $v_{t+1}=0.7 \times v_t+0.5 \times P \times r$ with $P=0.6$

t	v_t	t	v_t
0	20	7	47,529371
1	29	8	48,27056
2	35,3	9	48,789392
3	39,71	10	49,152574
4	42,797	11	49,406802
5	44,9579	12	49,584761
6	46,47053	13	49,709333



Example

- Initial speed is 20 km/h, reference speed is $r=50$ km/h.
- $v_{t+1}=0.7 \times v_t+0.5 \times P \times r$ with $P=0.6$

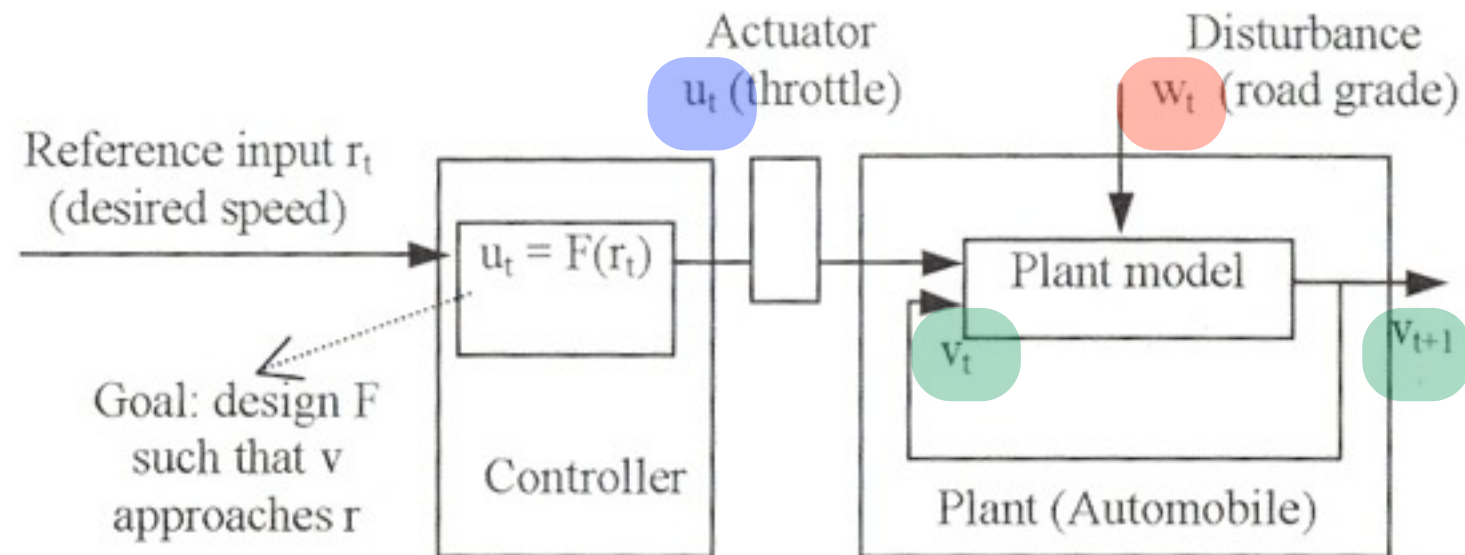


Drawbacks

- That example **works well** because the car is **highly predictable**
- The acceleration will be the same **in every possible condition**
- In practice, this is **not true** !
- We will now **refine** the model to take **disturbances** into account



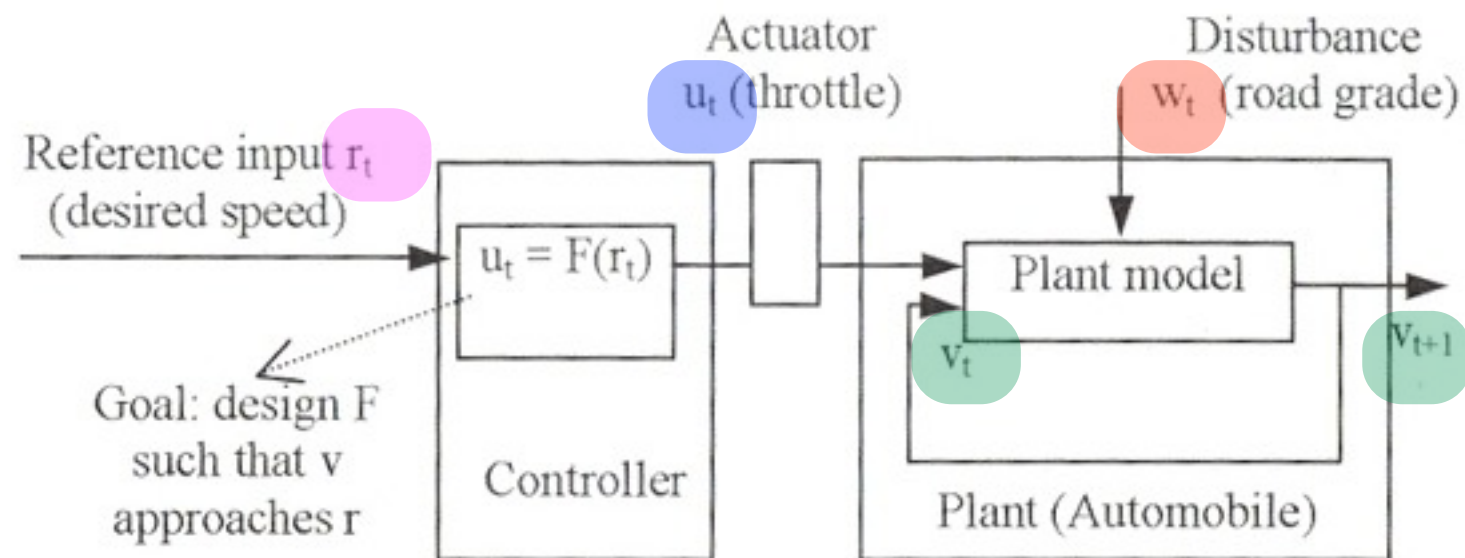
Open loop



- **New model:** $v_{t+1} = 0.7v_t + 0.5u_t - w_t$
- When the road grade is **negative**, the acceleration is **higher**



Open loop



- **New model:** $v_{t+1} = 0.7v_t + 0.5u_t - w_t$
- With the **same controller** we get:
 $v_{t+1} = 0.7 \times v_t + 0.3 \times r_t - w_t$



Example

- Initial speed is 20 km/h, reference speed is $r=50$ km/h.
- $v_{t+1} = 0.7 \times v_t + 0.3 r - w$ for $w = 5$

t	v_t	t	v_t
0	20	7	32,235276
1	24	8	32,564693
2	26,8	9	32,795285
3	28,76	10	32,9567
4	30,132	11	33,06969
5	31,0924	12	33,148783
6	31,76468	13	33,204148



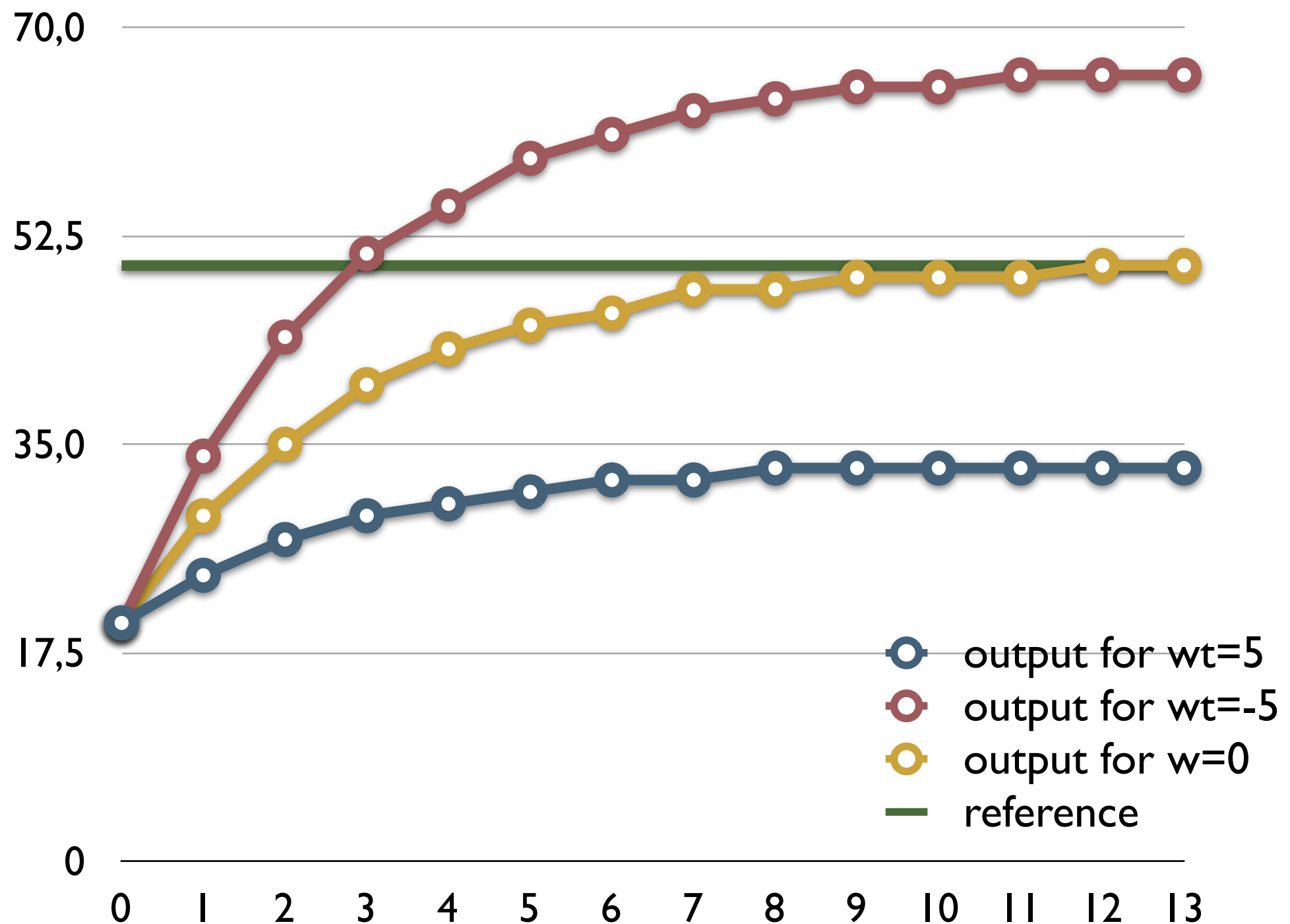
Example

- Initial speed is 20 km/h, reference speed is $r=50$ km/h.
- $v_{t+1}=0.7 \times v_t + 0.3 r - w$ for $w = -5$

t	v_t	t	v_t
0	20	7	62,823466
1	34	8	63,976426
2	43,8	9	64,783498
3	50,66	10	65,348449
4	55,462	11	65,743914
5	58,8234	12	66,02074
6	61,17638	13	66,214518



Example



Remark on the model

$$v_{t+1} = 0.7 \times v_t + 0.3 r - w$$

$$v_1 = 0.7 \times v_0 + 0.3 r - w$$

$$\begin{aligned} v_2 &= 0.7 \times (0.7 \times v_0 + 0.3 r - w) + 0.3 r - w \\ &= 0.7^2 \times v_0 + (0.7 + 1) \times 0.3 \times r - (0.7 + 1) \times w \end{aligned}$$

$$\begin{aligned} v_3 &= 0.7 \times [0.7^2 \times v_0 + (0.7 + 1) \times 0.3 \times r \\ &\quad - (0.7 + 1) \times w] + 0.3 r - w \end{aligned}$$

$$\begin{aligned} &= 0.7^3 \times v_0 + (0.7^2 + 0.7 + 1) \times 0.3 \times r \\ &\quad - (0.7^2 + 0.7 + 1) \times w \end{aligned}$$



Remark on the model

In general:

$$v_t = 0.7^t \times v_0 + (1 + \sum_{1 \leq i \leq t-1} 0.7^i) \times 0.3 \times r - (1 + \sum_{1 \leq i \leq t-1} 0.7^i) \times w$$

At any time, the output depends on the initial speed
But this influence decreases quickly.

t	0	1	2	3	4	5	6
0.7^t	0,7	0,49	0,343	0,24	0,168	0,118	0,082



Remark on the model

$$v_{t+1} = 0.7^t \times v_0 + \dots$$

- The **0,7** value is denoted α and called the **decay rate** of v_0
- The **larger** the α the **greater** the **influence** of the initial speed in subsequent speeds



Remark on the model

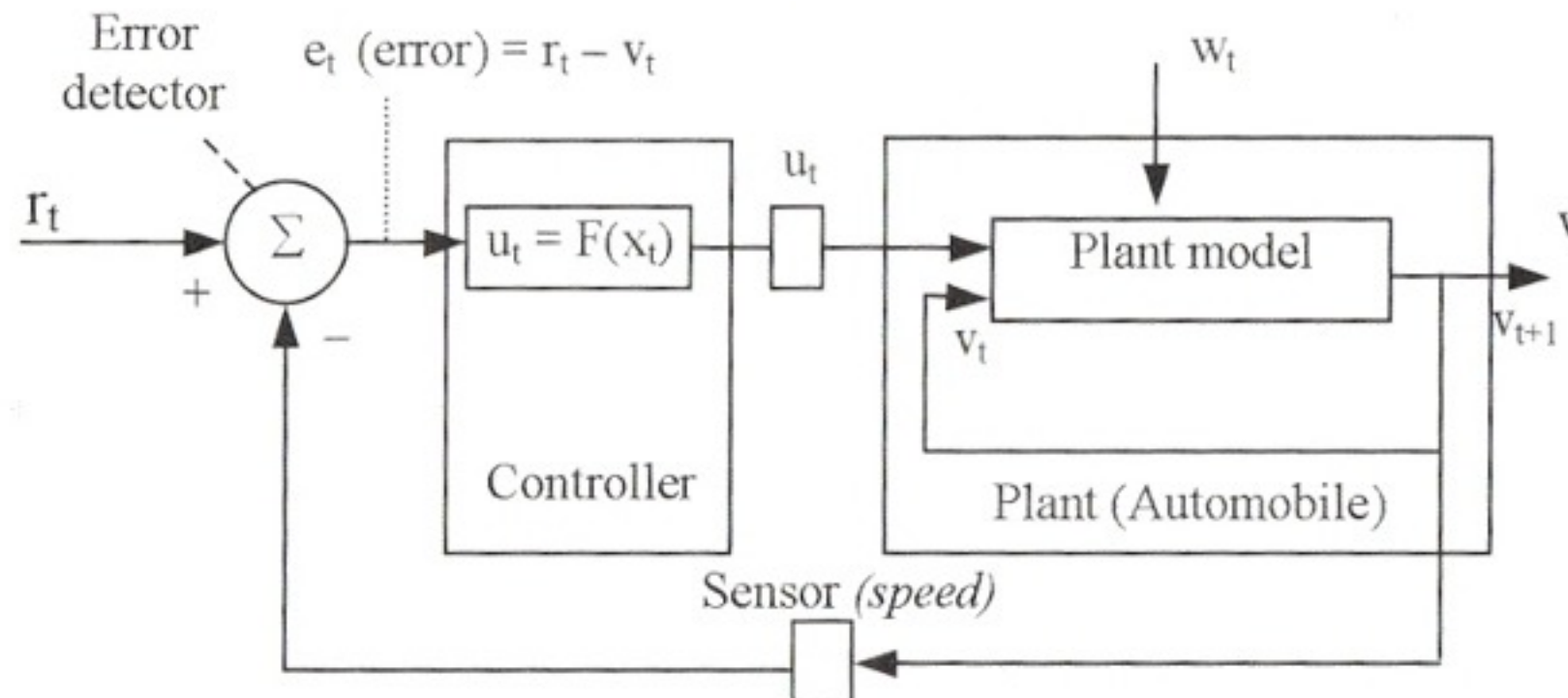
$$v_{t+1} = 0.7^t \times v_0 + \dots$$

- If $\alpha > 1$, the output **grows forever**, and does not stabilise !
- If α is **negative**, the output will **oscillate** !
- This will be important when dealing with closed loop controllers



Drawbacks

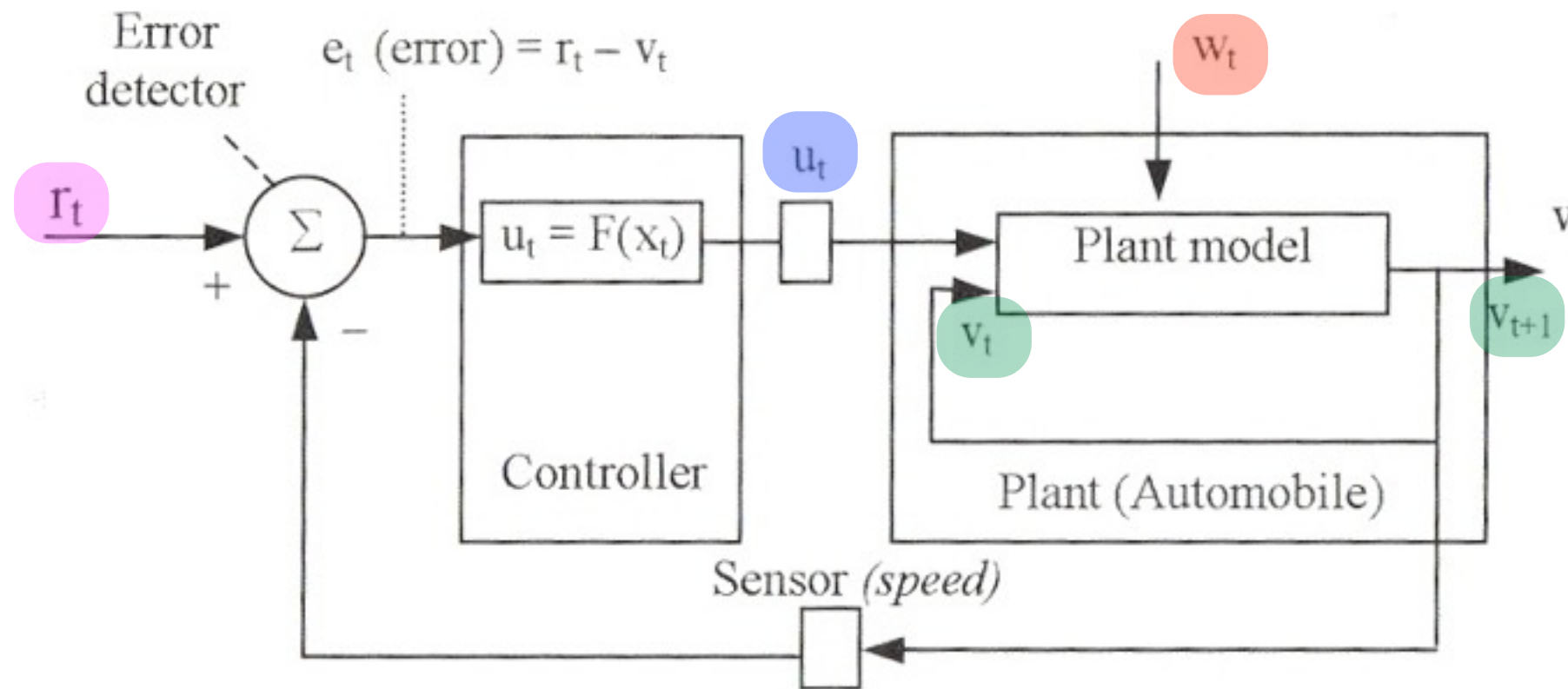
- To obtain a **more accurate** controller, we'll introduce a **feedback loop**
- The controller will **know** the **actual speed**, which in turn depends on the road grade, and will be able to **adapt** its policy to these **disturbances**



The background of the slide features a large, light blue circular seal of the University of Brussels. The seal contains a central emblem with a sunburst at the top, two torches crossed in the middle, and two lions at the bottom. The Latin motto "SCIENTIA VINCERE TENEBRAS" is inscribed along the top inner edge, and "UNIVERSITAS BRUXELLENSIS" is inscribed along the bottom inner edge.

Closed loop controller

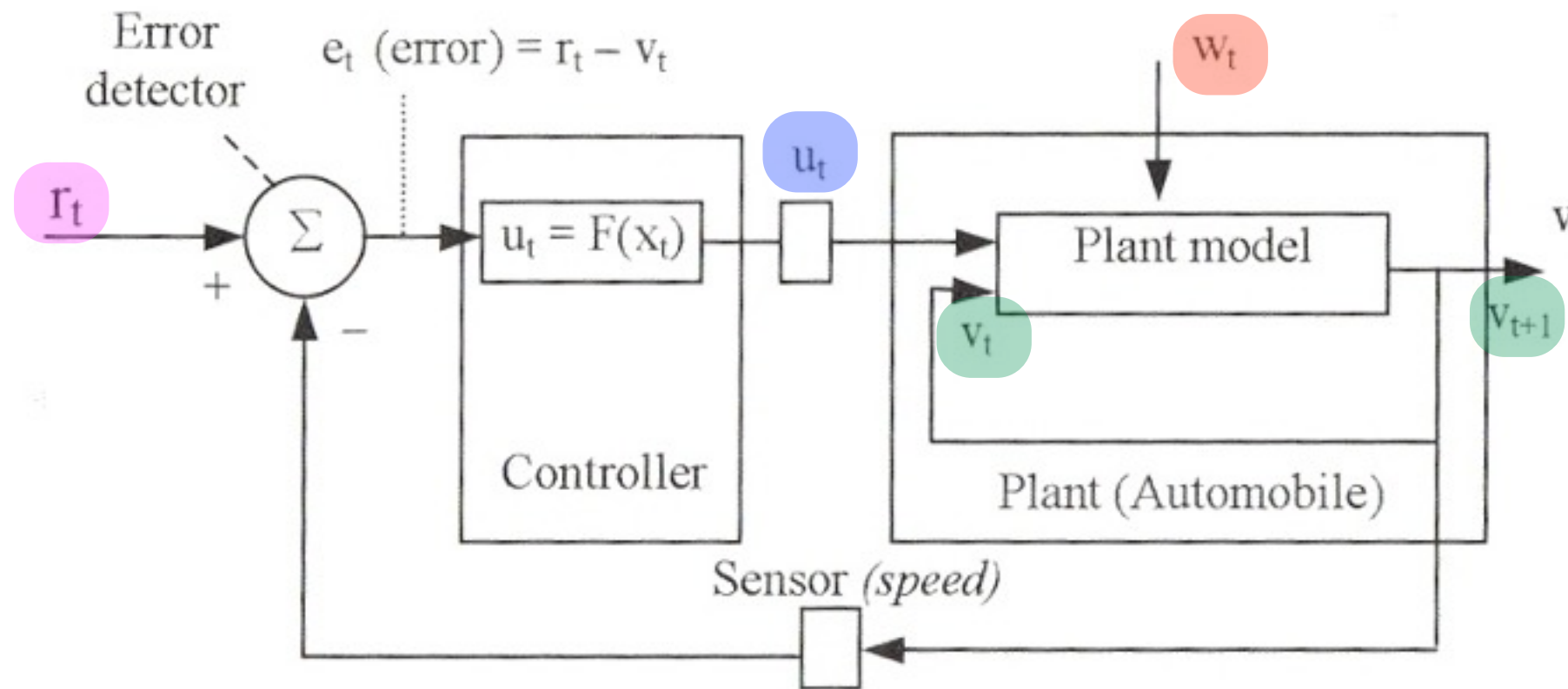
Closed loop



- **Remark:** The input to the controller is the **difference** between the desired speed and the current speed



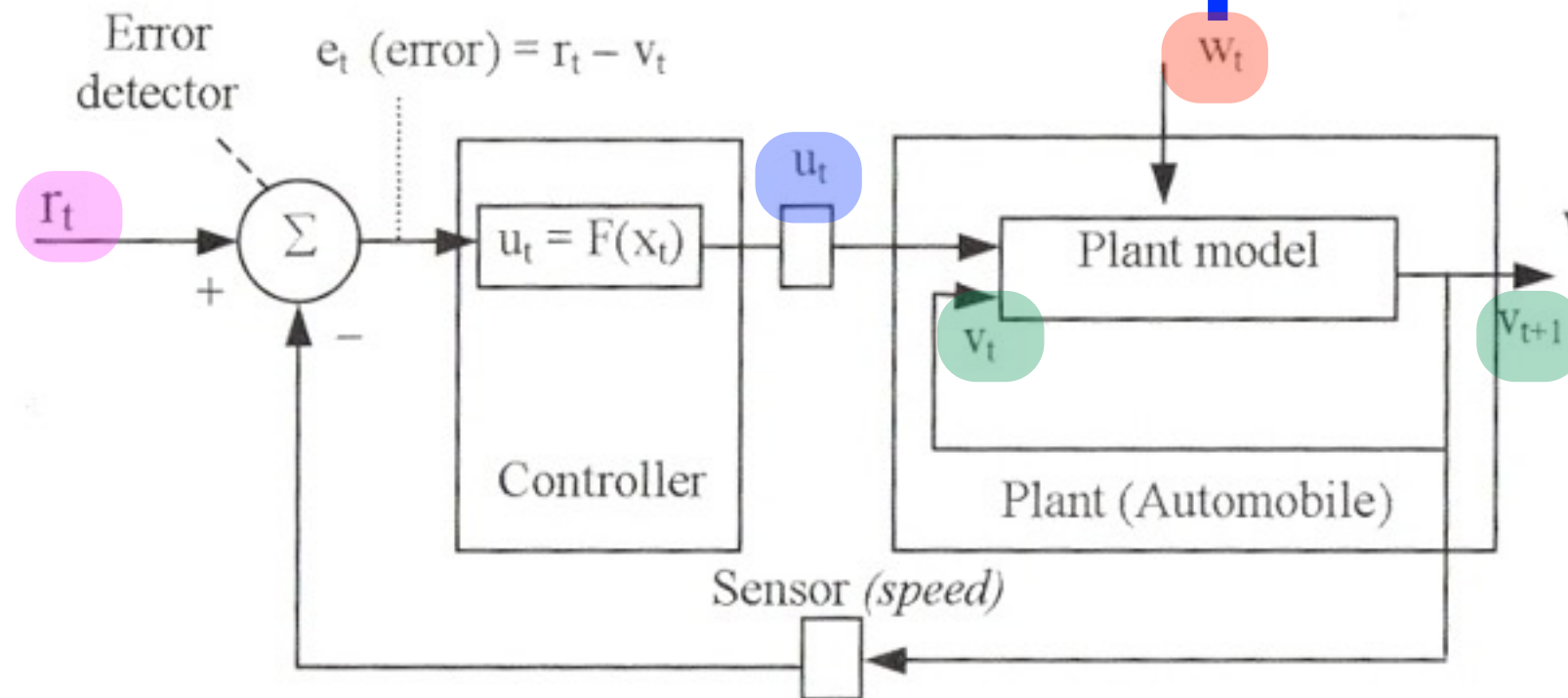
Closed loop



- Let us assume once again that the controller is a linear function $F(r_t - v_t) = P \times (r_t - v_t)$
- We keep $v_{t+1} = 0,7v_t + 0,5u_t - w_t$ as a model for the car



Closed loop



$$\left. \begin{aligned} F(r_t - v_t) &= P \times (r_t - v_t) \\ u_t &= F(r_t - v_t) \end{aligned} \right\} \text{Controller}$$

$$v_{t+1} = 0,7v_t + 0,5u_t - w_t \quad \left. \right\} \text{Car}$$

Hence:

$$v_{t+1} = 0,7 \times v_t + 0,5 \times P \times (r_t - v_t) - w_t$$



Closed loop

$$\begin{aligned} v_{t+1} &= 0,7 \times v_t + 0,5 \times P \times (r_t - v_t) - w_t \\ &= (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t \end{aligned}$$

- Now, the P parameter appears in the expression of v_0 :
 $\alpha = (0,7 - 0,5 \times P)$
- We have to take that in to account to keep $0 \leq \alpha < 1$
- This means we want $-0,6 < P \leq 1,4$



Examples

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t$$

$$w_t = 0 \quad P = -0,5 \quad \alpha = (0,7 - 0,5 \times P) = 0,95$$

t	v_t	t	v_t
0	20	7	-63,07519
1	6,5	8	-72,42143
2	-6,325	9	-81,30036
3	-18,50875	10	-89,73534
4	-30,08331	11	-97,74857
5	-41,07915	12	-105,3611
6	-51,52519	13	-112,5931



Examples

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t$$

$$w_t = 0 \quad P = -0,5 \quad \alpha = (0,7 - 0,5 \times P) = 0,95$$

By choosing

$$F(r_t - v_t) = -0,5 \times (r_t - v_t)$$

we have a controller that
decelerates when $r_t > v_t$...

Speeds < 0 don't make any
sense !



Examples

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t$$

$$w_t = 0 \quad P = 1 \quad \alpha = (0,7 - 0,5 \times 1) = 0,2$$

t	v_t	t	v_t
0	20	7	31,249856
1	29	8	31,249971
2	30,8	9	31,249994
3	31,16	10	31,249999
4	31,232	11	31,25
5	31,2464	12	31,25
6	31,24928	13	31,25



Examples

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t$$

$$w_t = 0 \quad P = 1,4 \quad \alpha = (0,7 - 0,5 \times 1,4) = 0$$

t	v_t	t	v_t
0	20	7	35
1	35	8	35
2	35	9	35
3	35	10	35
4	35	11	35
5	35	12	35
6	35	13	35



Examples

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t$$

$$w_t = 0 \quad P = 1,4 \quad \alpha = (0,7 - 0,5 \times 1,4) = 0$$

t	v _t	t	v _t
0	0	0	0
1	2	1	2
2	4	2	4
3	6	3	6
4	8	4	8
5	10	5	10
6	12	6	12
7	14	7	14
8	16	8	16
9	18	9	18
10	20	10	20
11	22	11	22
12	24	12	24
13	26	13	26
14	28	14	28
15	30	15	30
16	32	16	32
17	34	17	34
18	36	18	36
19	38	19	38
20	40	20	40
21	42	21	42
22	44	22	44
23	46	23	46
24	48	24	48
25	50	25	50
26	52	26	52
27	54	27	54
28	56	28	56
29	58	29	58
30	60	30	60
31	62	31	62
32	64	32	64
33	66	33	66
34	68	34	68
35	70	35	70

That controller stabilises
immediately at $0,7 \times r_t$



Examples

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t$$

$$w_t = 0 \quad P = 3 \quad \alpha = (0,7 - 0,5 \times 3) = -0,8$$

t	v_t	t	v_t
0	20	7	46,210496
1	59	8	38,031603
2	27,8	9	44,574717
3	52,76	10	39,340226
4	32,792	11	43,527819
5	48,7664	12	40,177745
6	35,98688	13	42,857804



Examples

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r_t - w_t$$

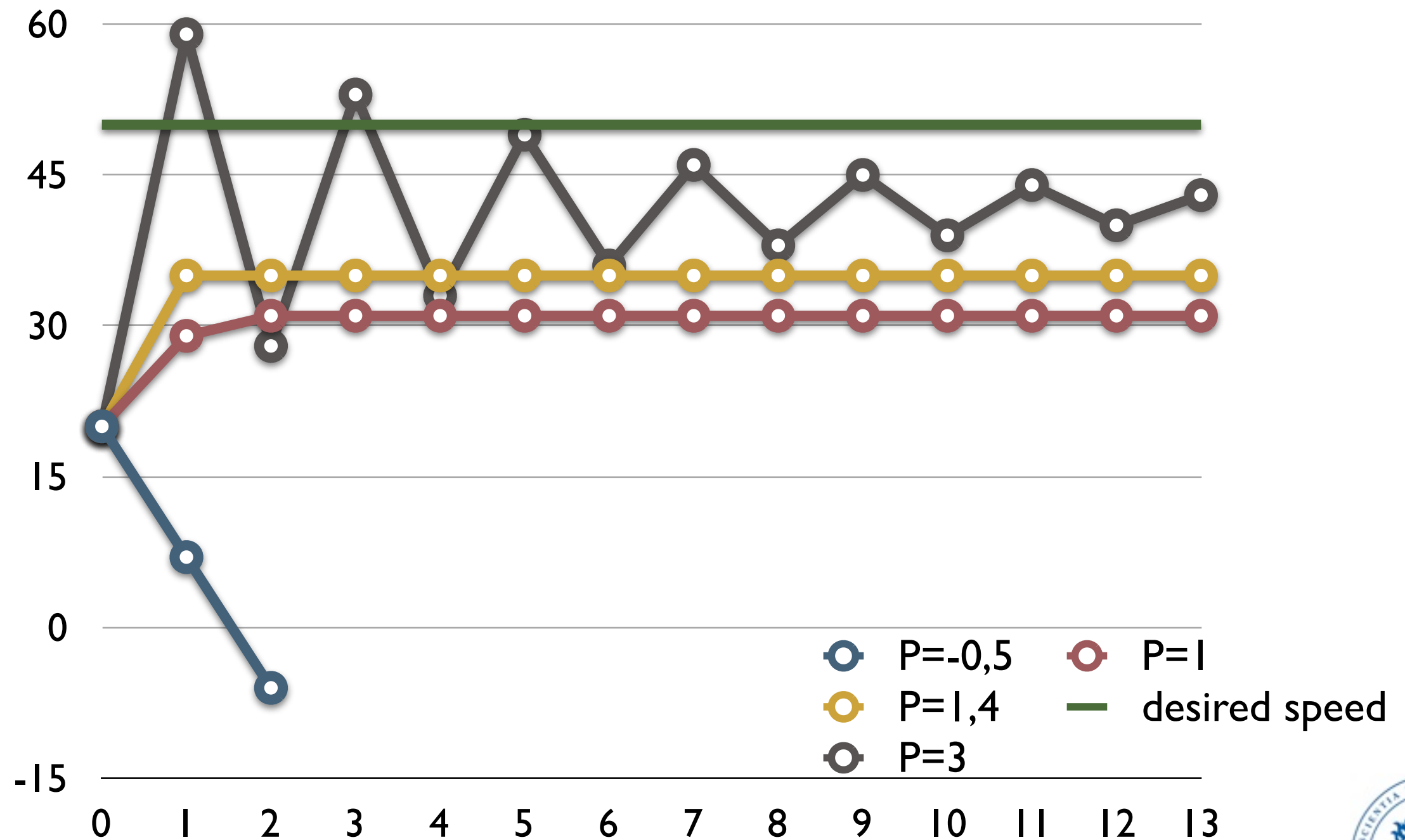
$$w_t = 0 \quad P = 3 \quad \alpha = (0,7 - 0,5 \times 3) = -0,8$$

t	v_t	t	v_t
0	20	7	46,210496
1	32,772	8	603
2	48,7664	9	717
3	35,98688	10	226
4	52,772	11	319
5	48,7664	12	40,177745
6	35,98688	13	42,857804

With a negative α
oscillations appear



Examples



To sum up

- Up to now, we have only considered the effect of P on α
- If $P \leq -0,6$, then $\alpha \geq 1$ and the system does not converge
- If $-0,6 < P \leq 0$, the controller is «reversed»
- If $0 < P \leq 1,4$: quick stability but large error, (15 km/h for $P=1,4$; 25 km/h for $P=0,5$)
- If $1,4 < P < 3,4$ the $\alpha < 0$ and the system oscillates but the error is low
- If $P \geq 3,4$, then $\alpha \geq 1$ and the system does not converge



Steady states

- Let us now fix a **control objective**: that the **v** speed be equal to the requested speed **r** in steady states

- **Steady states** are those where

$$v_{t+1} = v_t = v_{ss} = r$$

- By replacing v_t, v_{t+1} by v_{ss} in

$$v_{t+1} = (0,7 - 0,5 \times P) v_t + 0,5 \times P \times r - w_0$$

we get:

$$v_{ss} = \frac{0,5 \times P}{0,3 + 0,5 \times P} \times r - \frac{1}{0,3 + 0,5 \times P} \times w_0$$



Steady states

$$v_{ss} = \frac{0,5 \times P}{0,3 + 0,5 \times P} \times r - \frac{1}{0,3 + 0,5 \times P} \times w_0$$

- **First observation:** We now compute P by taking into account the perturbation w_0 (which is supposed constant)
- The best value for P thus depends upon w_0 : we can't find a single value that allows perfect control in every situation !



Steady states

$$v_{ss} = \frac{0,5 \times P}{0,3 + 0,5 \times P} \times r - \frac{1}{0,3 + 0,5 \times P} \times w_0$$

- We'll try to **minimise** the influence of perturbation by having the **2nd term as small** as possible
- We **want** to have $1/(0,3 + 0,5 \times P) < 1$
 - this **implies** $P > 1.4$
 - but then $\alpha \leq 0$, and we have **oscillations** !



Steady states

$$v_{ss} = \frac{0,5 \times P}{0,3 + 0,5 \times P} \times r - \frac{1}{0,3 + 0,5 \times P} \times w_0$$

- Moreover, we'd like $v_{ss} = r$
- To achieve this, the coefficient of the first term should be as close to 1 as possible
- Thus P should be as large as possible



Steady states

$$v_{ss} = \frac{0,5 \times P}{0,3 + 0,5 \times P} \times r - \frac{1}{0,3 + 0,5 \times P} \times w_0$$

P	coeff. 1	coeff. 2
0,5	0,4545454545	1,8181818182
1	0,625	1,25
2	0,7692307692	0,7692307692
10	0,9433962264	0,1886792453
100	0,9940357853	0,0198807157
10000	0,9999400036	0,000199988

Larger values of P are better !

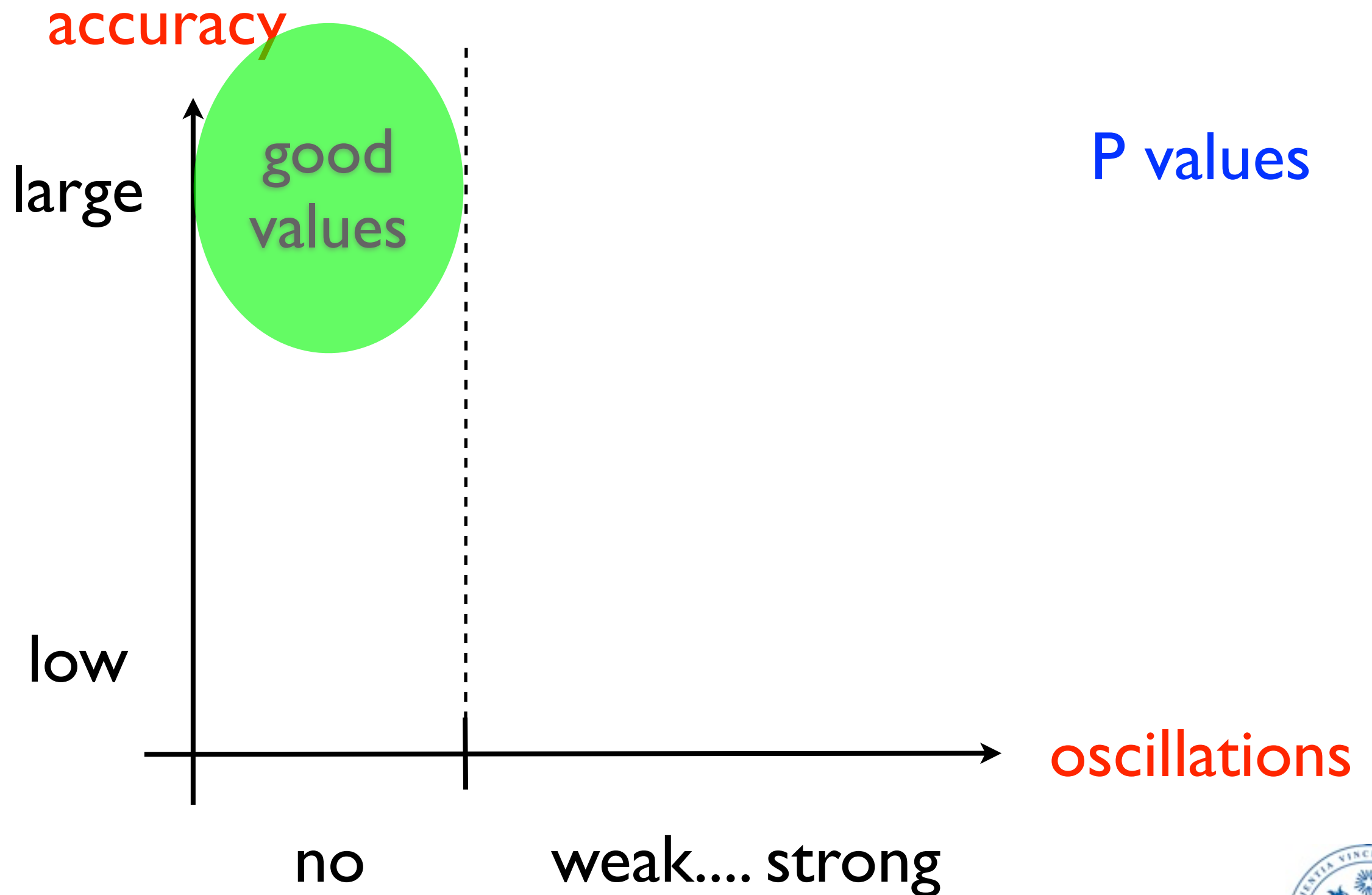


Steady states

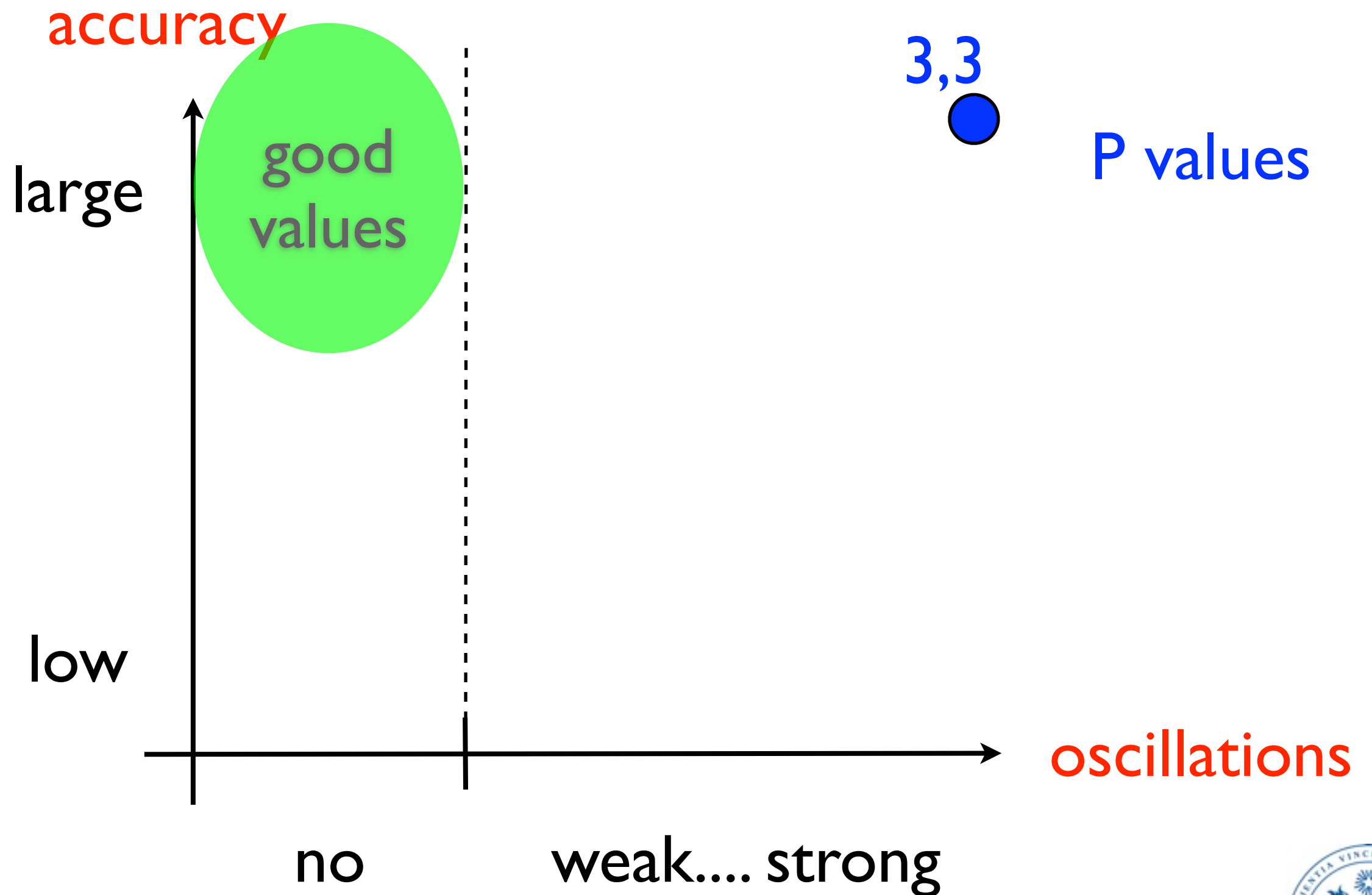
- We could try to use a P as big as possible...
- ... however, with $P > 3,4$ we get oscillations
- In this case, two control objectives are contradictory !
 - accuracy vs
 - smooth convergence (i.e. passenger comfort ;-)



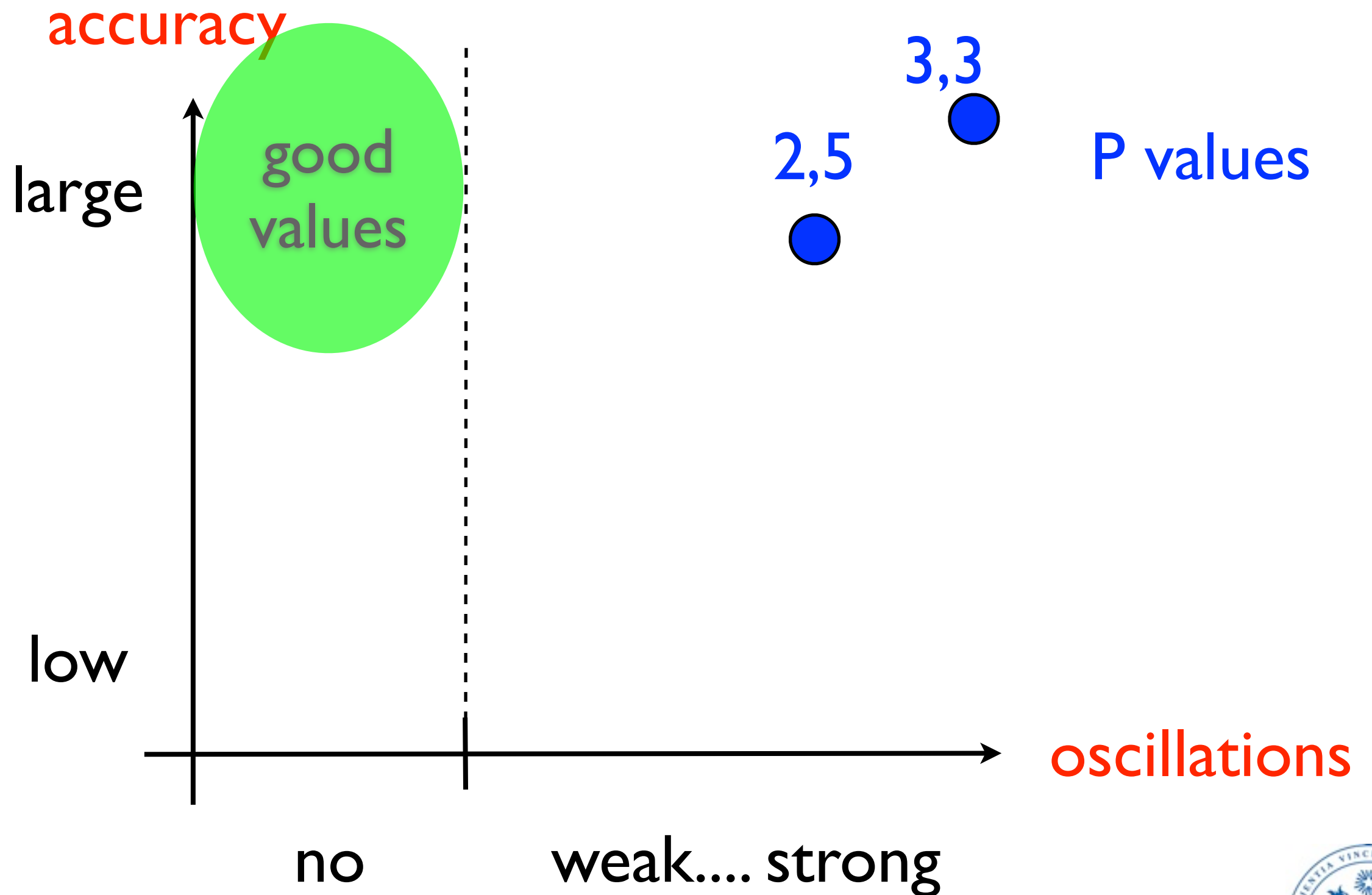
Control objectives



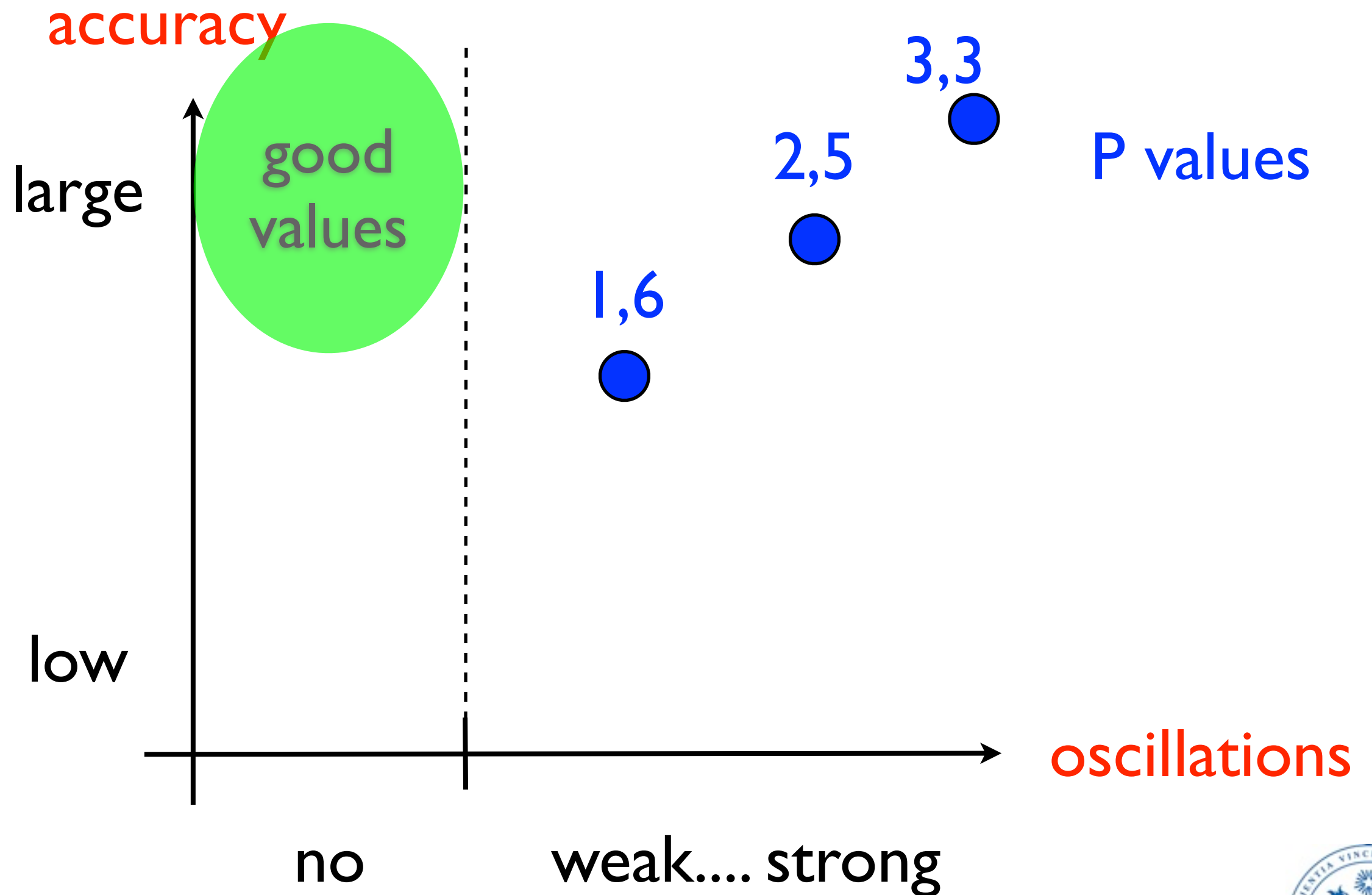
Control objectives



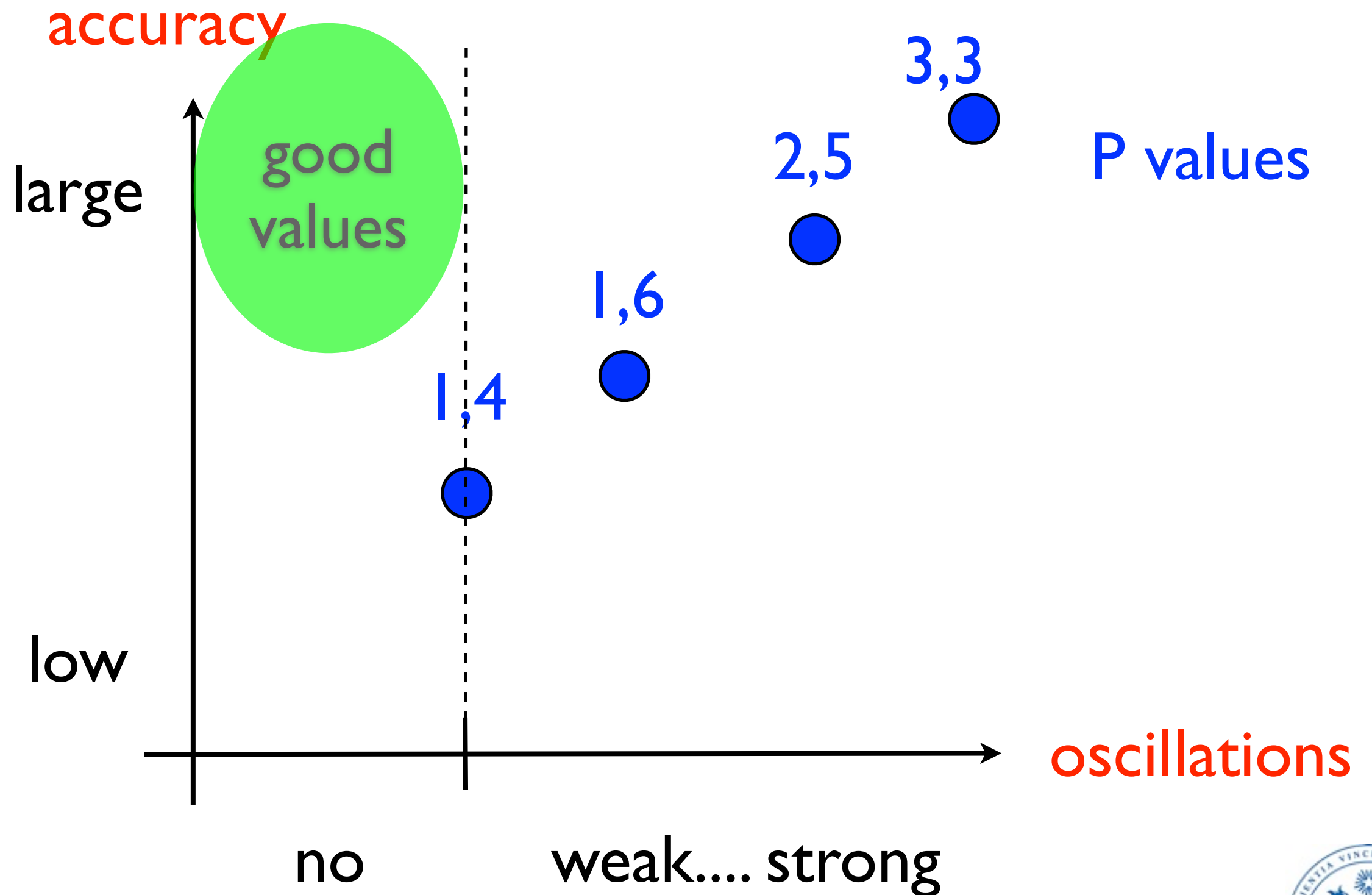
Control objectives



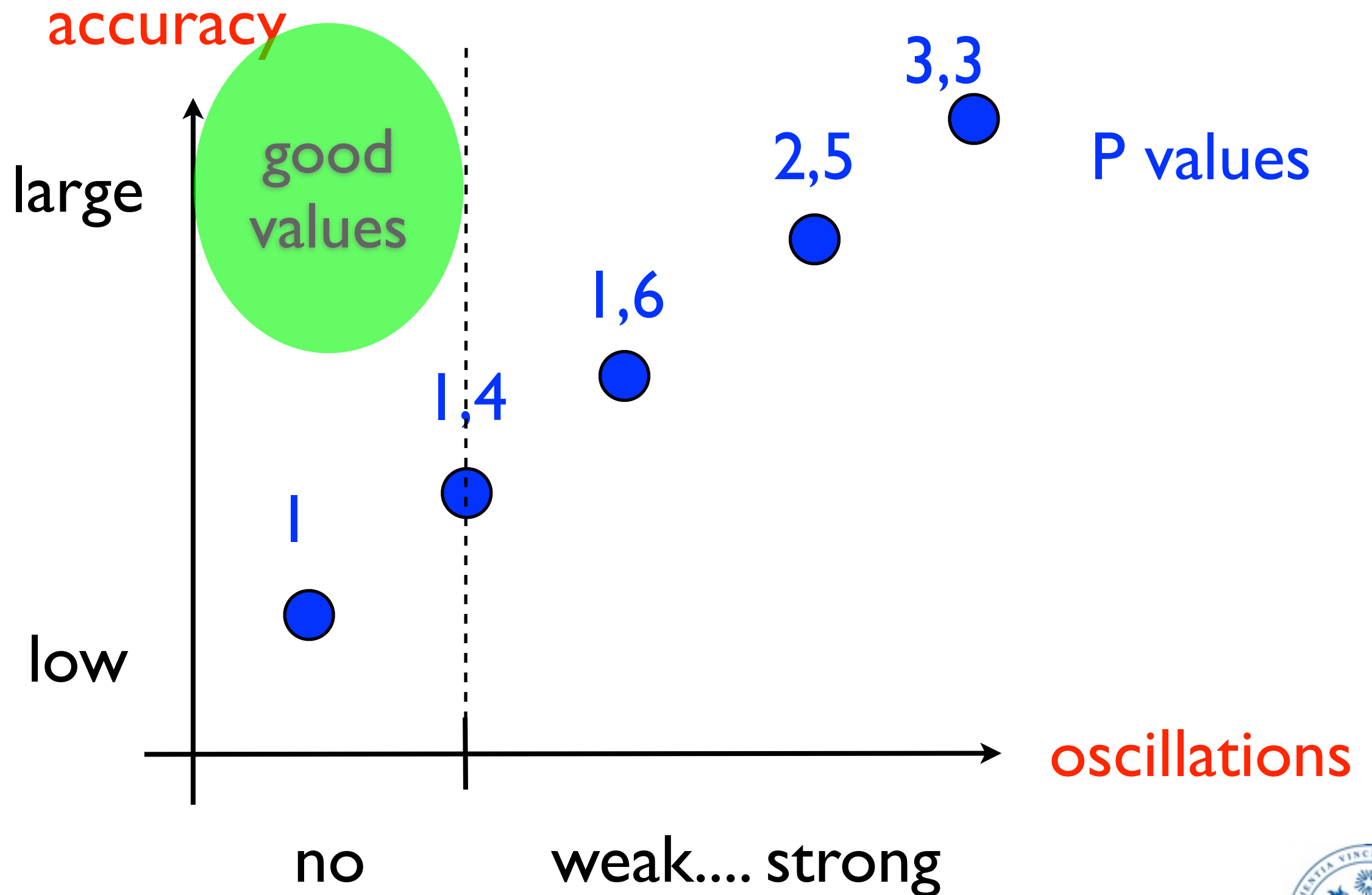
Control objectives



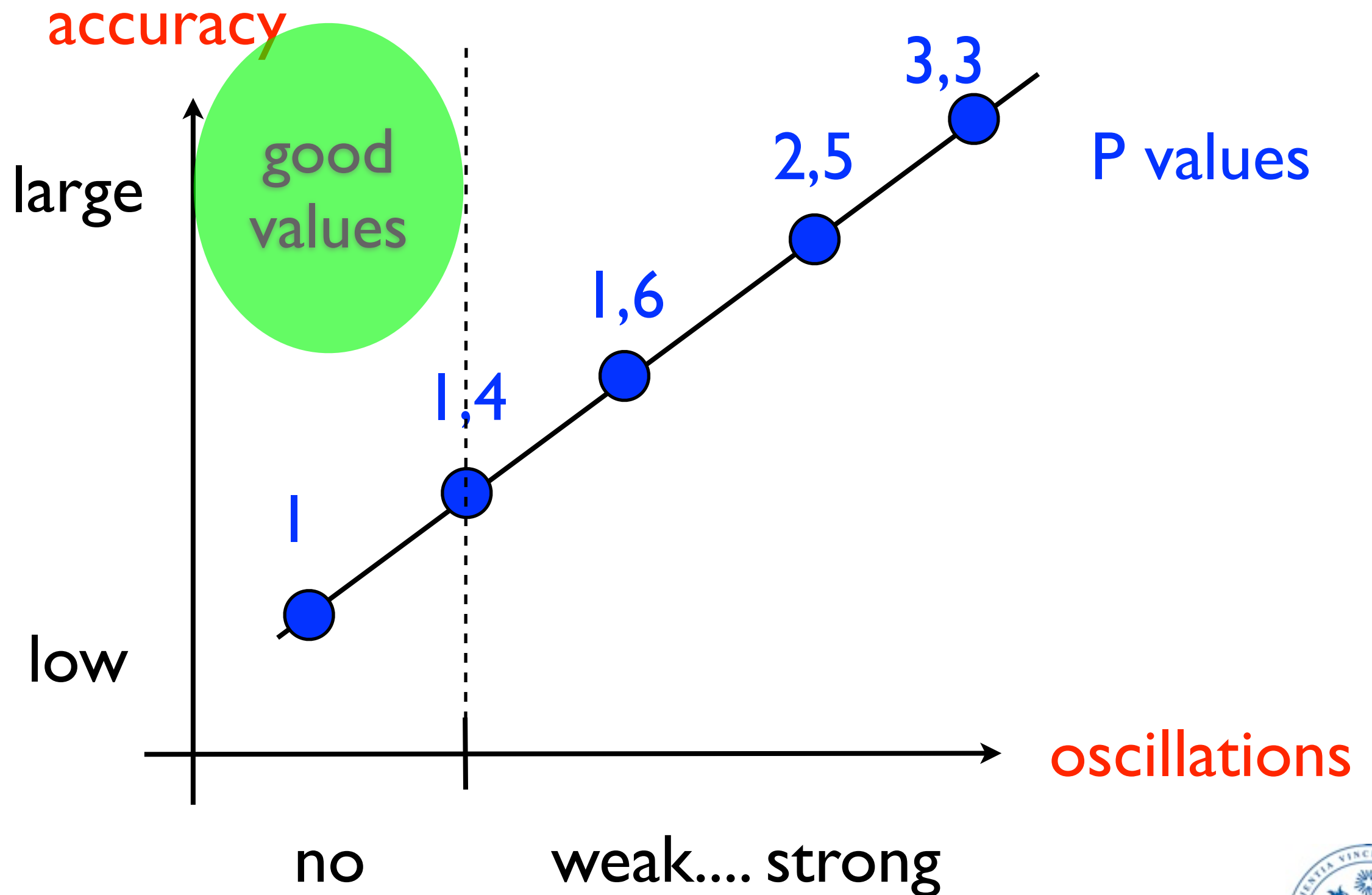
Control objectives



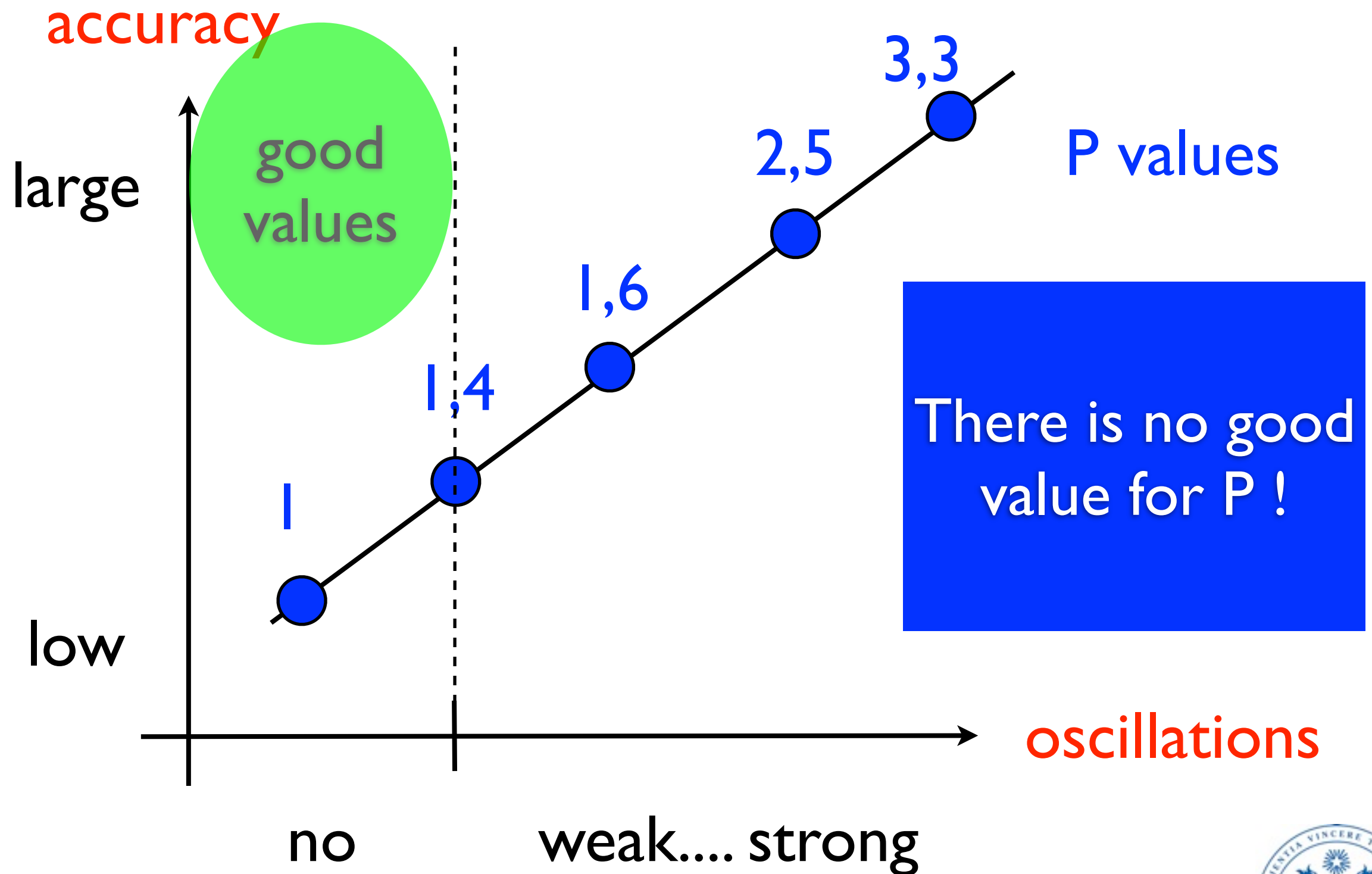
Control objectives



Control objectives



Control objectives



Contradictory objectives

- We have to take **decisions**:
 - With a value such as $P=3,3$ we **advantage** the **accuracy**, but the **oscillations** will threaten the passengers' comfort
 - With a value such as $P=1$ there are **no oscillations**, but the speed **stabilises** way **under** the **desired** speed
- Nothing better can be achieved with such a simplistic controller



Disturbances

- Let us remind we had introduced closed loop controllers to **cope correctly with disturbances**
- It is the case ?



Disturbances

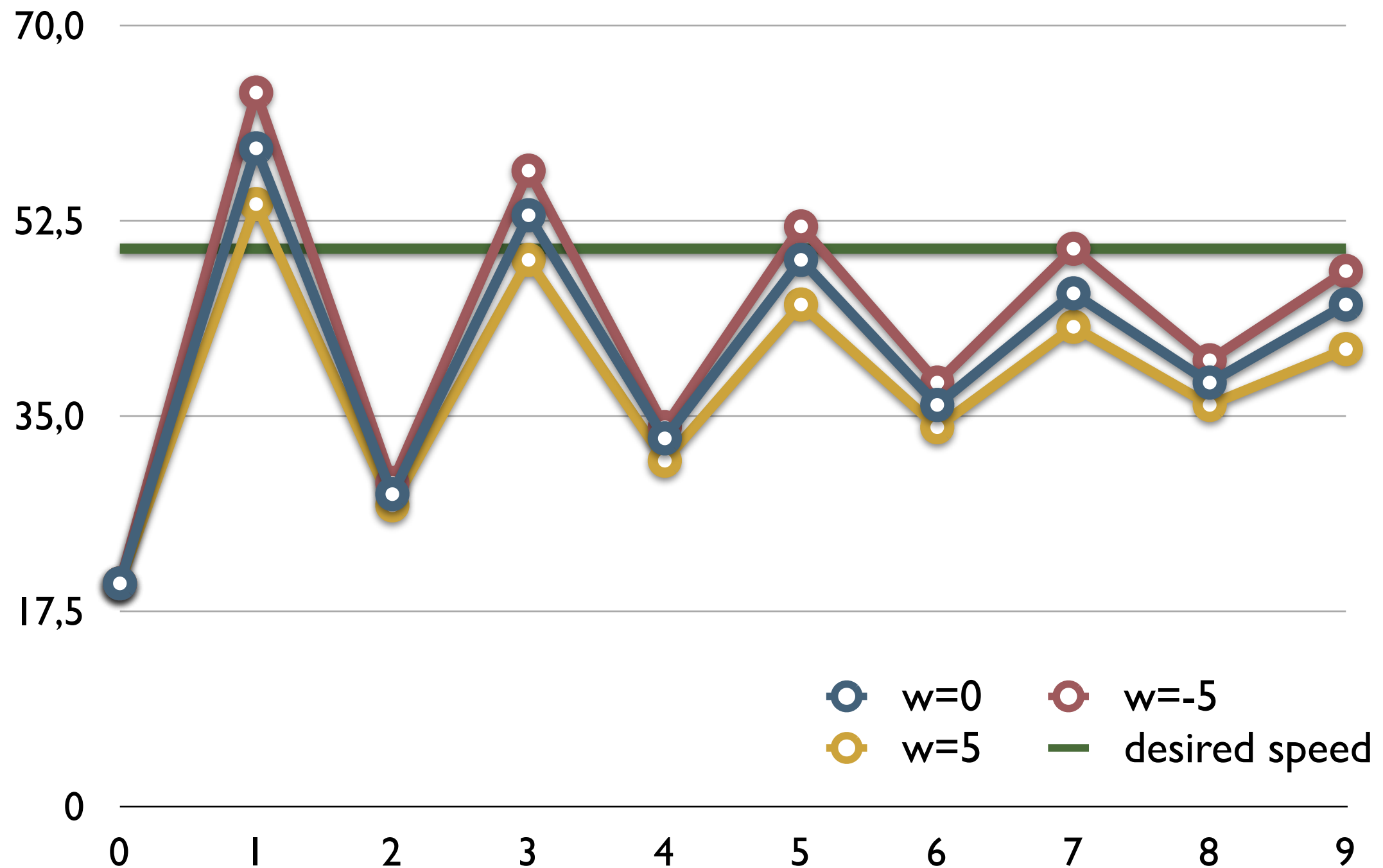
$$P = 3 \quad r = 50 \quad v_0 = 20$$

t	v for w=0	v for w=-5	v for w=5
0	20	20	20
1	59	64	54
2	28	29	27
3	53	57	49
4	33	34	31
5	49	52	45
6	36	38	34
7	46	50	43
8	38	40	36
9	45	48	41



Disturbances

$$P = 3 \quad r = 50 \quad v_0 = 20$$



Disturbances

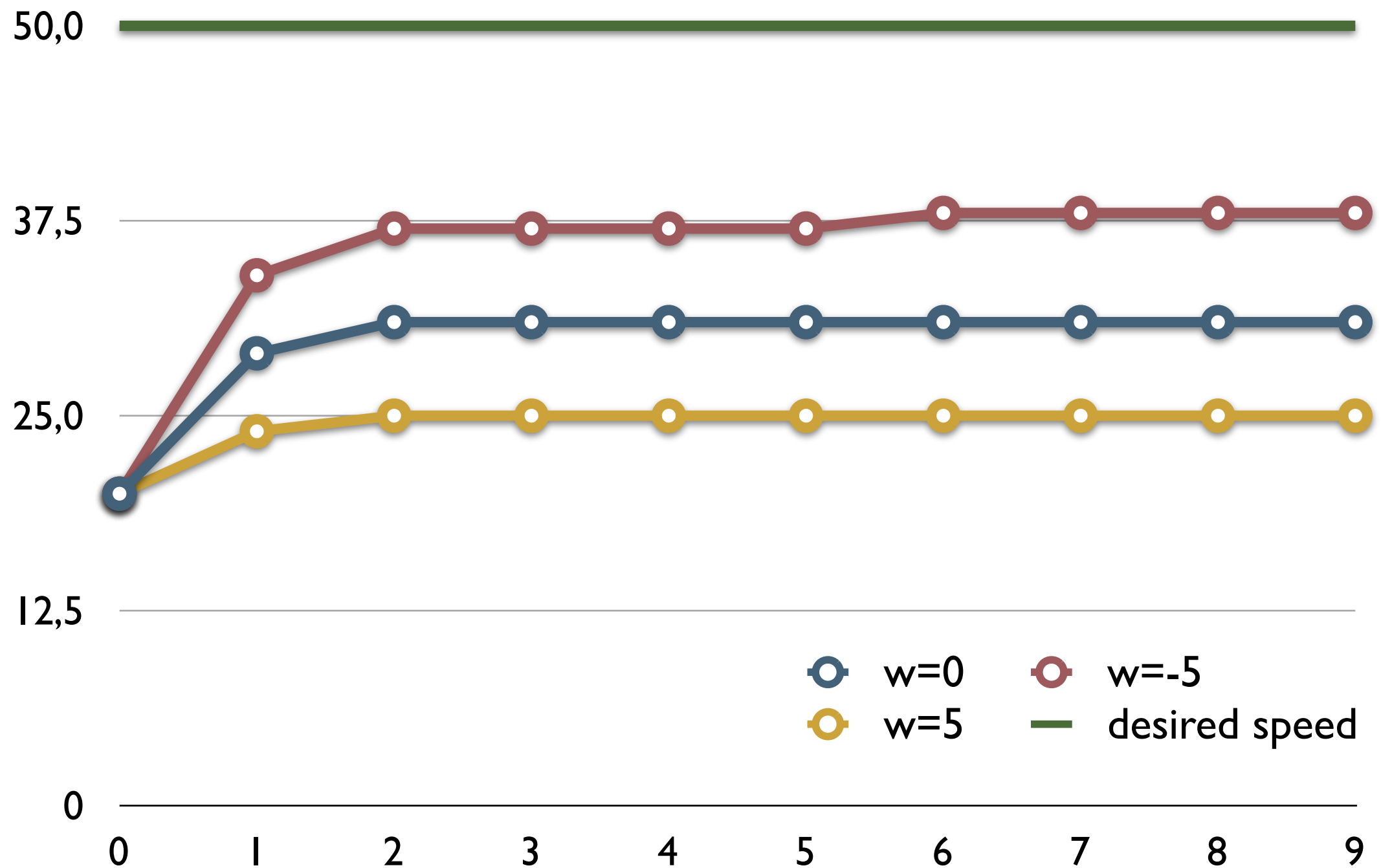
$$P = 1 \quad r = 50 \quad v_0 = 20$$

t	v for w=0	v for w=-5	v for w=5
0	20	20	20
1	29	34	24
2	31	37	25
3	31	37	25
4	31	37	25
5	31	37	25
6	31	38	25
7	31	38	25
8	31	38	25
9	31	38	25



Disturbances

$$P = I \quad r = 50 \quad v_0 = 20$$



Disturbances

- One can see that the plots for the different w values are indeed much closer
- In the $P=3$ case, they converge to very close values.





PID Controllers

PID

- Let us now study a more general form of closed loop controller: **PID controller**
- We will **discuss** with more details the **control objectives**.



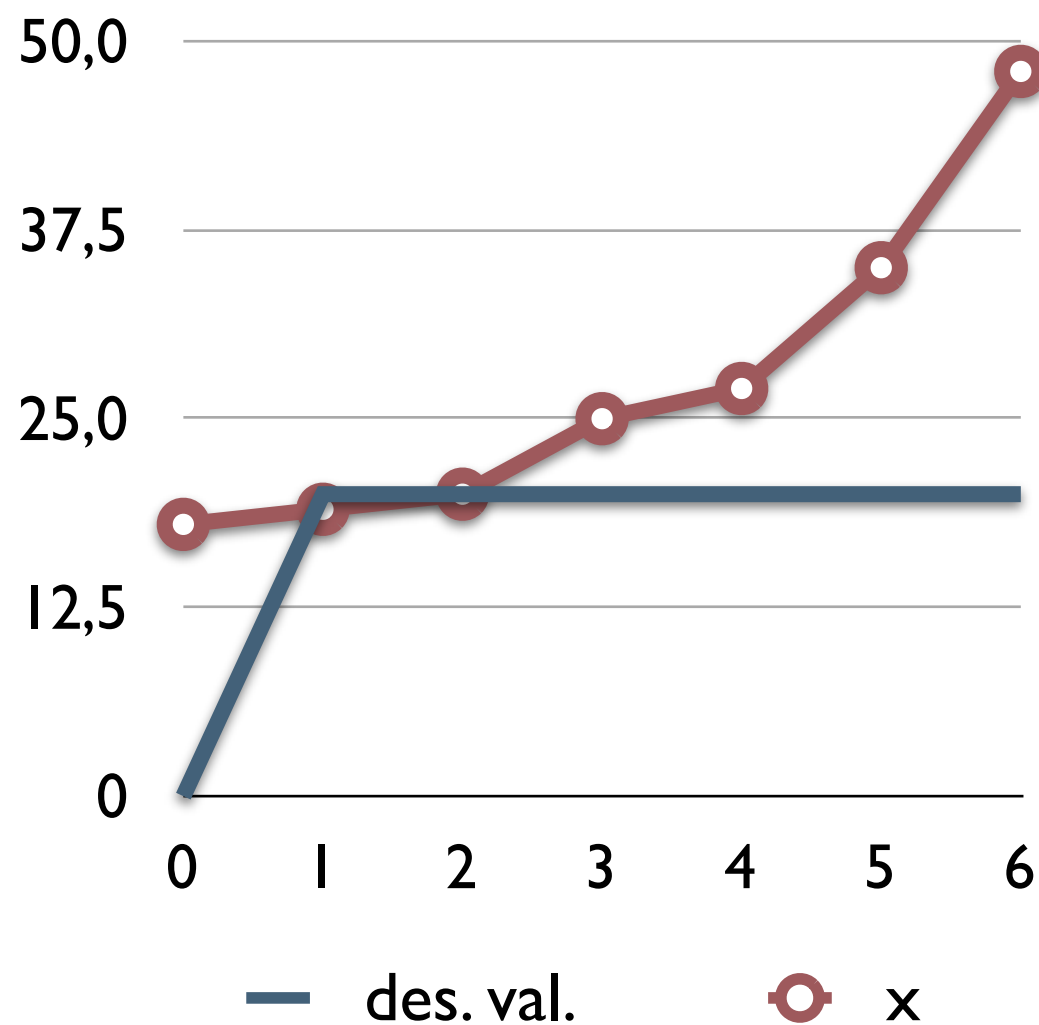
Control objectives

- **Stability**: All the system's **variables** (including error) must be **bounded**.
- Ideally **error variables** should **converge** to 0
- **Stability** is the **main control objective**
 - Without stability there is no actual control...

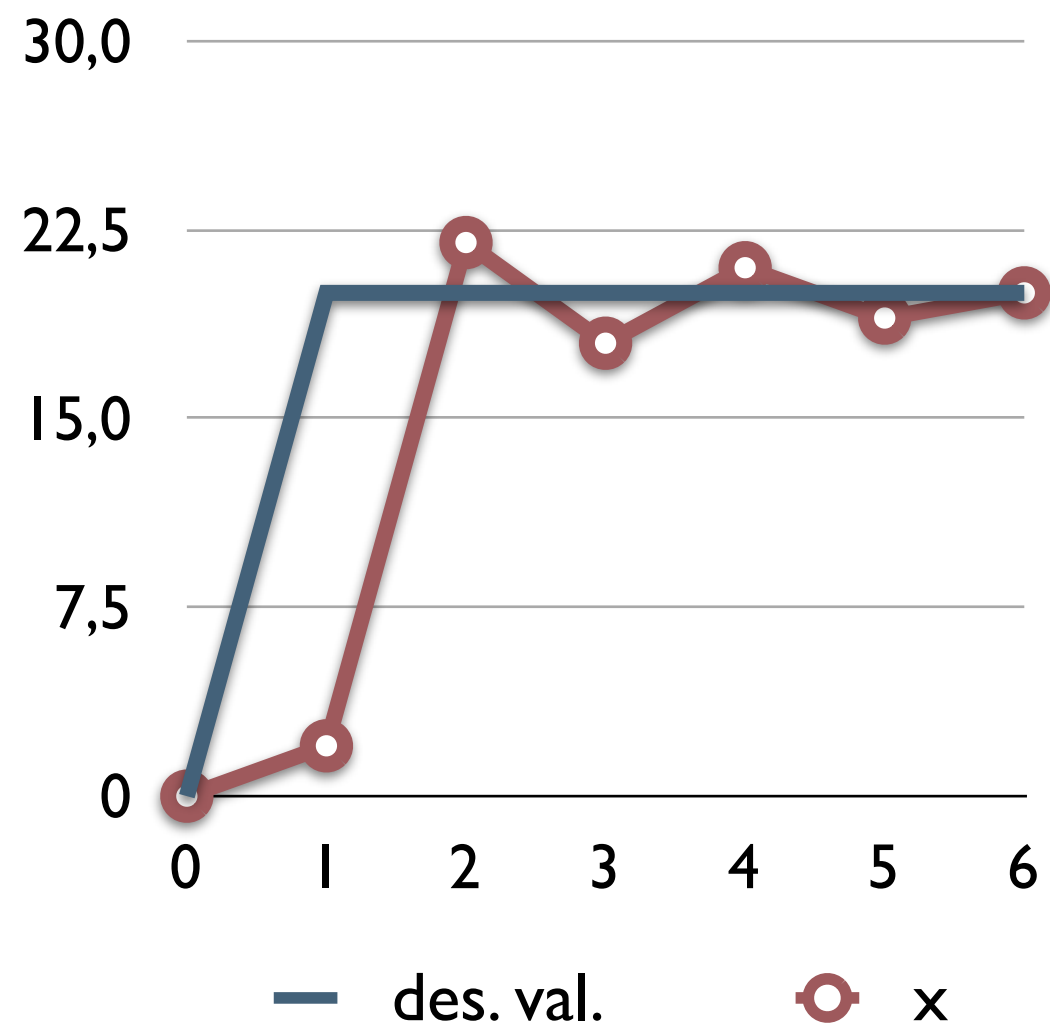


Control objective

Not stable



Stable

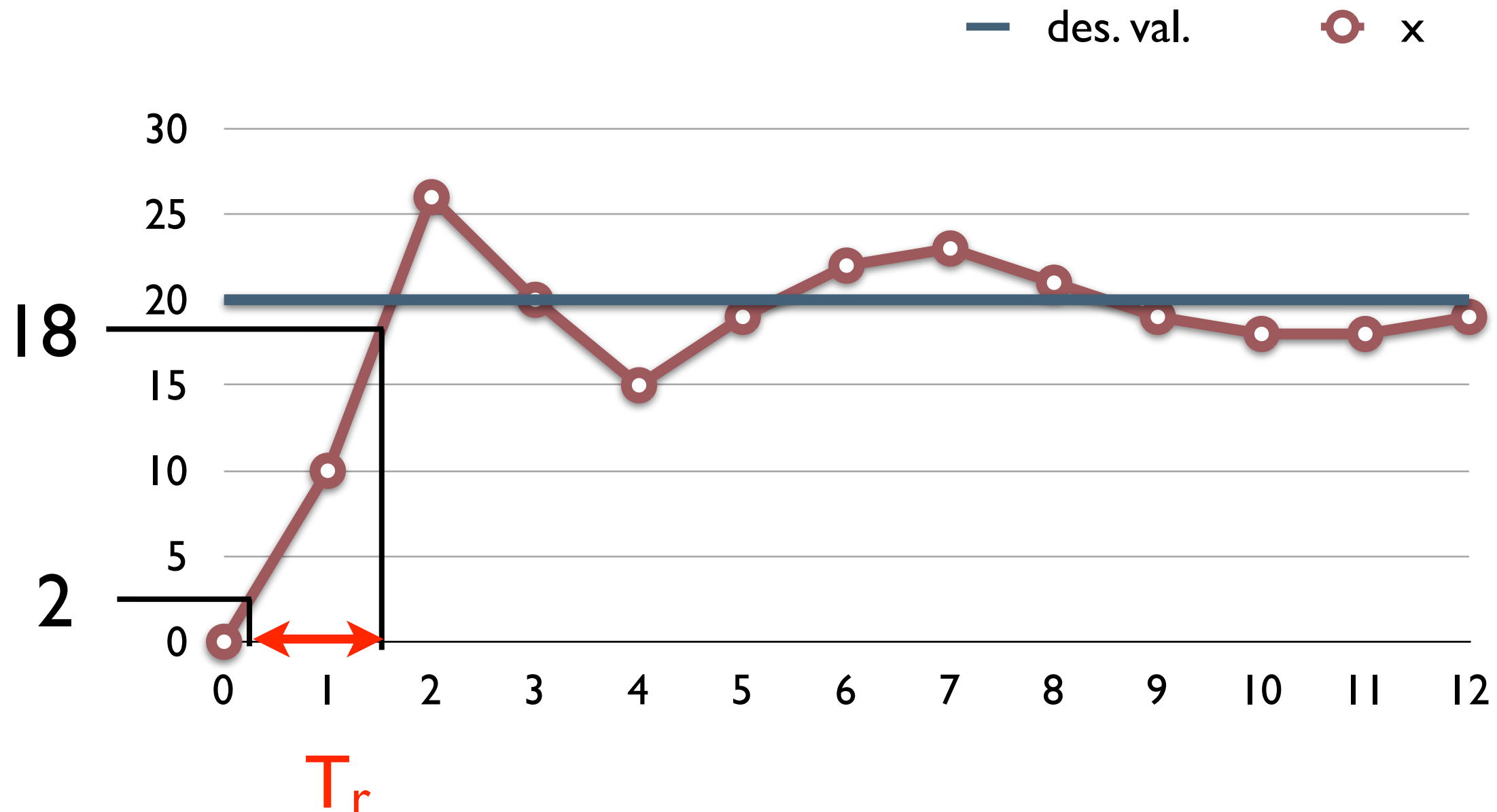


Control objectives

- **Performance:** In a **stable system**, performances measures, thanks to **several parameters**, the **distance between the output and the desired value**
- **Raising time T_r** . Let **d** be the distance between the initial value **i** and the desired value **r**, then **T_r** is the time need to see the output change from **$i + 10\% \times d$** to **$i + 90\% \times d$**



Control objectives

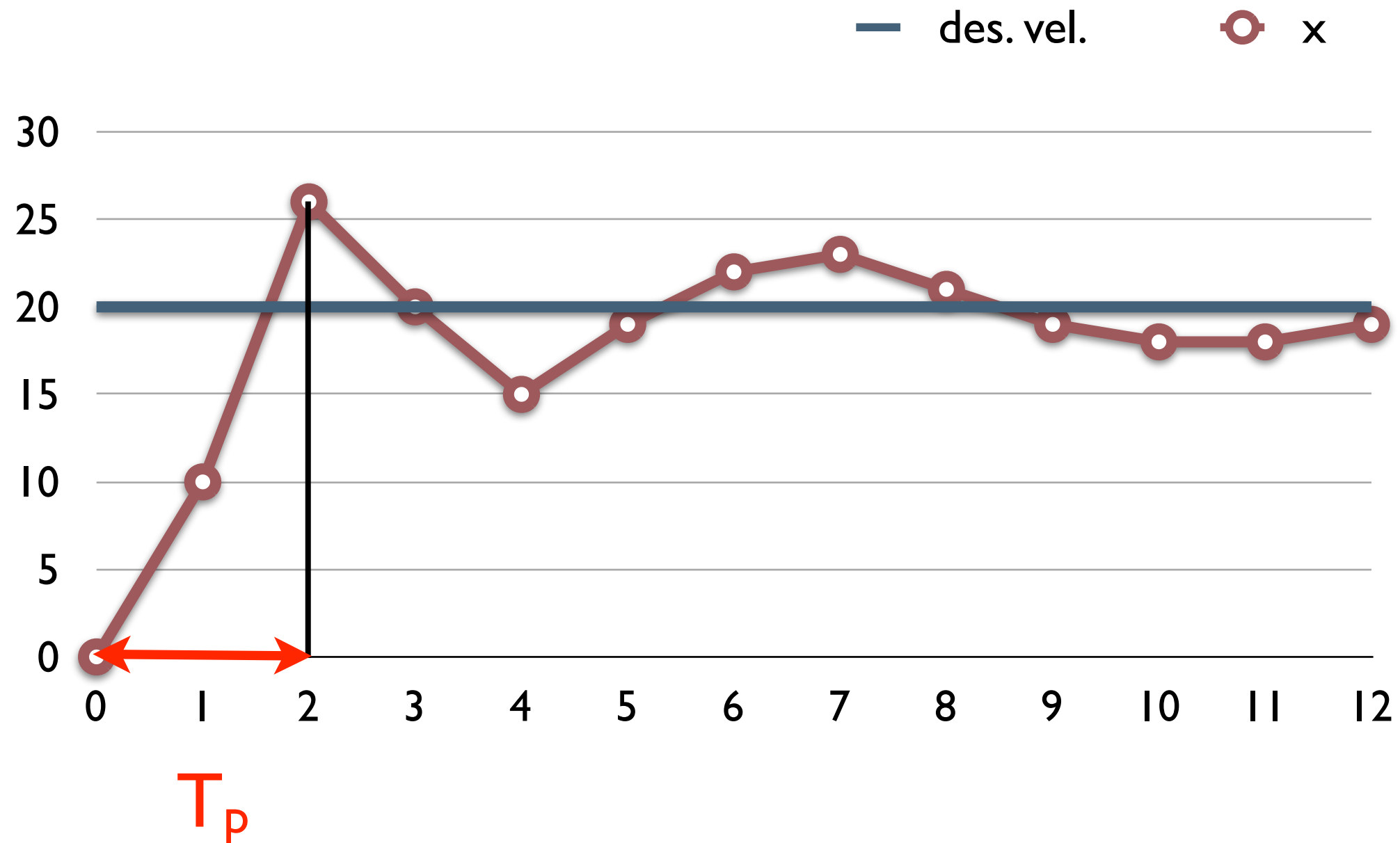


Control objectives

- **Performance:**
- **Peak time T_p :** Time to reach the **first peak** in the output



Control objectives

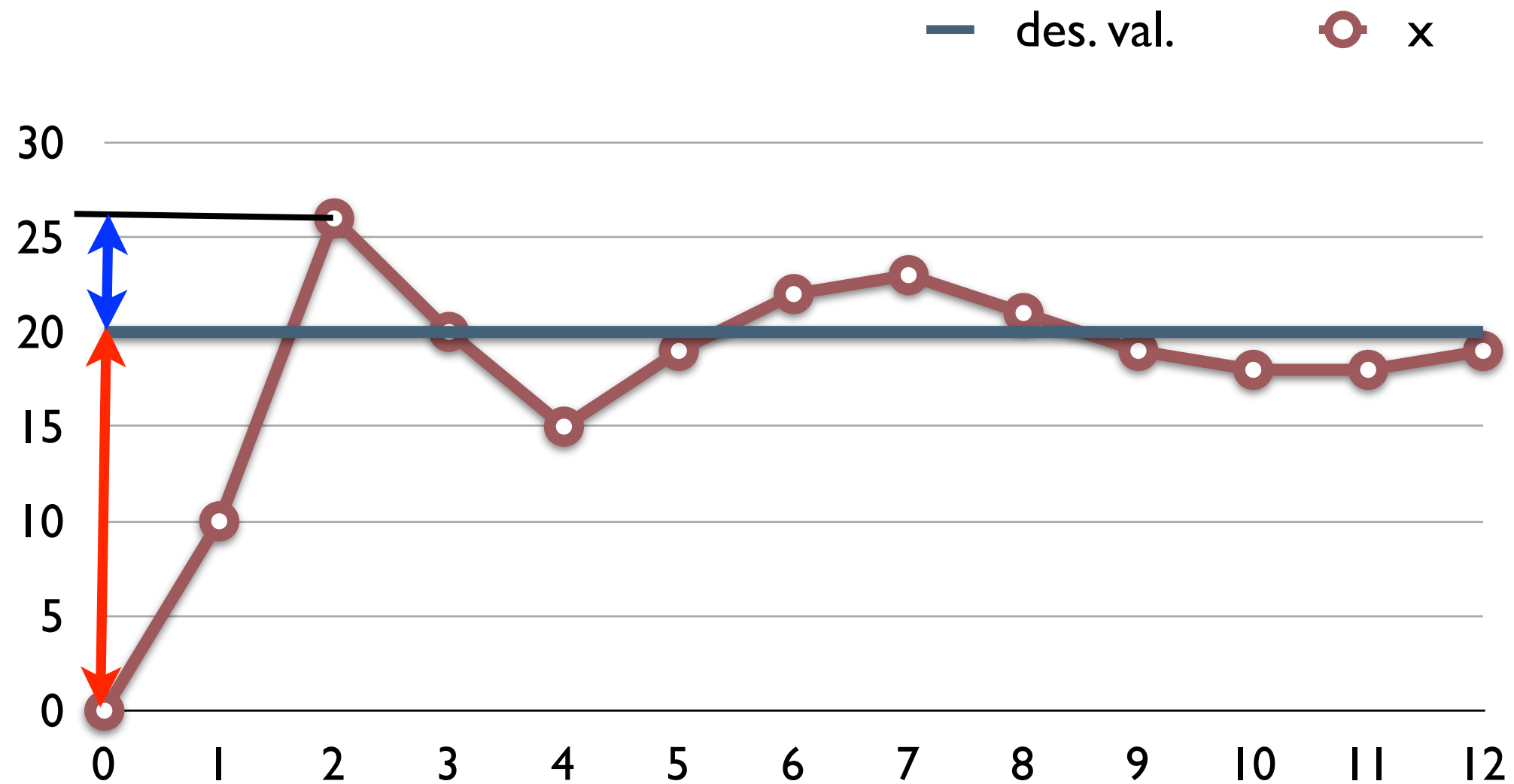


Control objectives

- **Performance:**
- **Overshoot M_p :** Percentage amount by which the peak **exceeds** the desired value



Control objectives



$$M_p = \frac{\text{blue arrow}}{\text{red arrow}} \text{ (in \%)}$$

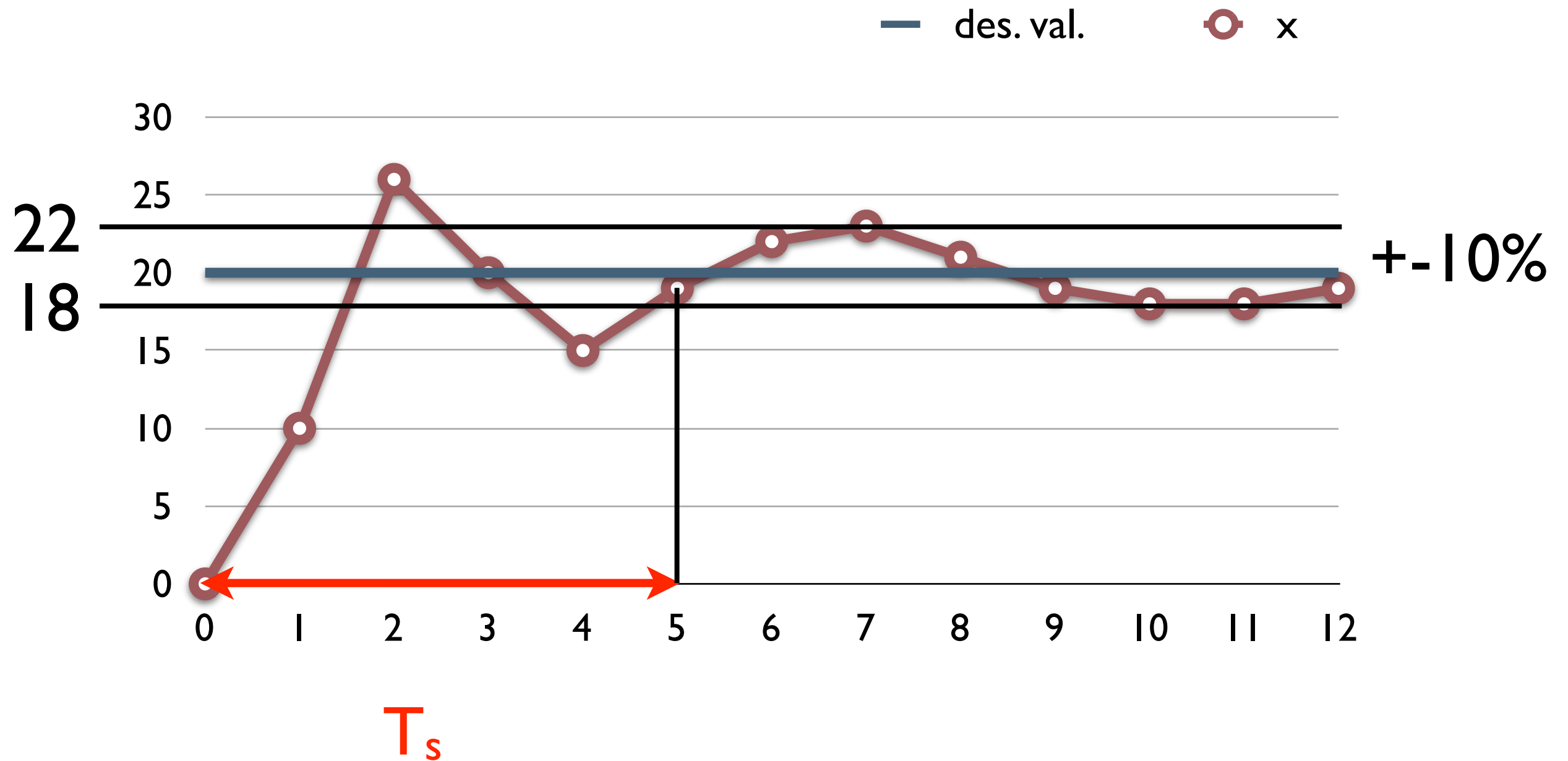


Control objectives

- **Performance:**
- **Settling time T_s :** Time required for the system to **settle down** within an **interval of $x\%$** centered on the **desired value** ($x=1, 10, \dots$)



Control objectives



Control objectives

- **Disturbance rejection**: the system should not be sensitive to disturbances
- **Robustness**: Remind that we are reasoning on a **model** of the environment. **Slight changes** in the behaviour of the environment should **not affect significantly the controller.**



Types of controllers

- From now on we only consider **closed loop controllers**
- The controller we have seen so far is a **proportional controller (P)**
- Its control law is $P \times e_t$, where e_t is the error $r_t - v_t$ at time t
- Its output is **proportional** to the error



P controllers

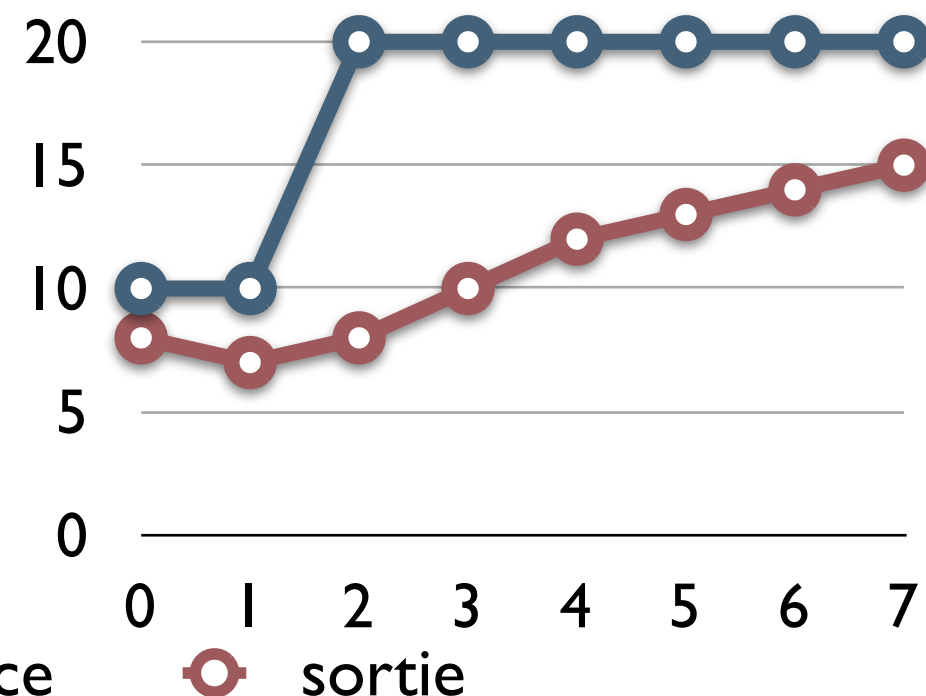
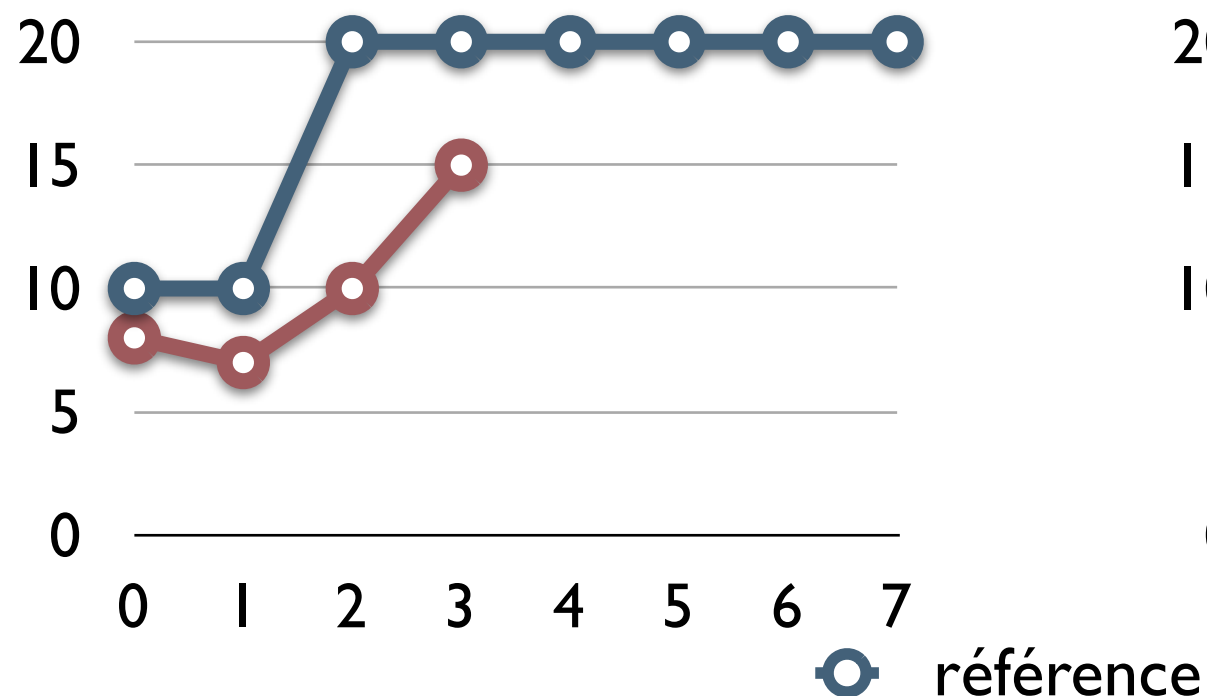
$$P \times e_t$$

- A **P controller**, has access to the **previous error only**
- In many cases, control would be much easier to achieve if one could “**predict future errors**”



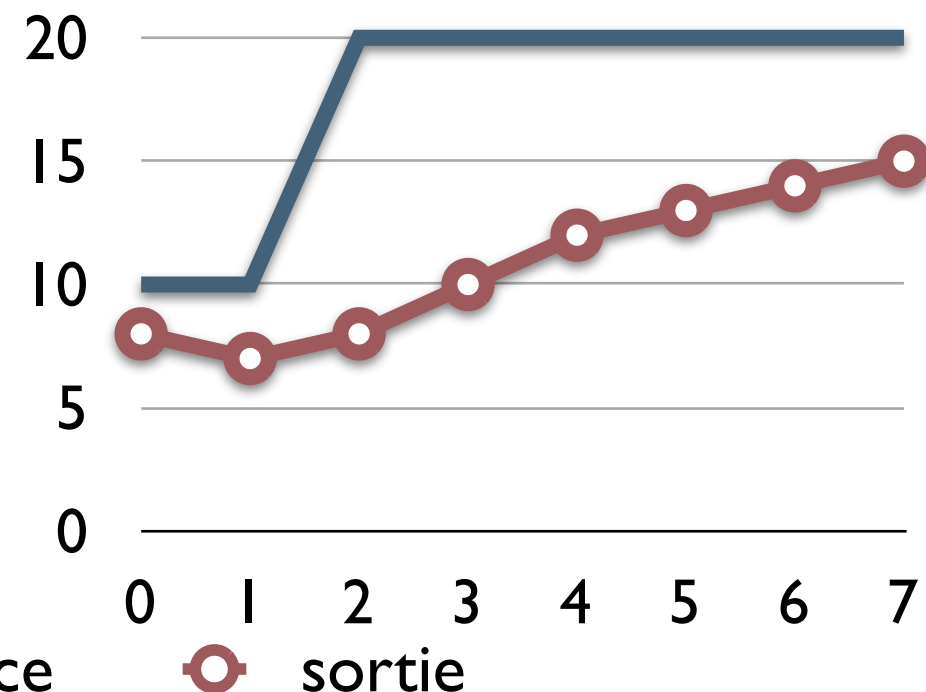
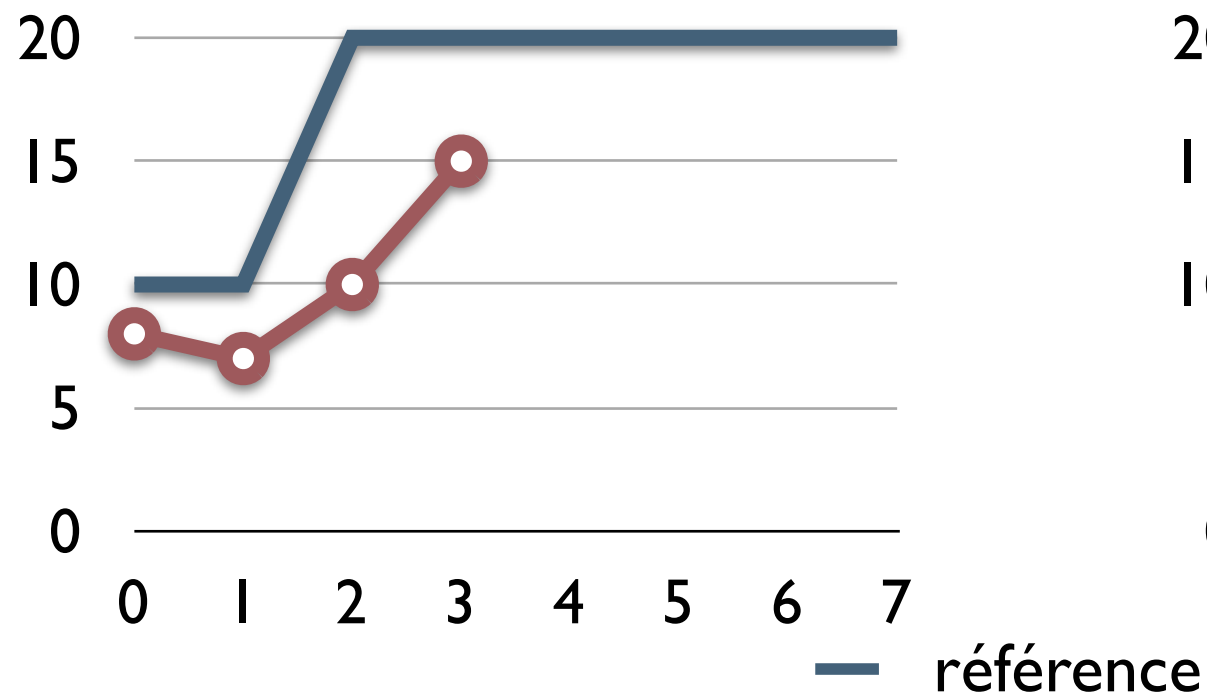
P controller

- At $t=3$, left and at $t=7$, right the errors are equal
- A P controller thus has the same reaction
- However, the ideal reaction is different



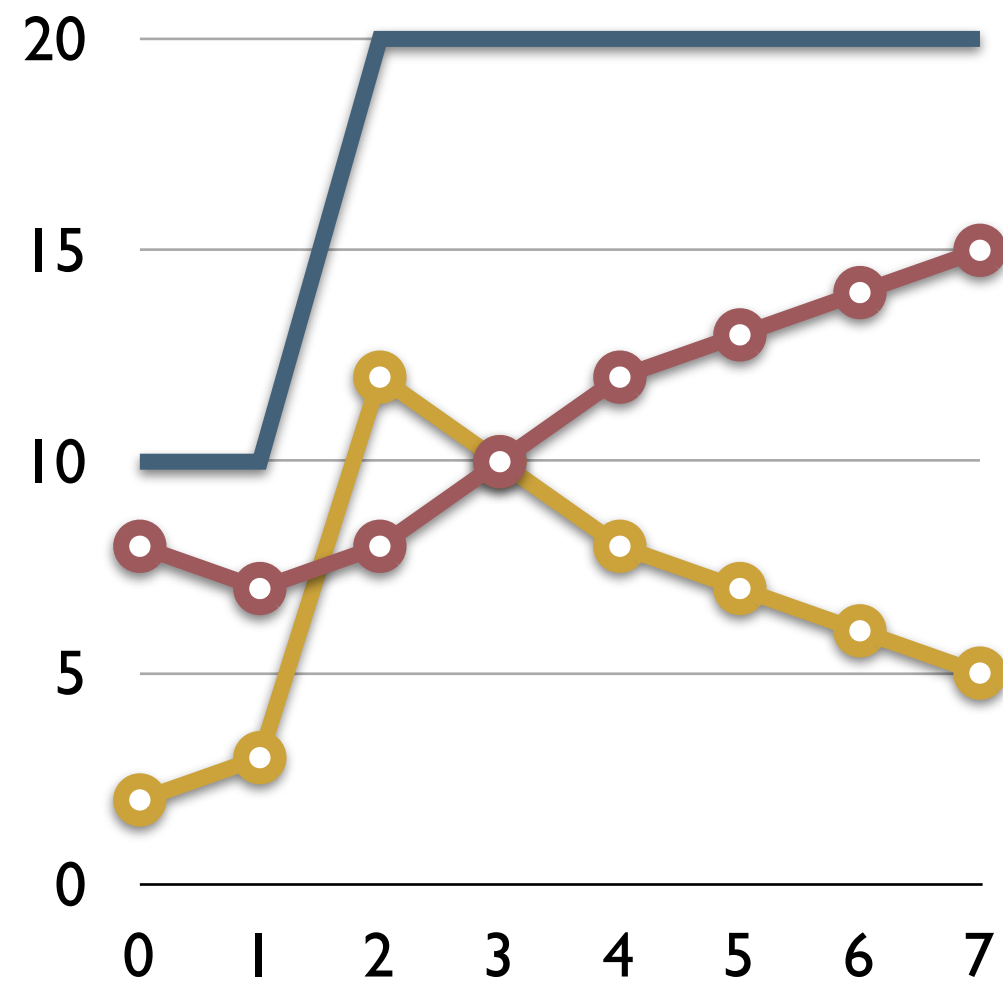
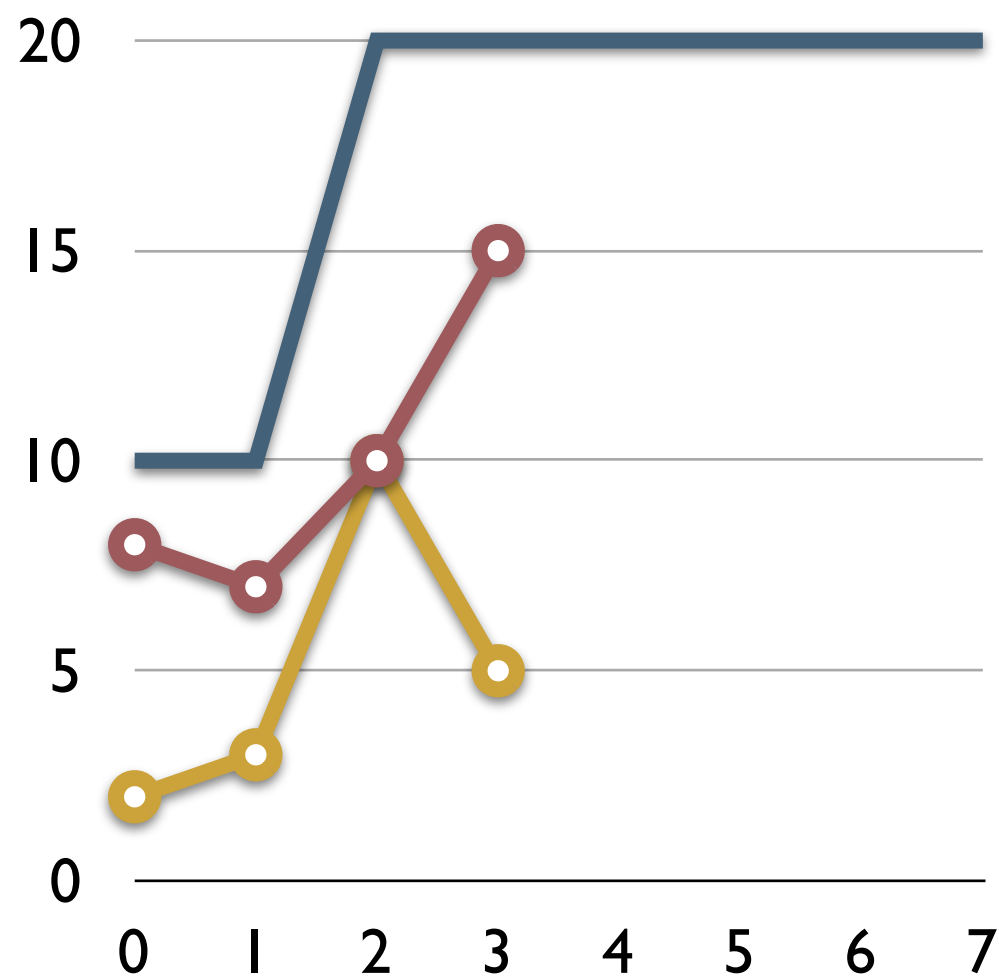
P controller

- **To the left:** one should **slow down** the grow rate to avoid overshooting the reference
- **To the right:** that rate should be **accelerated**



P controller

How the error behaves:



Quick
decrease

— ref. ○ output ○ error

Slow decrease



PD Controllers

- By taking into account the first derivative of the error, we can have a more precise controller
 - We'll use the derivative to predict how the error behaves
- Such controllers are called Proportional and Derivative (PD)
- Control law is of the form:
$$u_t = P \times e_t + D \times (e_t - e_{t-1})$$
- When $D=0$, we have a P controller



Example

- Let's go back to our example, considering r_t is constant and w_t is null.

$$v_{t+1} = 0,7v_t + 0,5u_t - w_t$$

$$u_t = P \times e_t + D \times (e_t - e_{t-1})$$

$$e_t = r - v_t$$

We get:

$$v_{t+1} = (0,7 - 0,5 \times (P + D)) \times v_t + 0,5 \times D \times v_{t-1} + 0,5 \times P \times r$$



Example

- Let's consider again the steady states: $v_{t+1} = v_t = v_{t-1} = v_{ss}$

$$v_{t+1} = (0,7 - 0,5 \times (P + D)) \times v_t + 0,5 \times D \times v_{t-1} + 0,5 \times P \times r$$

$$v_{t+1} = v_t = v_{t-1} = v_{ss}$$

We get:

$$v_{ss} = (0,5 \times P / (0,3 + 0,5 \times P)) \times r$$



Example

- Let's consider again the steady states: $v_{t+1} = v_t = v_{t-1} = v_{ss}$

$$v_{t+1} = (0,7 - 0,5 \times (P + D)) \times v_t + 0,5 \times D \times v_{t-1} + 0,5 \times P \times r$$

$$v_{t+1} = v_t = v_{t-1} = v_{ss}$$

We get:

$$v_{ss} = (0,5 \times P / (0,3 + 0,5 \times P)) \times r$$

It is the same result as with the P controller



Example

- The equation describing **steady states** is the same **as for the P controller**
- This is not (should not be ?) **surprising**. In the **steady states**, the output does not vary, **neither the error !**
- Adding a **derivative term** does **not** affect steady states
- But it allows **more flexibility**



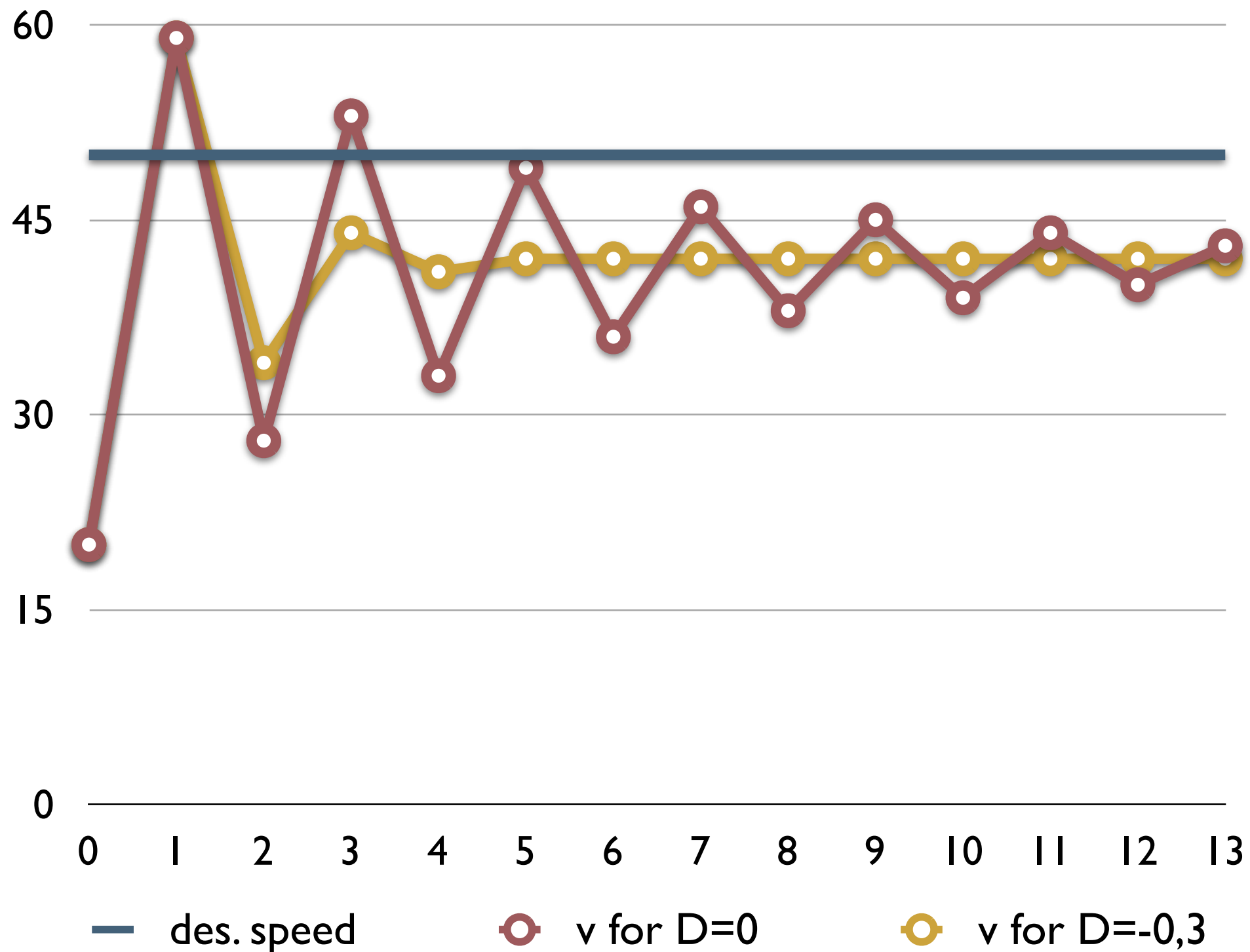
Example

- **Remember:** With P controllers, when **P** was high, we had high accuracy but many oscillations
- We will adapt the value of **D** in the PD controller to avoid oscillations
- D will clearly have a negative value: if the error grows fast (first derivative highly positive) we have to reduce the input to the environment.



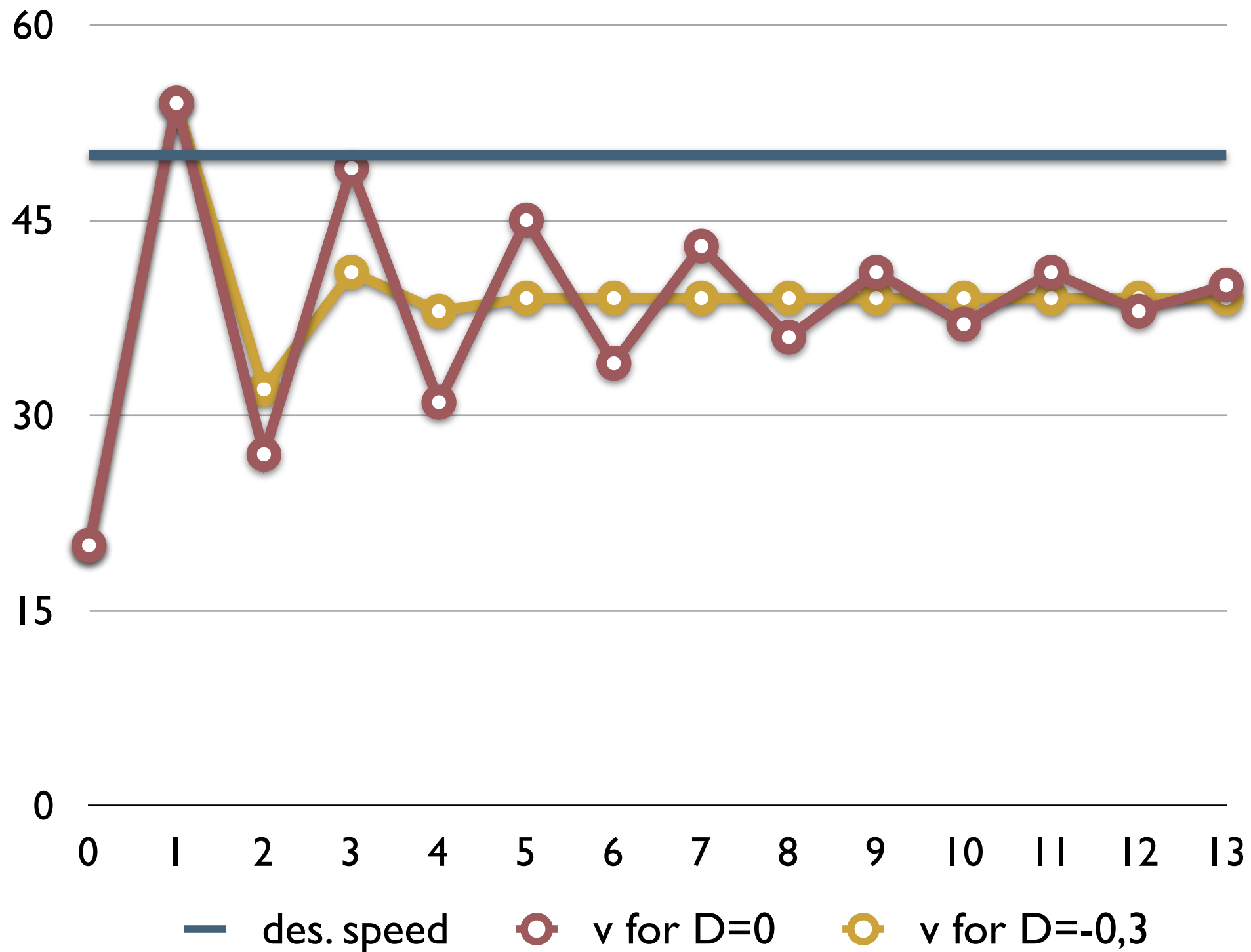
Examples

$w = 0$ $r = 50$ $P = 3$ $u_t = P \times e_t + D \times (e_t - e_{t-1})$



Examples

$w = 5$ $r = 50$ $P = 3$ $u_t = P \times e_t + D \times (e_t - e_{t-1})$



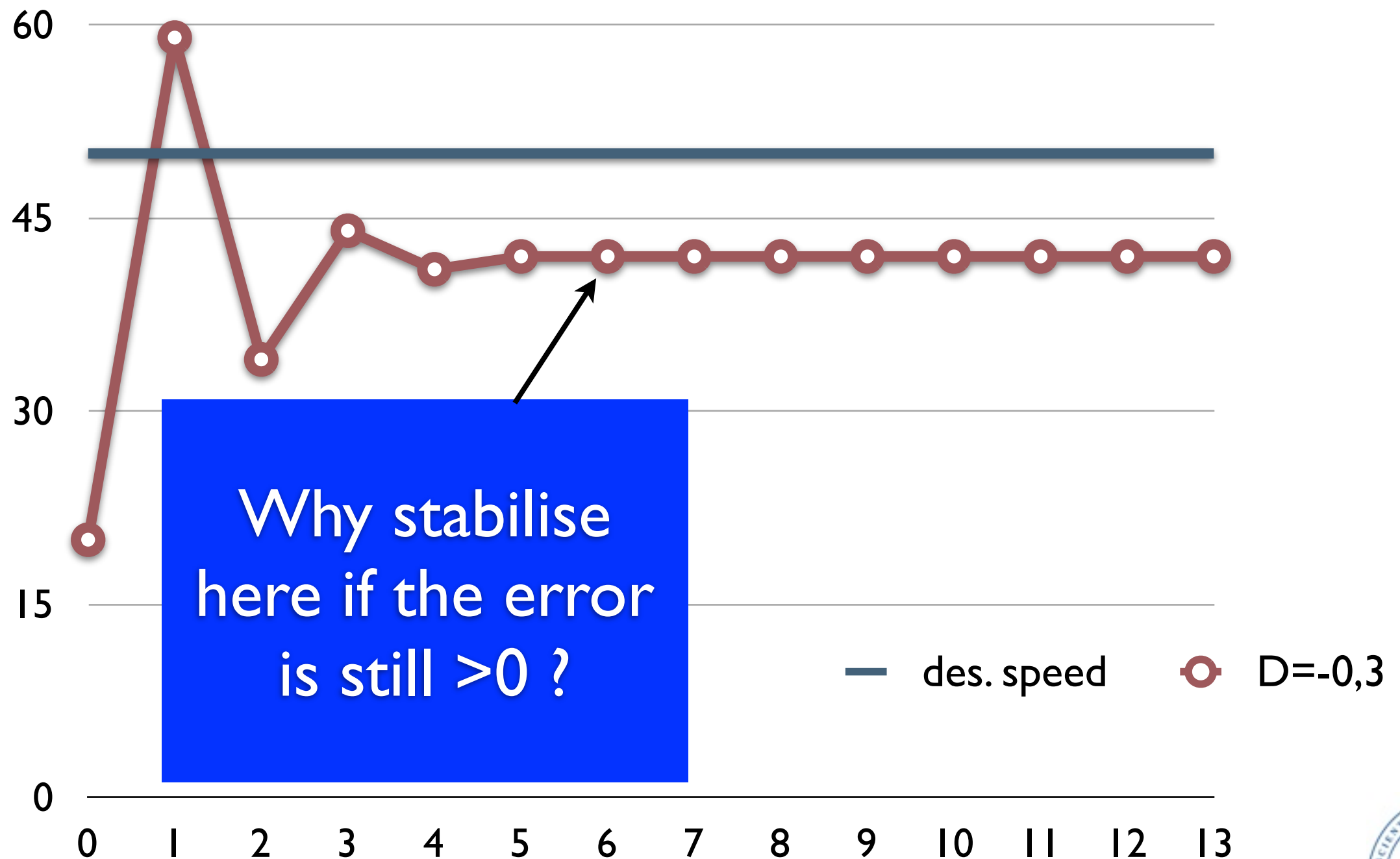
IP controller

- Adding the differential term has allowed to reduce oscillations
- Unfortunately, accuracy is still poor



Poor accuracy

$w = 0$ $r = 50$ $P = 3$



Poor accuracy

- We have seen that in steady states, the DP controller behaves like a P controller (for the same value of P)
- There is thus no reason that a PD controller converges with more accuracy than a P controller
- To obtain higher accuracy, we, should take a larger P, but then the system diverges...

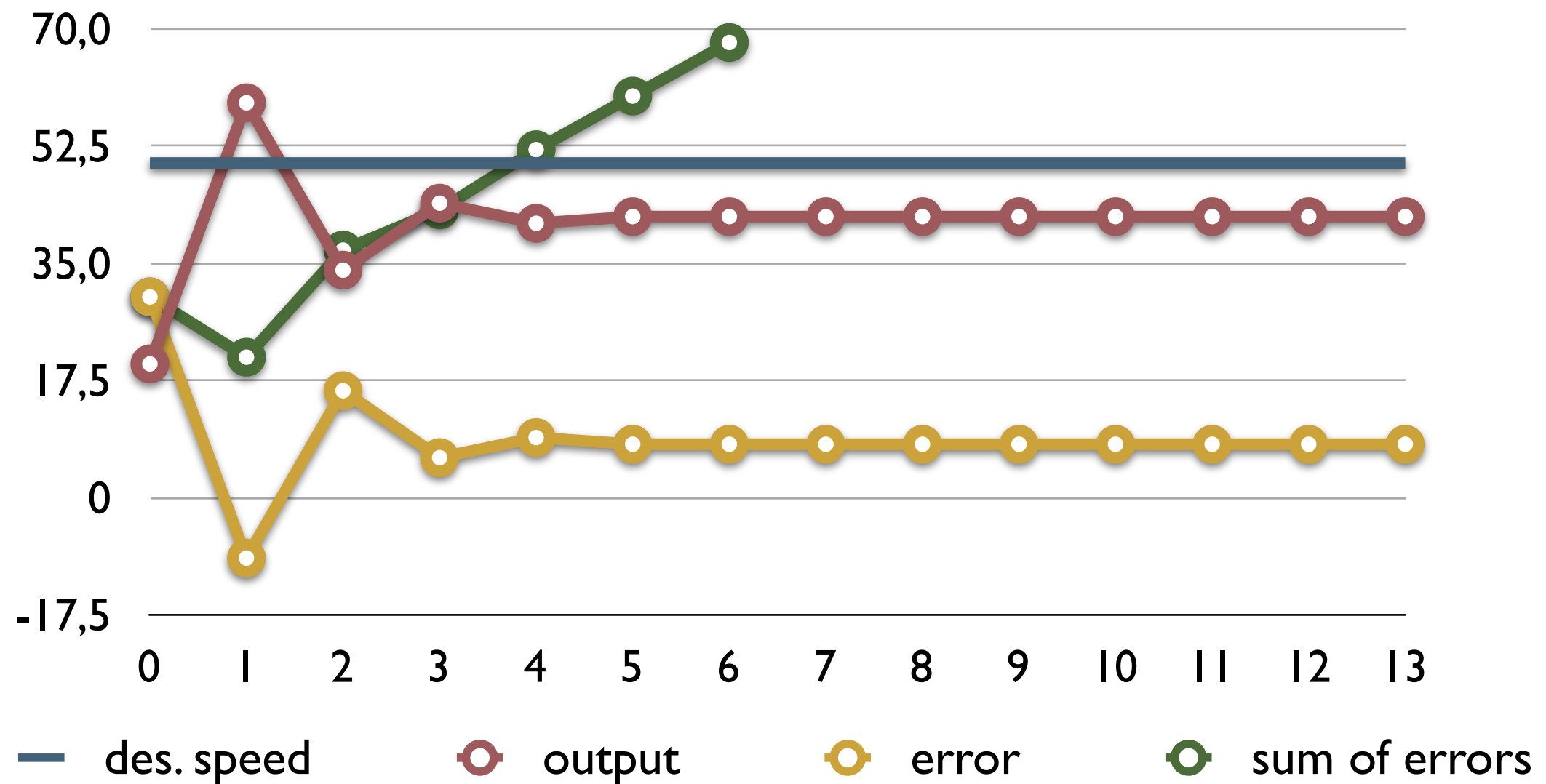


IP controller

- A **solution** consists in taking into account not only the punctual error, but also the **sum of all errors** since the initial point.
- This is an **Integral - Proportional (IP) controller**



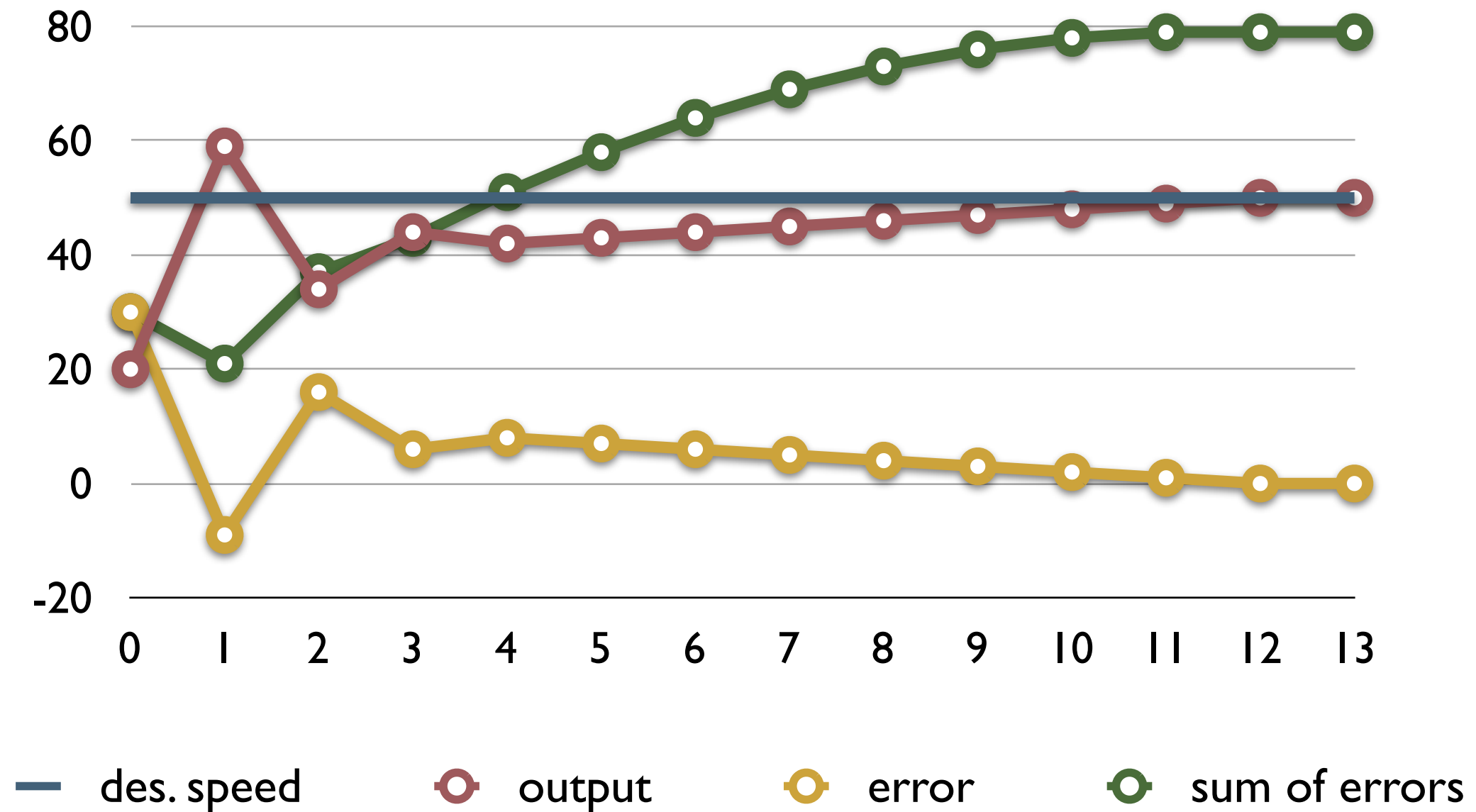
Convergence



In this example **speed** and **error converge**
but the **sum of errors** does not !



Convergence



If we manage to have the output **converge** to the reference, the **sum of errors** will converge too.



IP Controller

- The **control law** for IP controllers is of the form:

$$u_t = P \times e_t + I \times \sum_{0 \leq i \leq t} e_i$$

- It contains **two parameters**: P and I
 - When $I=0$, we have a P controller
- The controller **lets the error converge to 0** to avoid **u_t diverging** (i.e. growing infinitely)



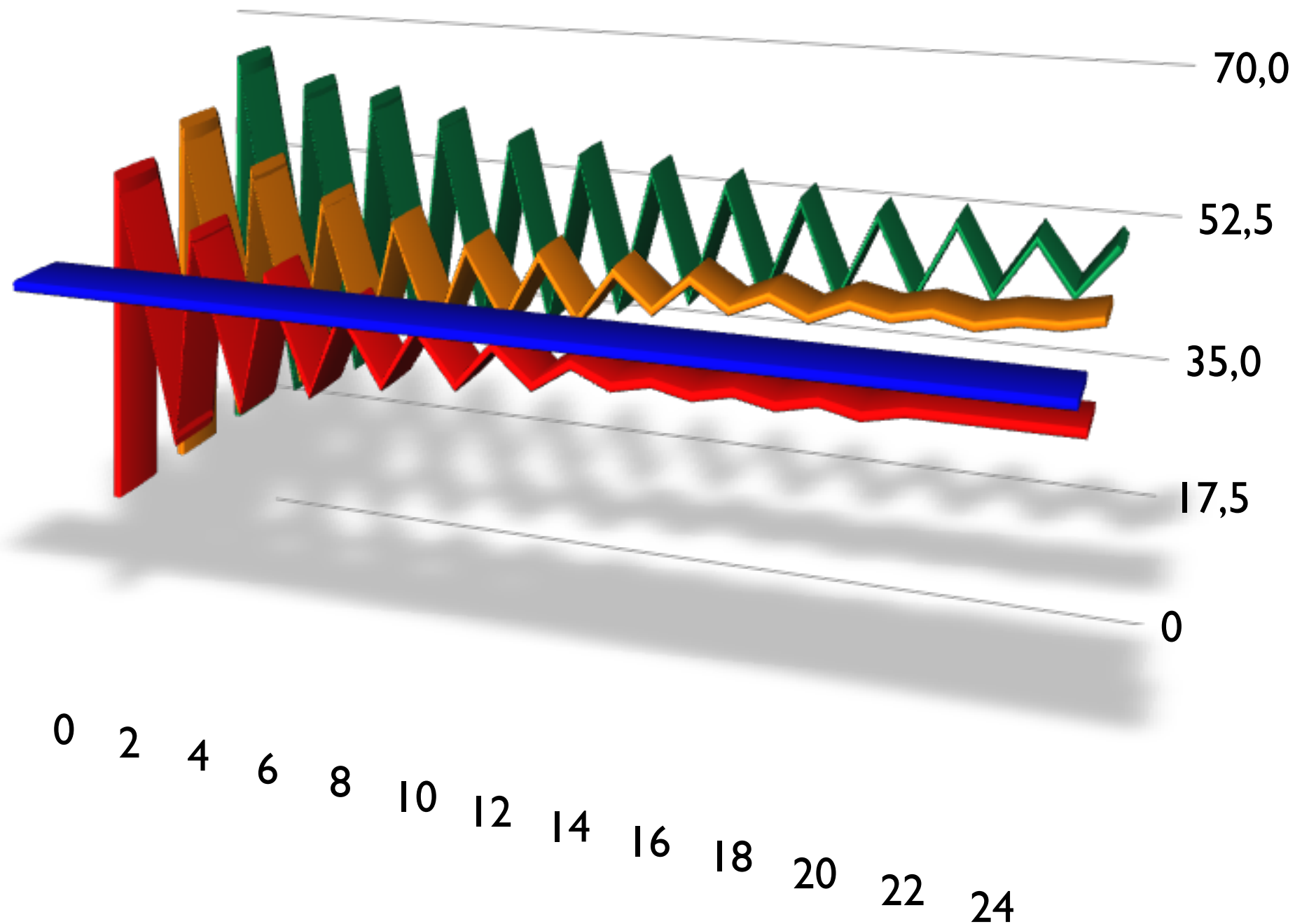
Example

- Again, the closed loop controller
- We use $P = 3$ since it gave good results with the P and PD controllers



Examples

$$w = 0 \quad r = 50 \quad P = 3 \quad u_t = P \times e_t + I \times \sum_{0 \leq i \leq t} e_i$$

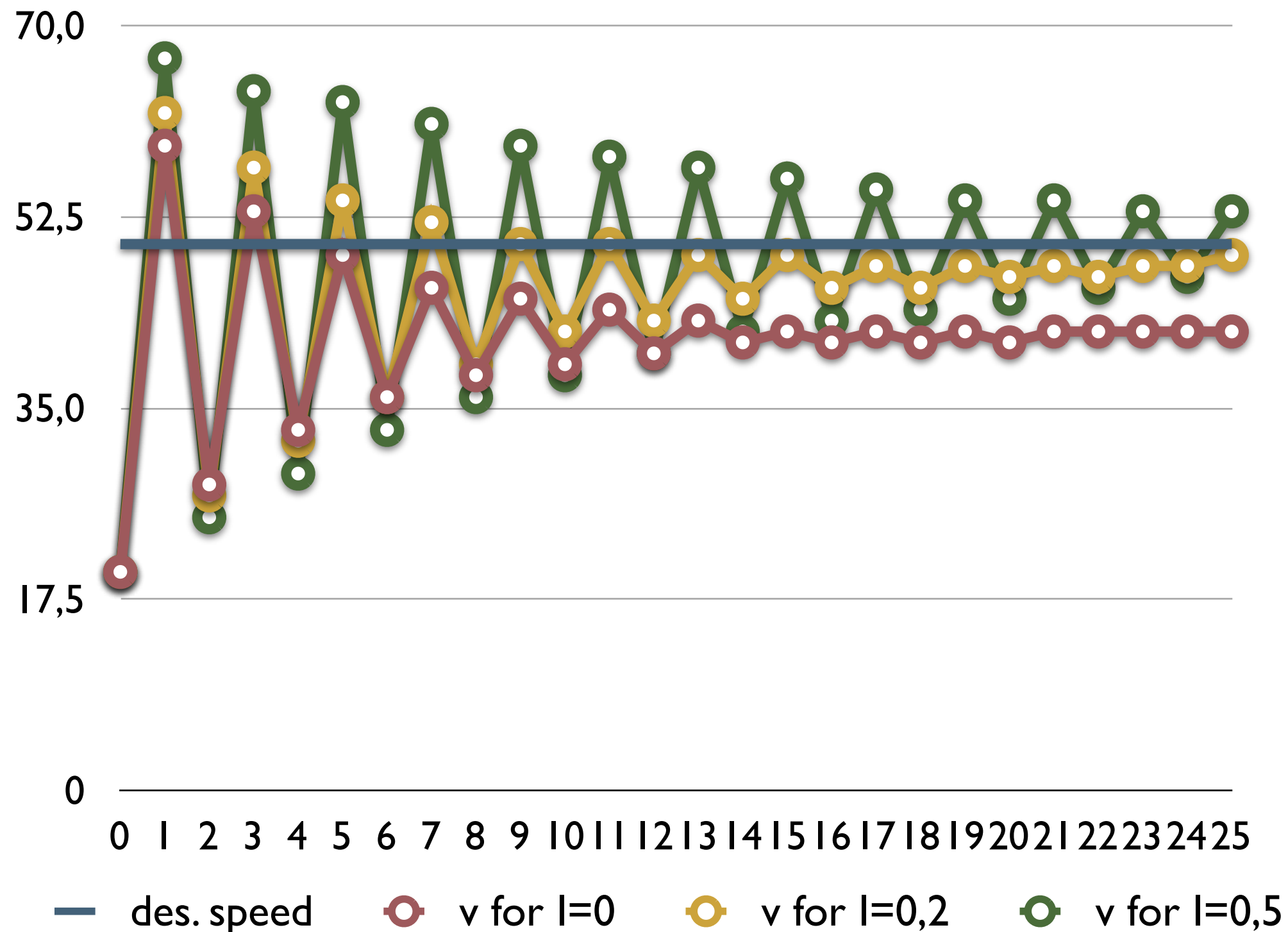


— des. speed — v for I=0 — v for I=0,2 — v for I=0,5



Examples

$w = 0$ $r = 50$ $P = 3$ $u_t = P \times e_t + I \times \sum_{0 \leq i \leq t} e_i$



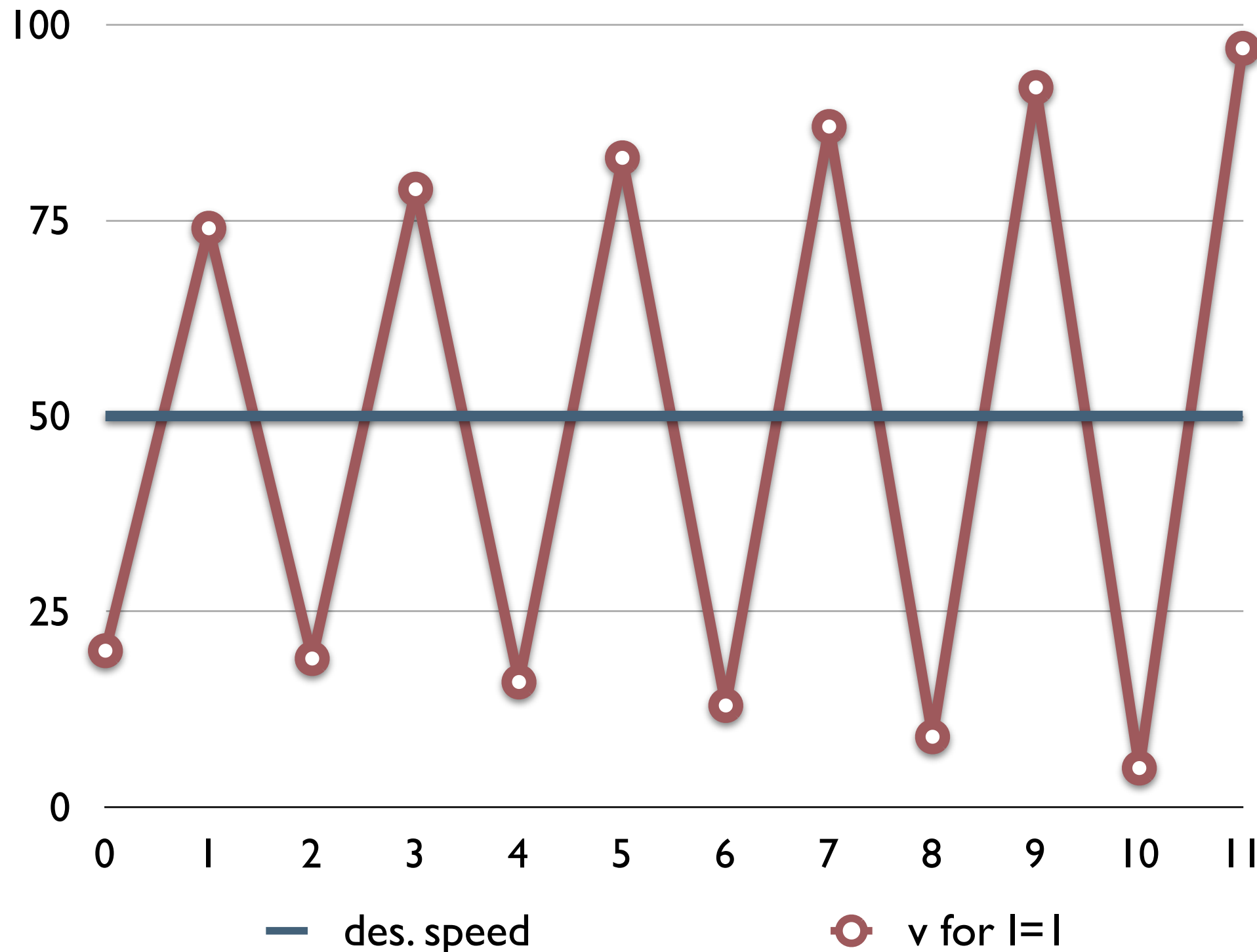
IP Controller

- With $I > 0$, we can see that the output converges to the reference value
- Unfortunately we have large oscillations
- Taking a bigger I , exacerbates the phenomenon
- For some values of I , the system diverges



Example - divergence

$w = 0$ $r = 50$ $P = 3$ $u_t = P \times e_t + I \times \sum_{0 \leq i \leq t} e_i$



DP vs. IP

- To sum up:
 - The differential term allows to avoid oscillations, but does not increase accuracy
 - The integral term lets the controller converge to the reference value but exacerbates the oscillations that were already present in the P controller



PID Controller

- We can combine DP and IP controllers to (hopefully) combine their advantages
- This is a Proportional - Integral - Derivative (PID) controller



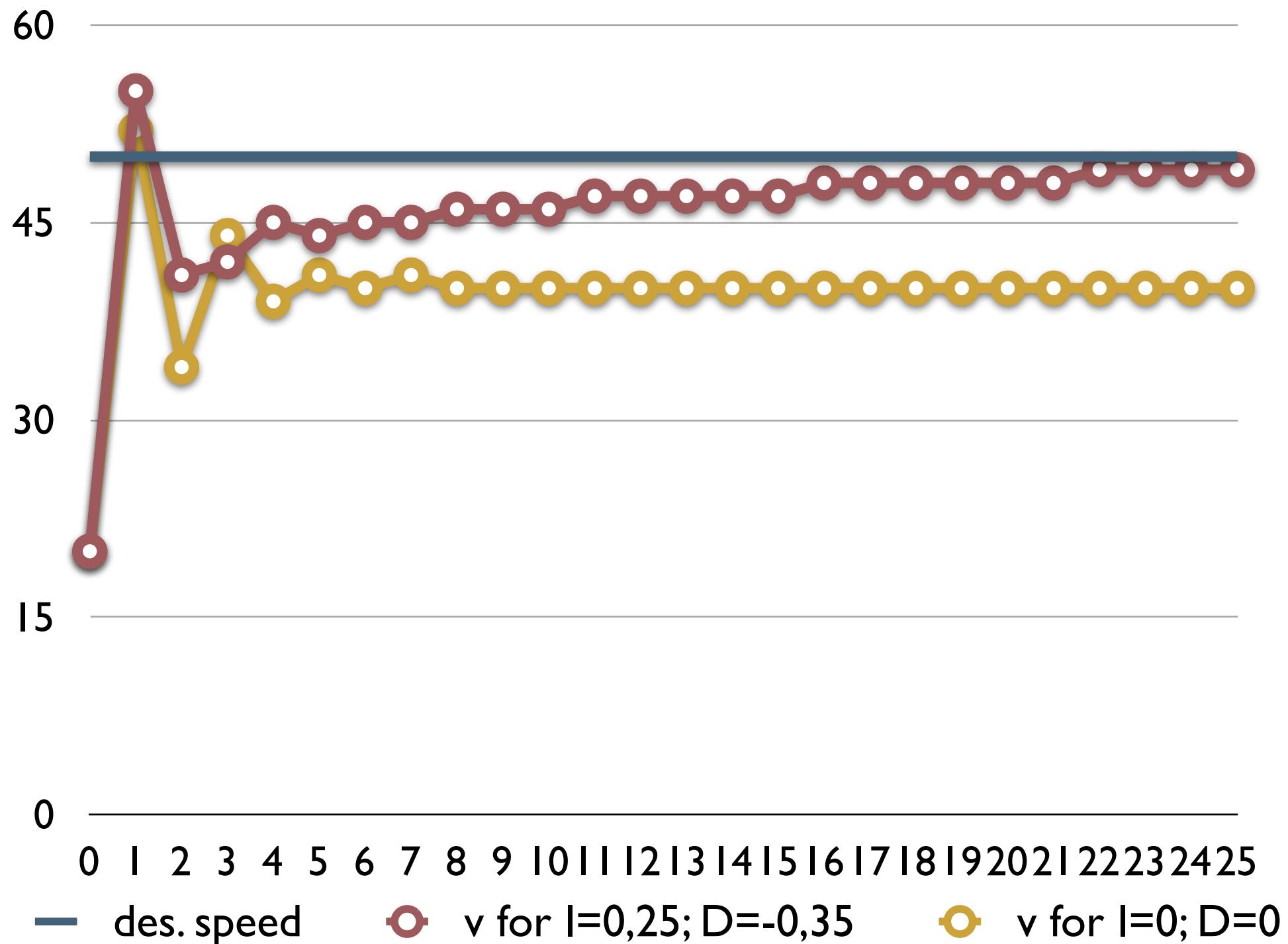
PID Controller

- The control law is:
$$u_t = P \times e_t + I \times \sum_{0 \leq i \leq t} e_i + D \times (e_t - e_{t-1})$$
- There are now 3 parameters, that one should tune finely
 - If $D = 0$ et $I \neq 0$, we get an IP controller
 - If $D \neq 0$ et $I = 0$, we get a DP controller
 - If $D = I = 0$, we get a P controller



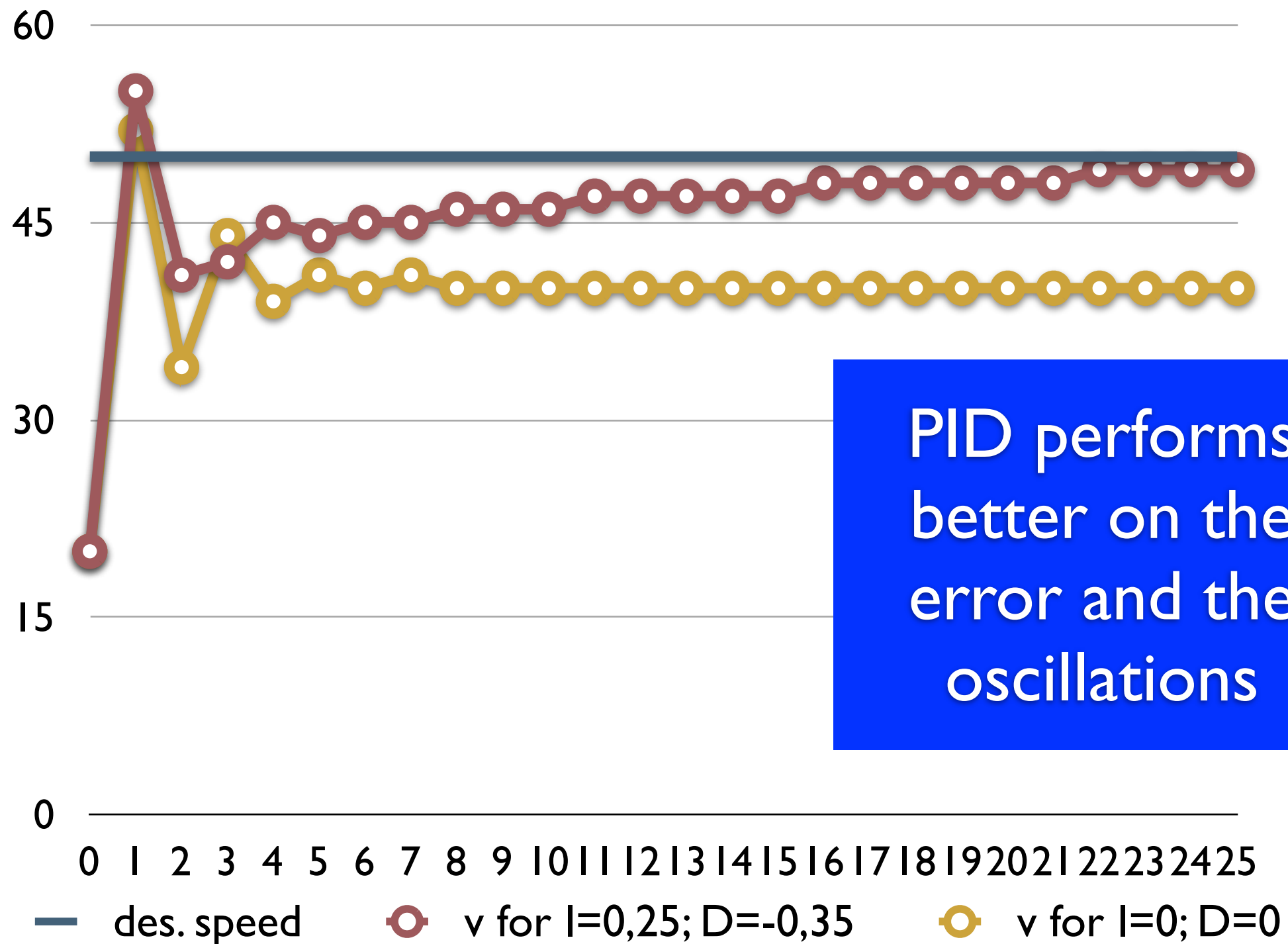
Example - PID vs P

$w = 0$ $r = 50$ $P = 2,5$ PID



Example - PID vs P

$w = 0$ $r = 50$ $P = 2,5$ PID

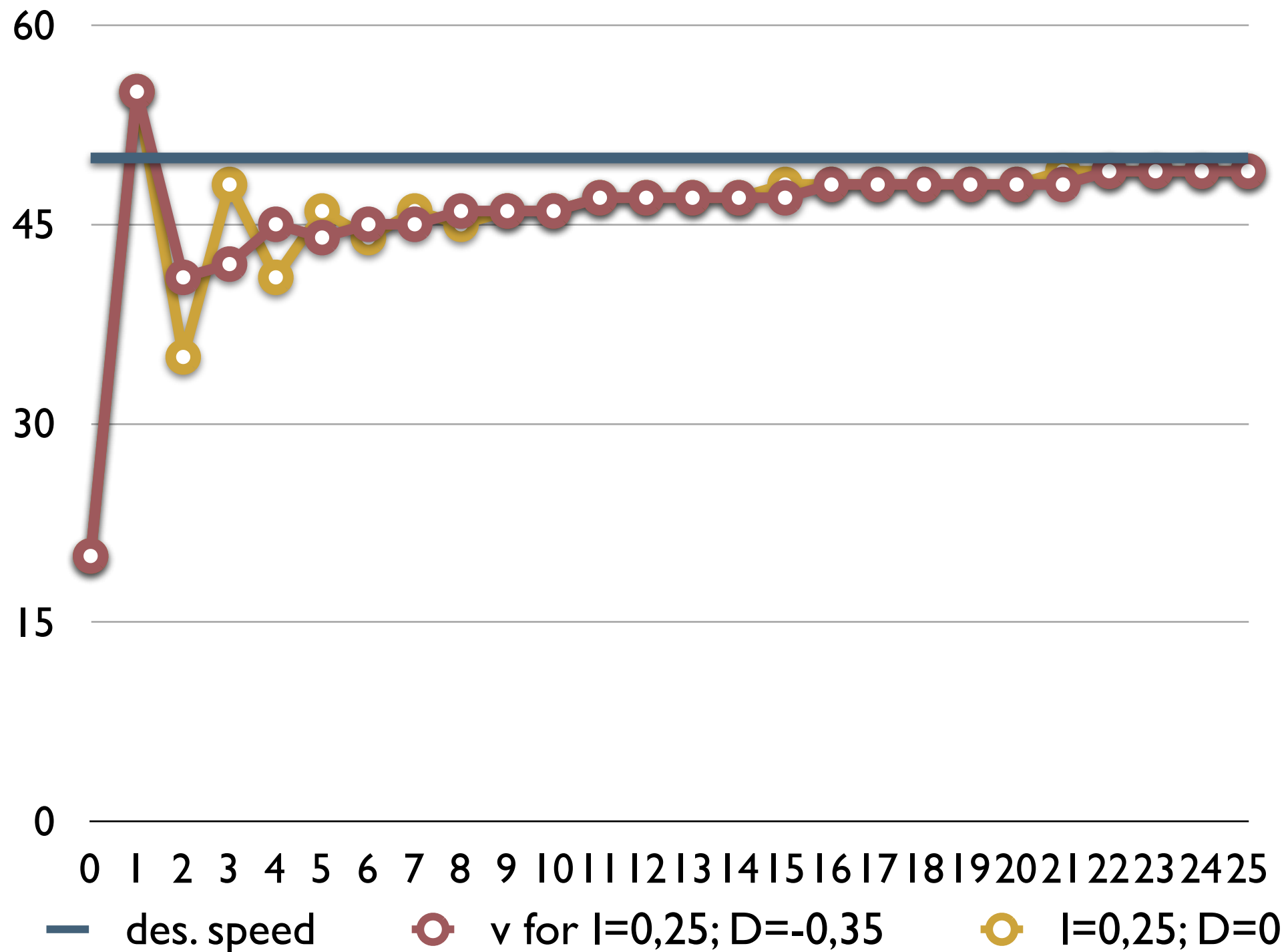


PID performs better on the error and the oscillations



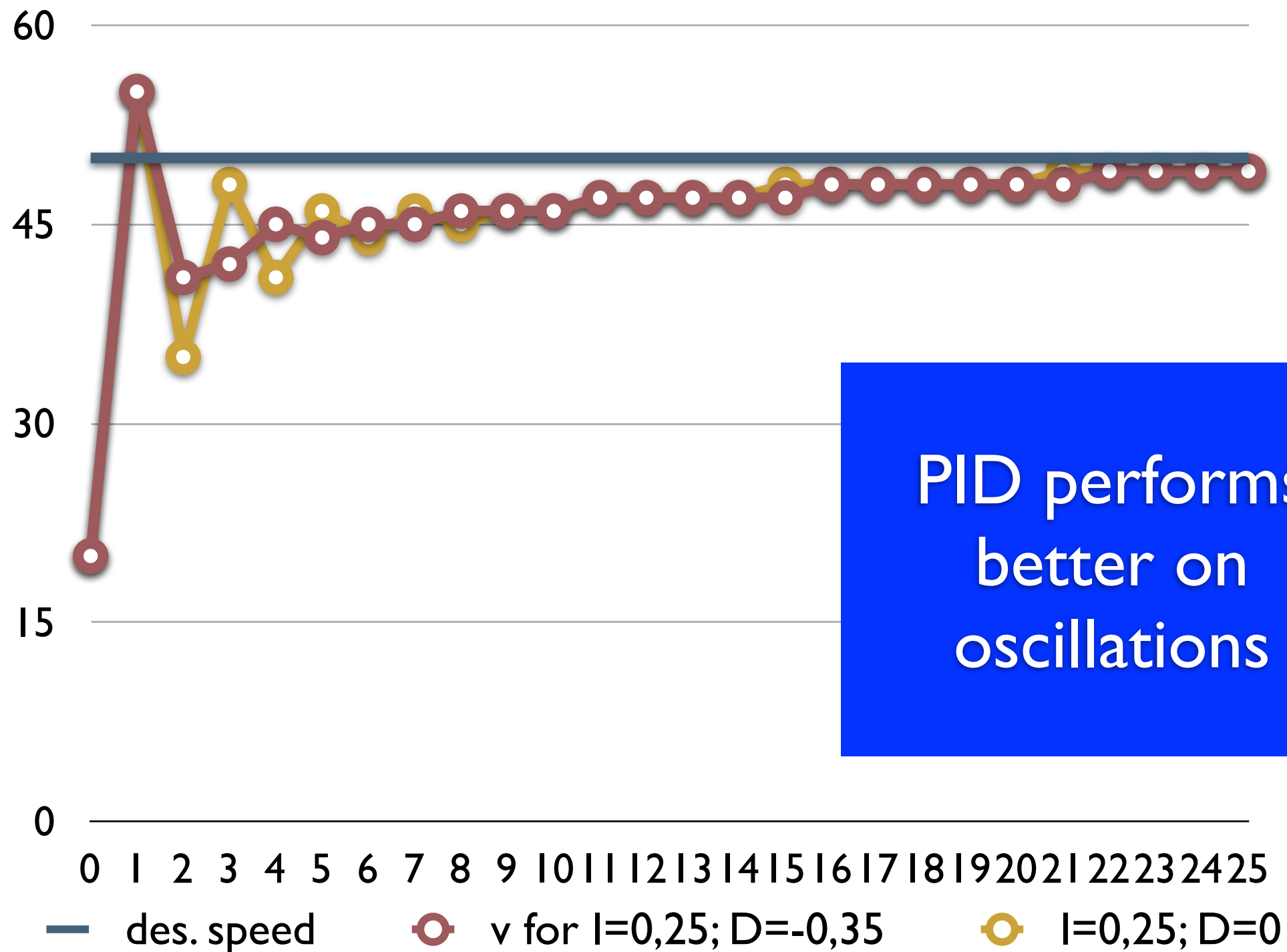
Examples PID vs IP

$w = 0$ $r = 50$ $P = 2,5$ PID



Examples PID vs IP

$w = 0$ $r = 50$ $P = 2,5$ PID

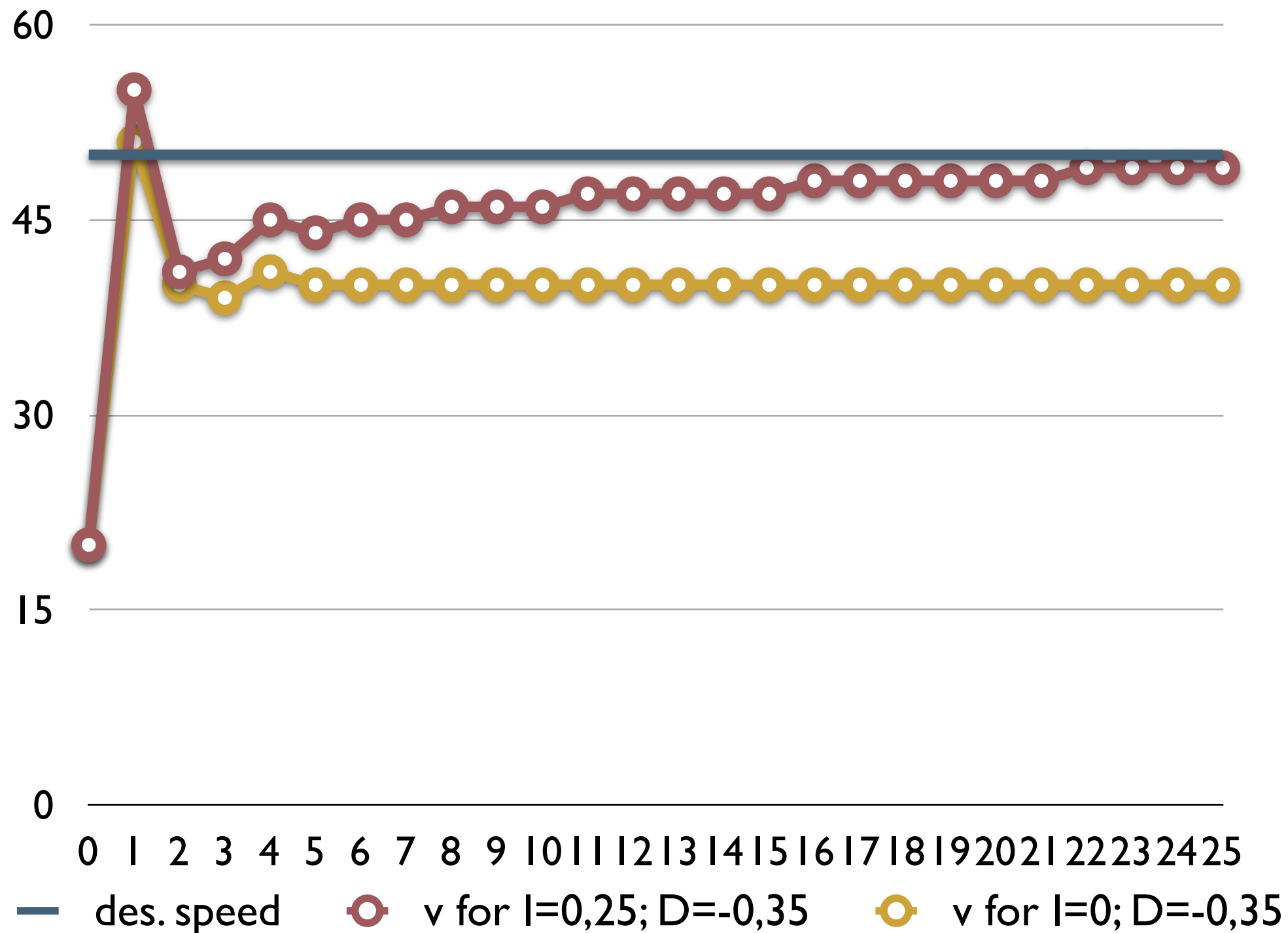


PID performs
better on
oscillations



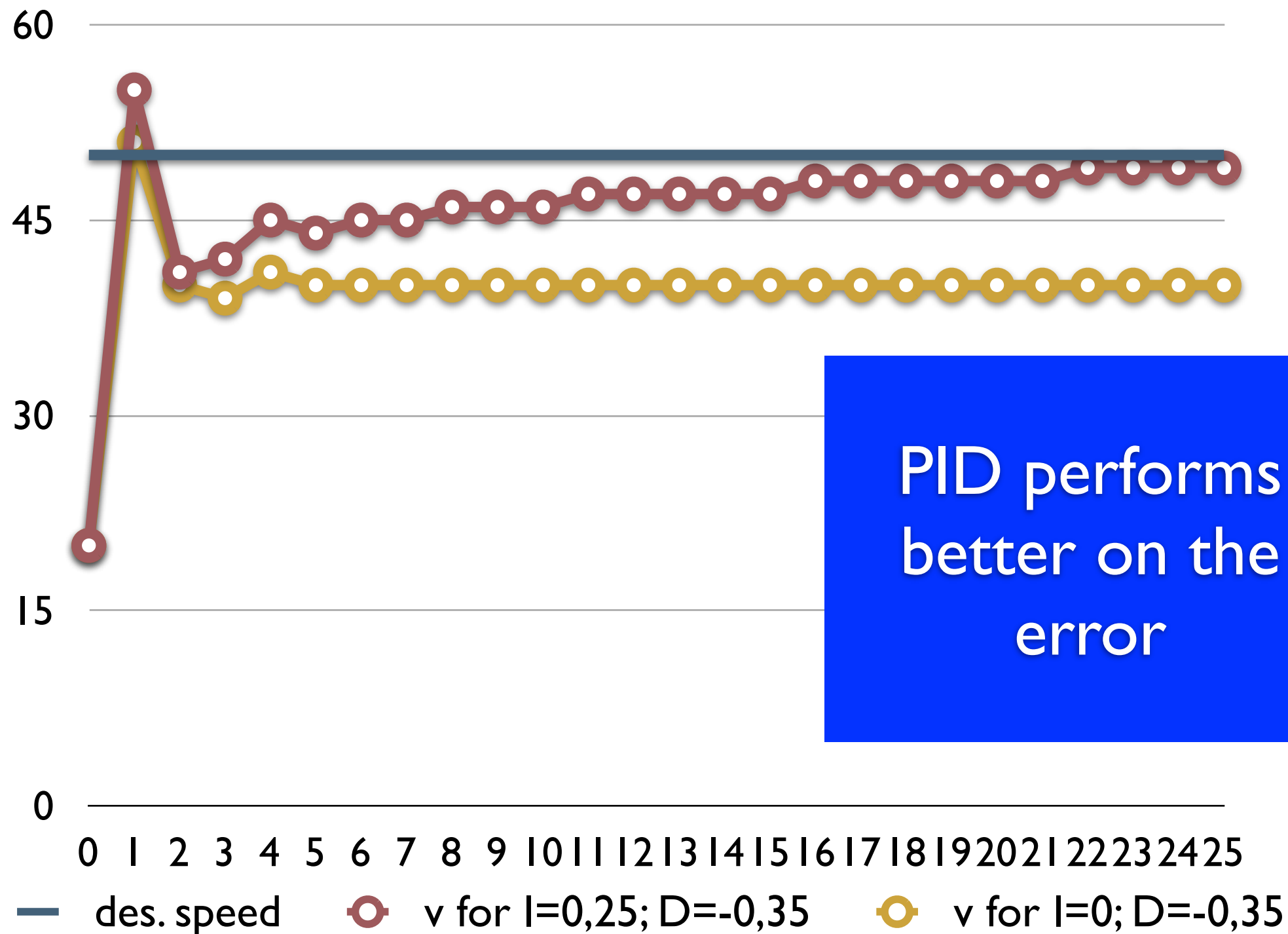
Example - PID vs DP

$w = 0$ $r = 50$ $P = 2,5$ PID



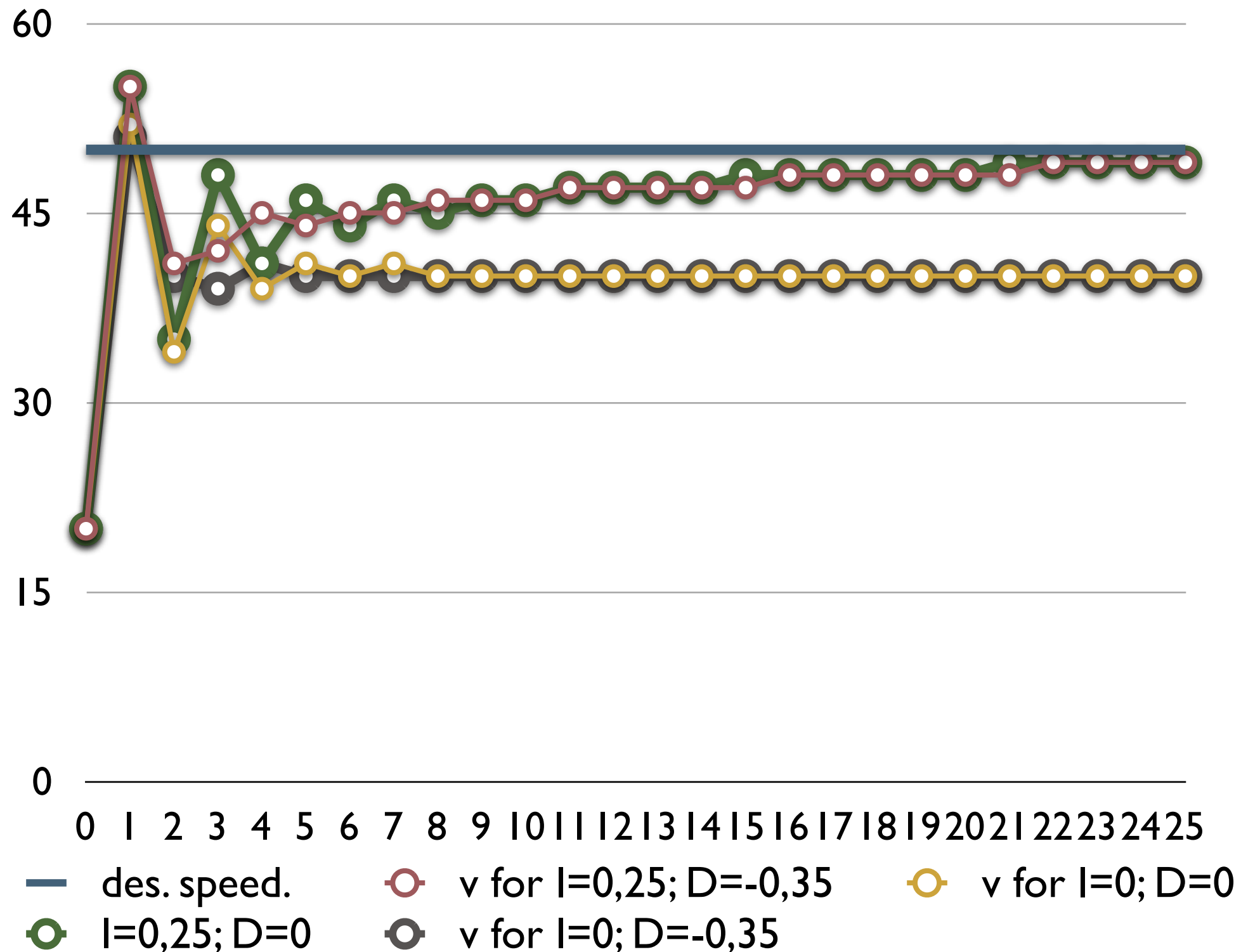
Example - PID vs DP

$w = 0$ $r = 50$ $P = 2,5$ PID



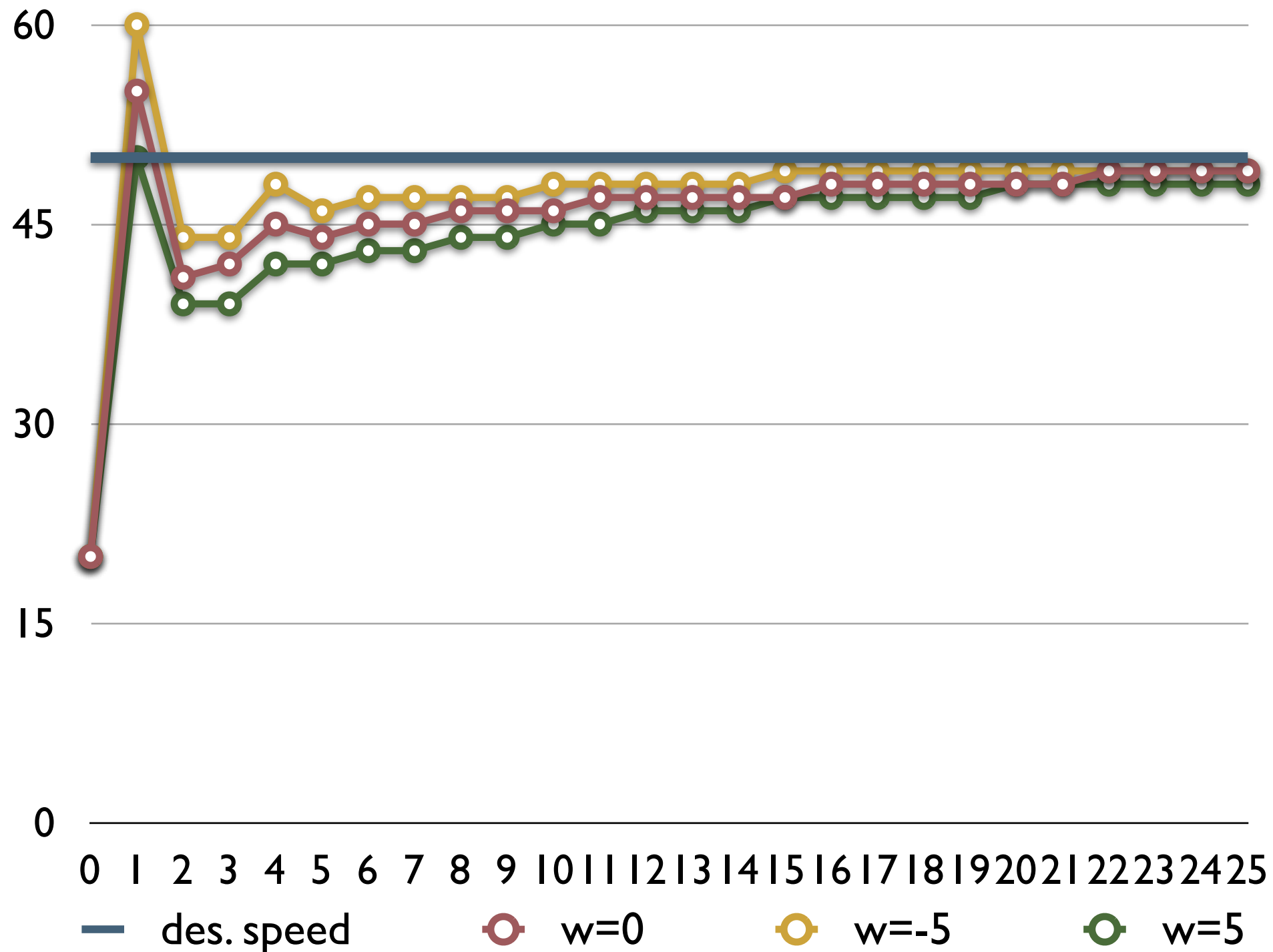
Examples

$w = 0$ $r = 50$ $P = 2,5$ PID

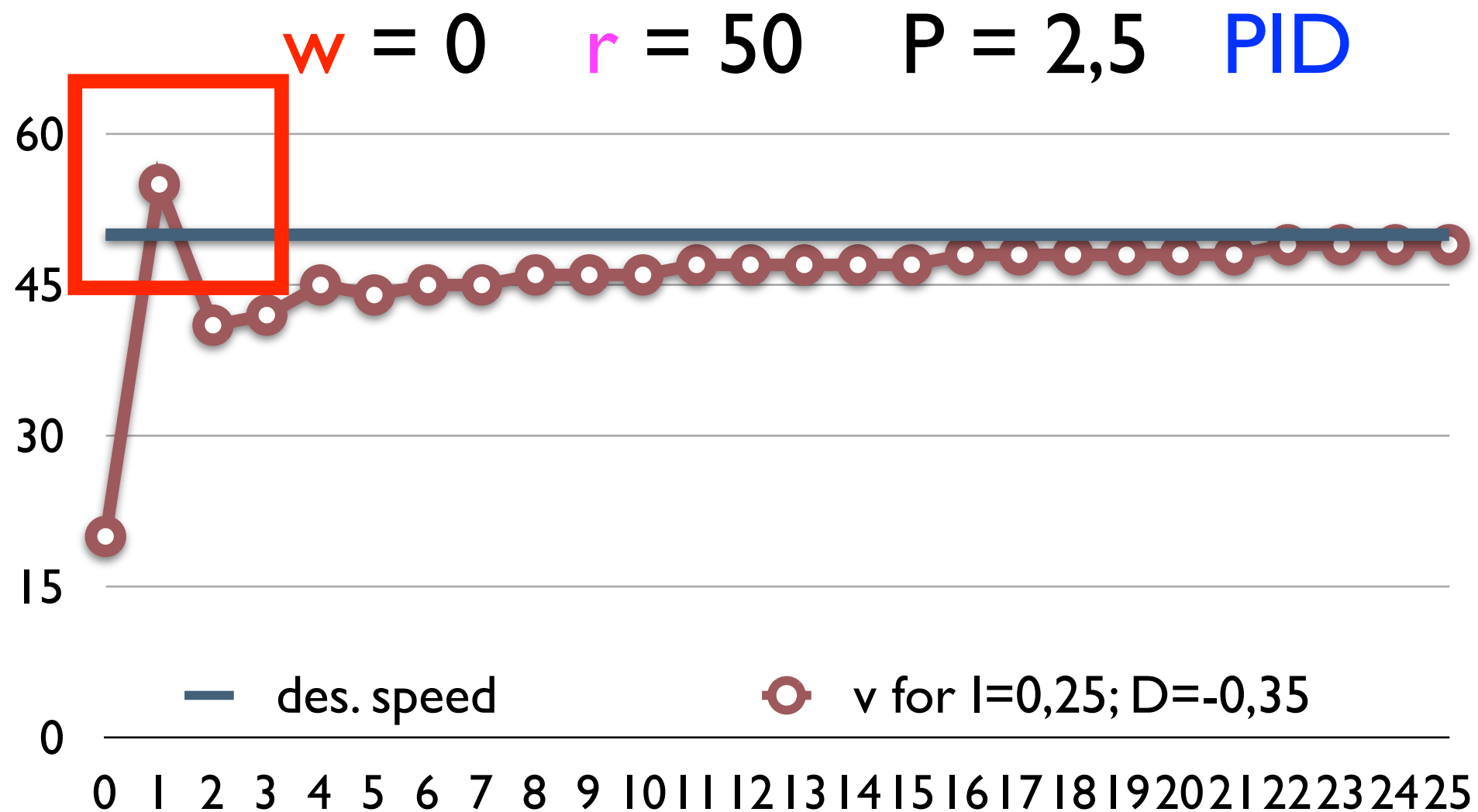


Disturbance rejection

$r = 50$ $P = 2,5$ $I = 0,25$ $D = -0,35$



Remark



The output still has a **high peak value**
This is due in part to the **car model**.
In fact, acceleration will be **smoother** !



Remark - Saturation

- In **practice**, in a **realistic model**, one should take into account the **saturation** of some variables
- For example, u_t (gas pedal) should always be **between 0 and 45** !
- In our **model**, we haven't remarked that the **controller** set u_t to 90 at time 1, hence the peak



Remark - Saturation

- We can thus **modify the controller**:

$$u_t = P \times e_t + I \times \sum_{0 \leq i \leq t} e_i + D \times (e_t - e_{t-1})$$

as:

$$u_t = \min\{P \times e_t + I \times \sum_{0 \leq i \leq t} e_i + D \times (e_t - e_{t-1}), 45\}$$

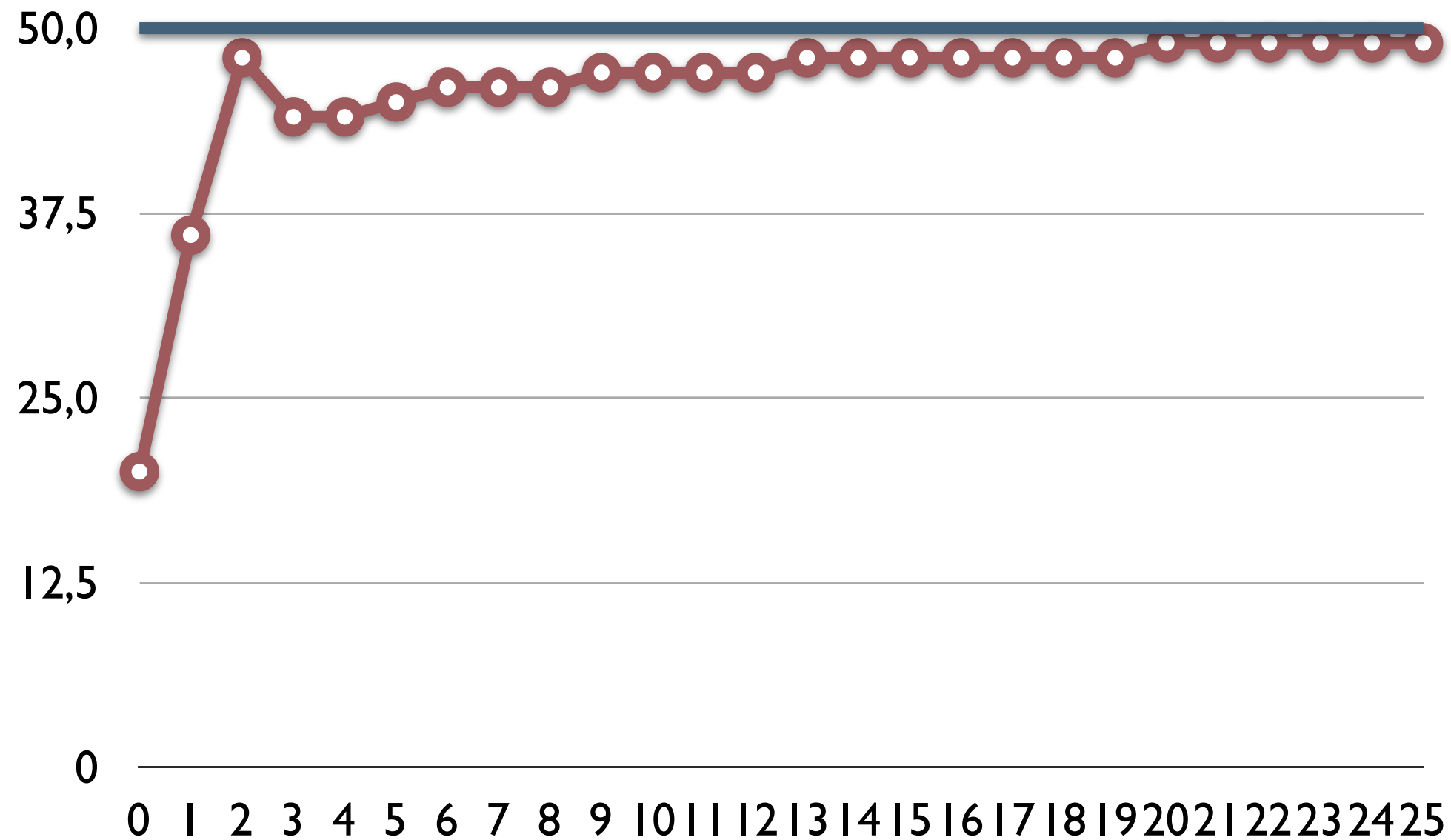
to **avoid overshooting**

- The same can be done **symmetrically**
(with a max) to avoid outputs < 0



Example - saturation

$w = 0$ $r = 50$ $P = 2,5$ PID



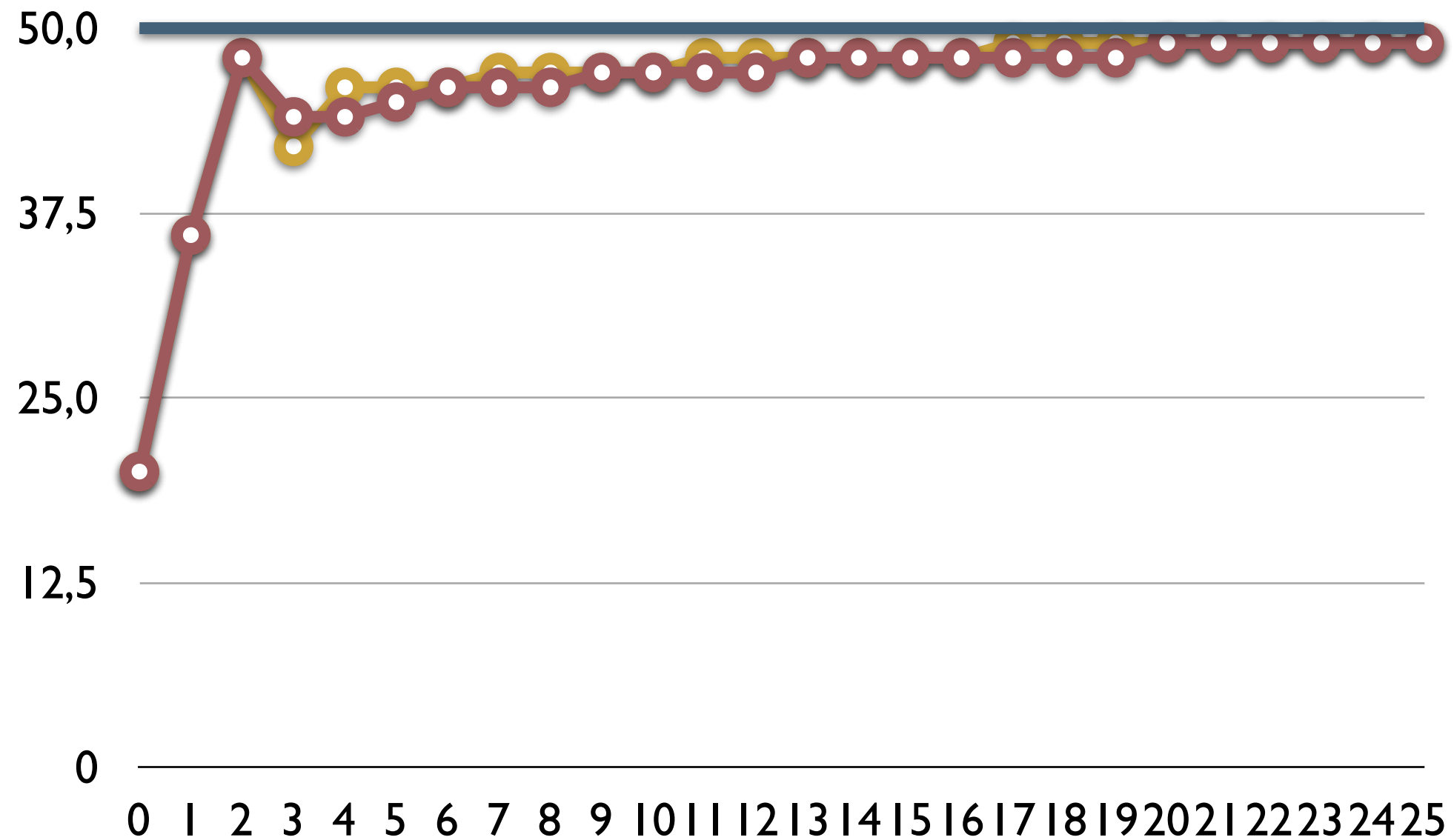
— des. speed.

○ v for $I=0,25$; $D=-0,35$



Saturation - PID vs IP

$w = 0$ $r = 50$ $P = 2,5$ PID

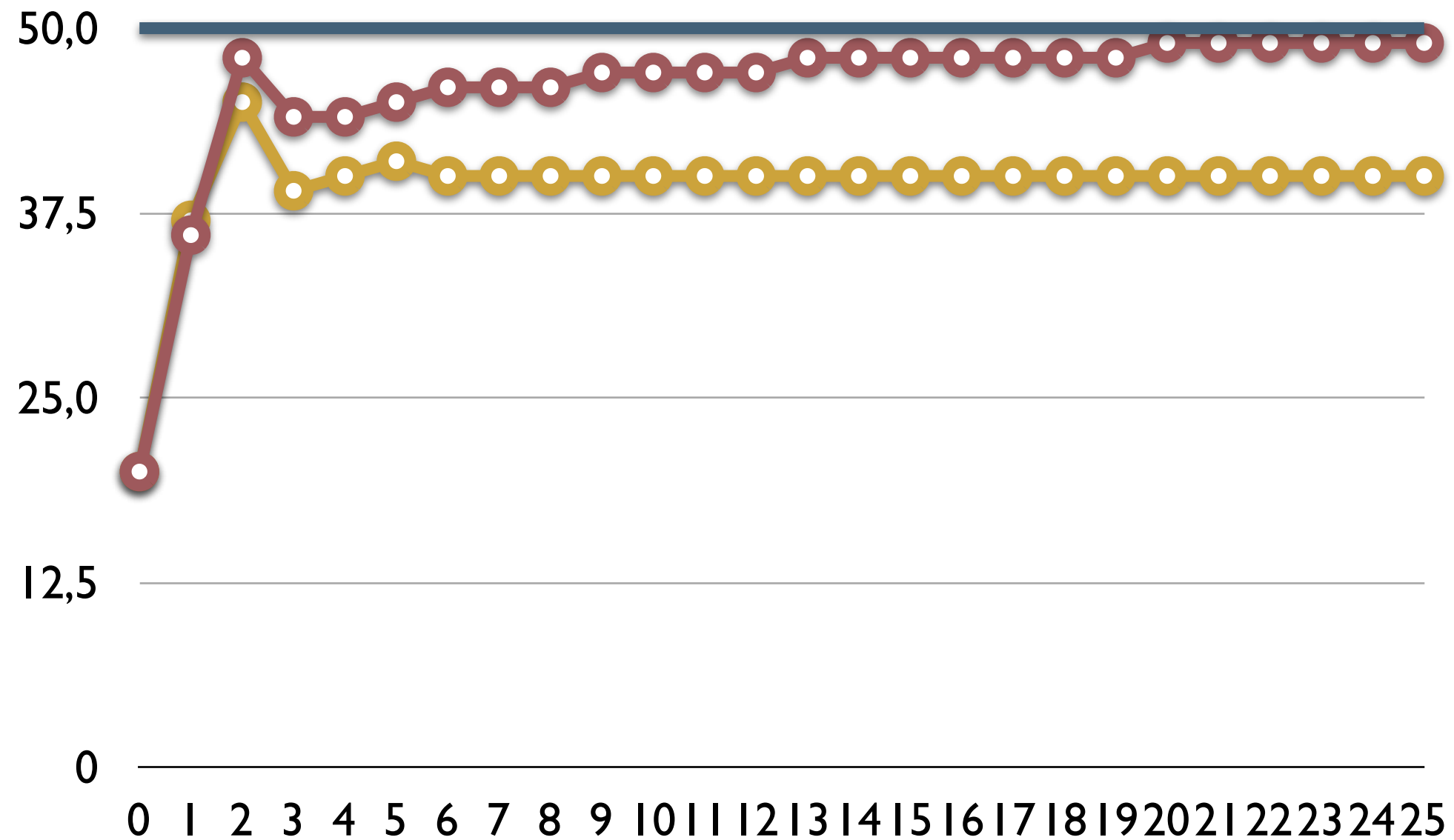


— des. speed. $\circ$ v for $I=0,25; D=-0,35$ $\circ$ v for $I=0,25; D=0$



Saturation - PID vs DP

$w = 0$ $r = 50$ $P = 2,5$ PID

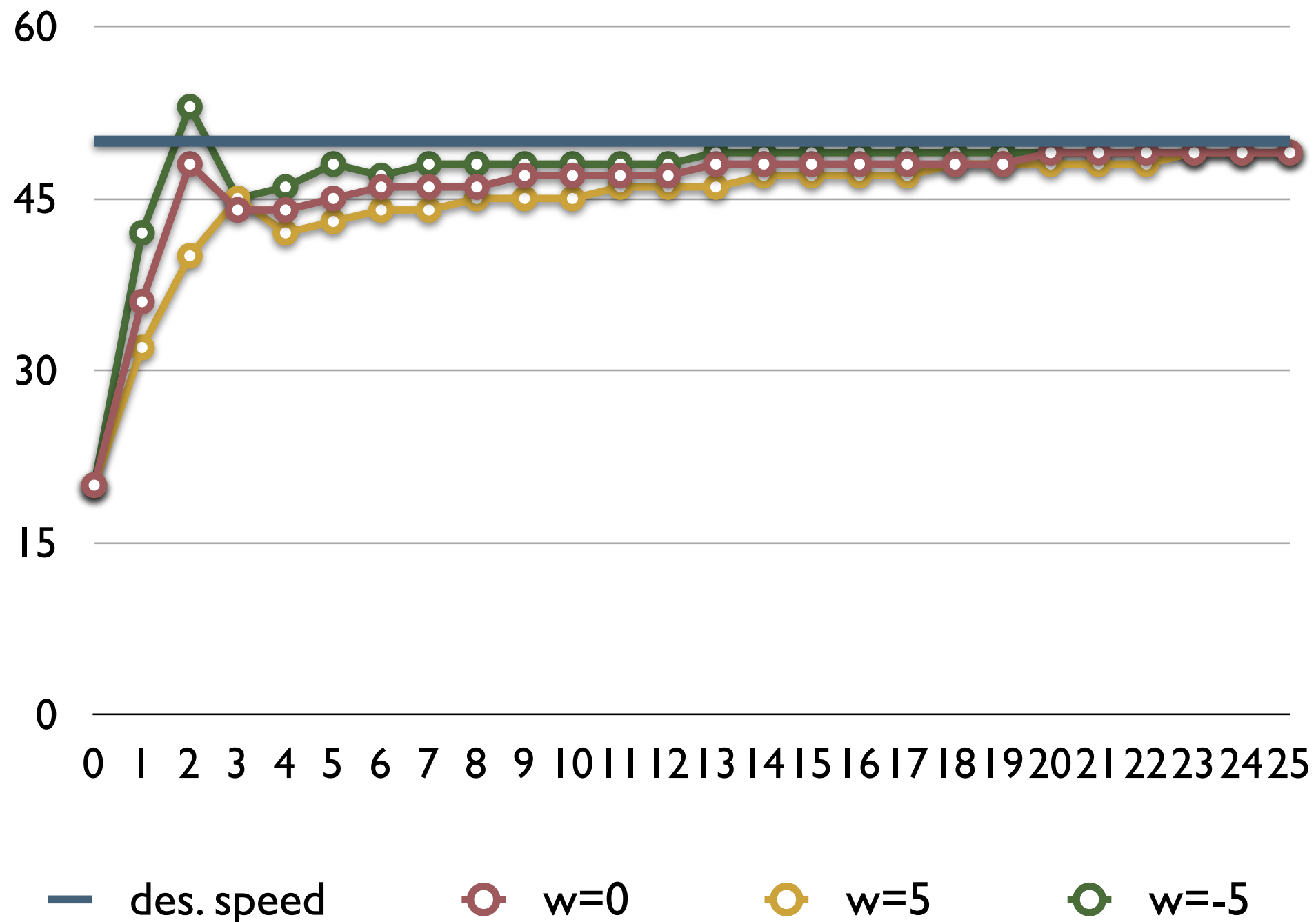


— des. speed. ○ v for $I=0,25$; $D=-0,35$ ○ v for $I=0$; $D=-0,35$



Saturation - disturbances

$r = 50$ $P = 2,5$ $I = 0,25$ $D = -0,35$





Conclusion

Conclusion

- With PID controllers we have **achieved**:
 - **fast convergence** to the desired speed
 - **few and small oscillations**



Conclusion

- However, PID controllers are **extremely simple**:
- **few operations** have to be computed at each step
- trivial to **implement** !
- Exist as ready-made **IC** !



Conclusion

- Those controllers are **sufficient** in many real- case **applications**
- The “**differential**” and “**integral**” terms exist as operators in SCADE library

